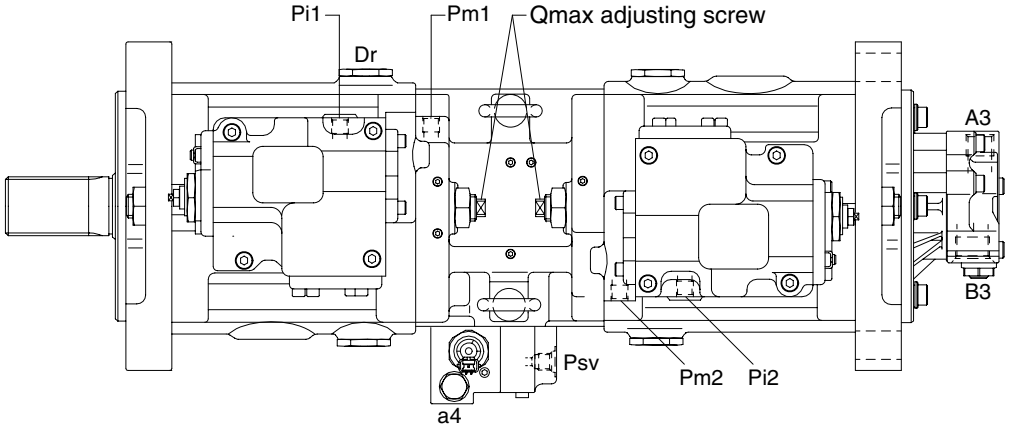
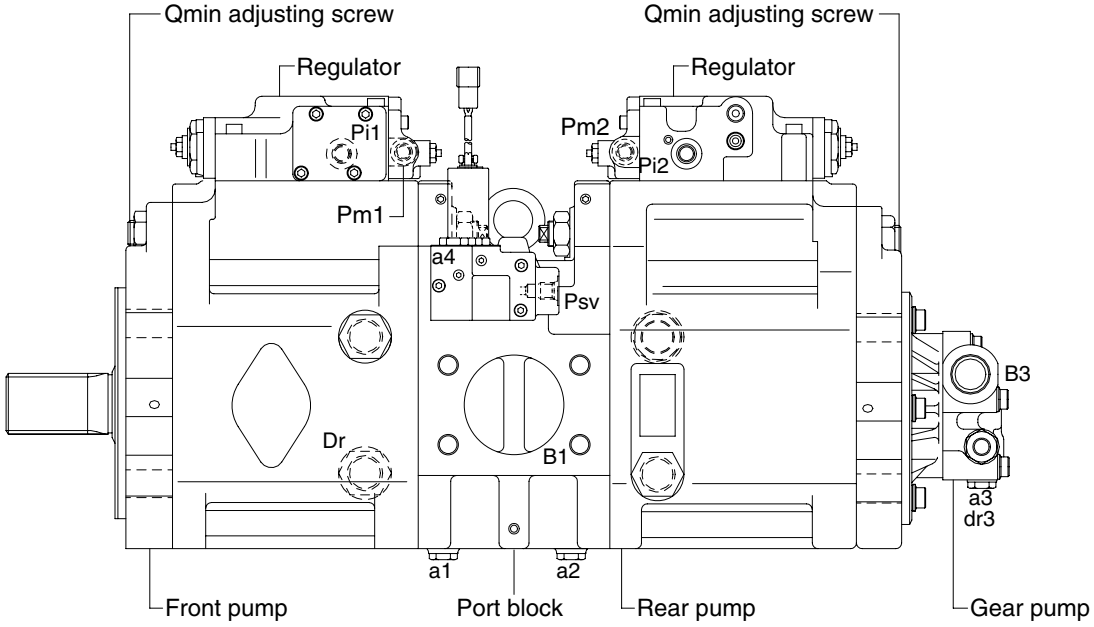
SECTION 2 STRUCTURE AND FUNCTION

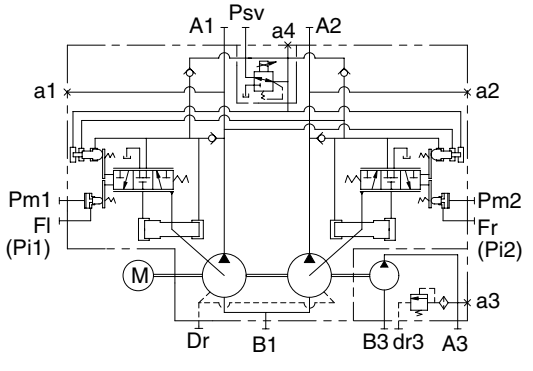
GROUP 1 PUMP DEVICE

1. STRUCTURE

The pump device consists of main pump, regulator and gear pump.





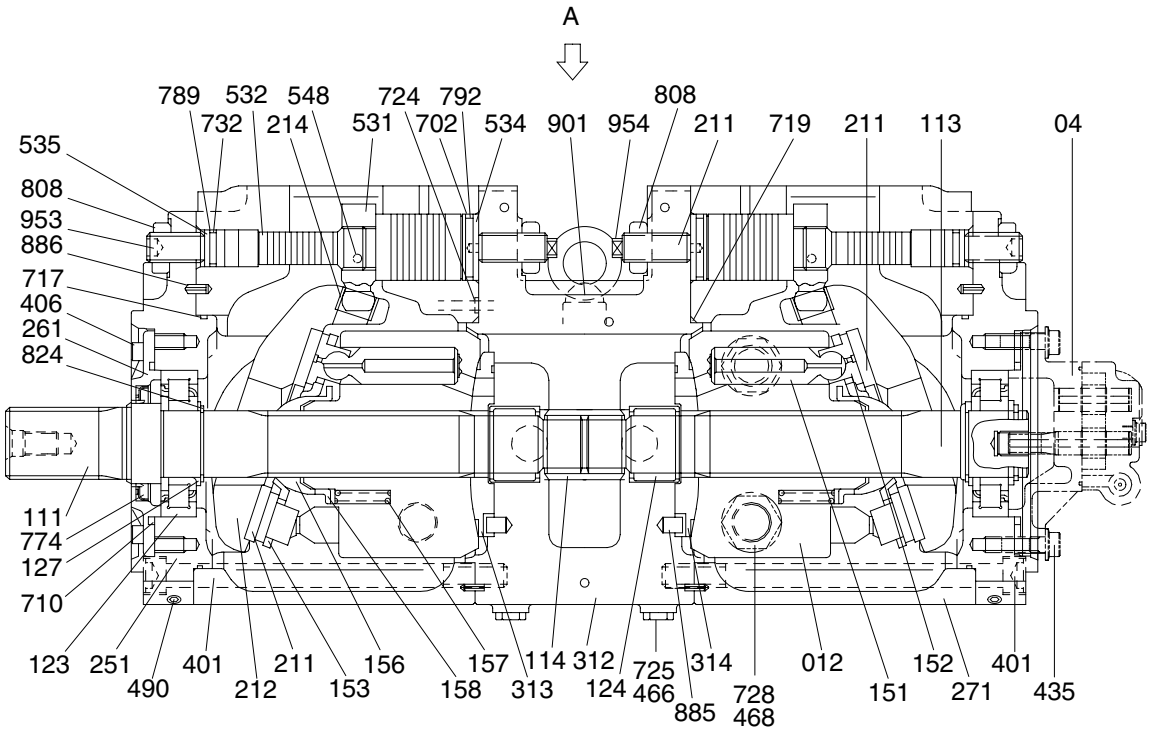


32072ST02

Port	Port name	Port size
A1,2	Delivery port	SAE6000psi 1"
B1	Suction port	SAE2000psi 3"
Dr	Drain port	PF 3/4 - 23
Pi1,i2	Pilot port	PF 1/4 - 15
Pm1,m2	Qmax cut port	PF 1/4 - 15
Psv	Servo assist port	PF 1/4 - 15
a1,2,4	Gauge port	PF 1/4 - 15
a3	Gauge port	PF 1/4 - 14
A3	Gear pump delivery port	PF 1/2 - 19
B3	Gear pump suction port	PF 3/4 - 20.5
dr3	Gear pump drain port	PF 3/8

1) MAIN PUMP(1/2)

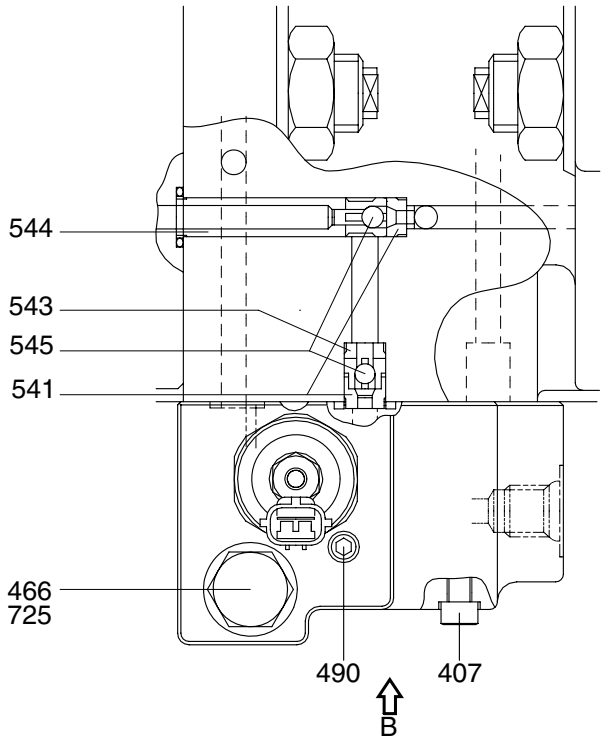
The main pump consists of two piston pumps(front & rear) and valve block.



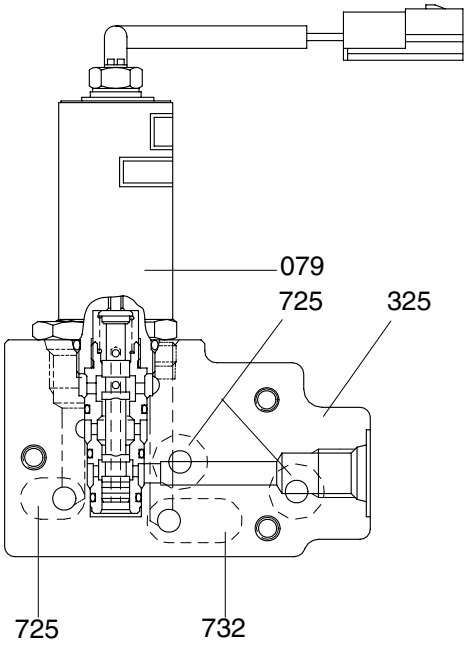
32072ST03

04	Gear pump	261	Seal cover(F)	717	O-ring
111	Drive shaft(F)	271	Pump casing	719	O-ring
113	Drive shaft(R)	312	Valve block	724	O-ring
114	Spline coupling	313	Valve plate(R)	725	O-ring
123	Roller bearing	314	Valve plate(L)	728	O-ring
124	Needle bearing	401	Hexagon socket bolt	732	O-ring
127	Bearing spacer	406	Hexagon socket bolt	774	Oil seal
141	Cylinder block	435	Flange socket bolt	789	Back up ring
151	Piston	466	VP plug	792	Back up ring
152	Shoe	468	VP plug	808	Hexagon head nut
153	Push plate	490	VP plug	824	Snap ring
156	Bushing	531	Tilting pin	885	Pin
157	Cylinder spring	532	Servo piston	886	Spring pin
158	Spacer	534	Stopper(L)	901	Eye bolt
211	Shoe plate	535	Stopper(S)	953	Set screw
212	Swash plate	548	Pin	954	Set screw
214	Bushing	702	O-ring		
251	Swash plate support	710	O-ring		

MAIN PUMP(2/2)



VIEW A

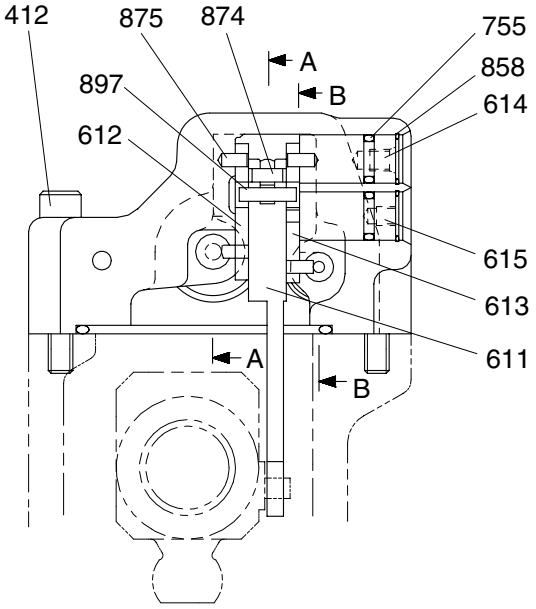


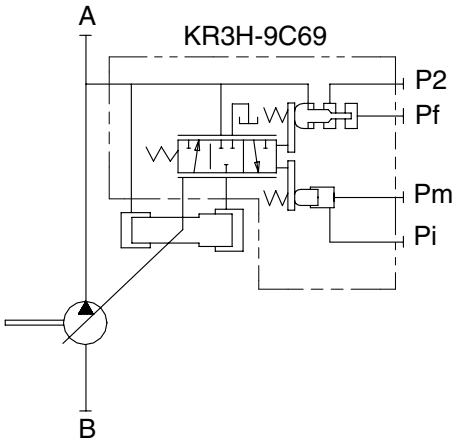
VIEW B

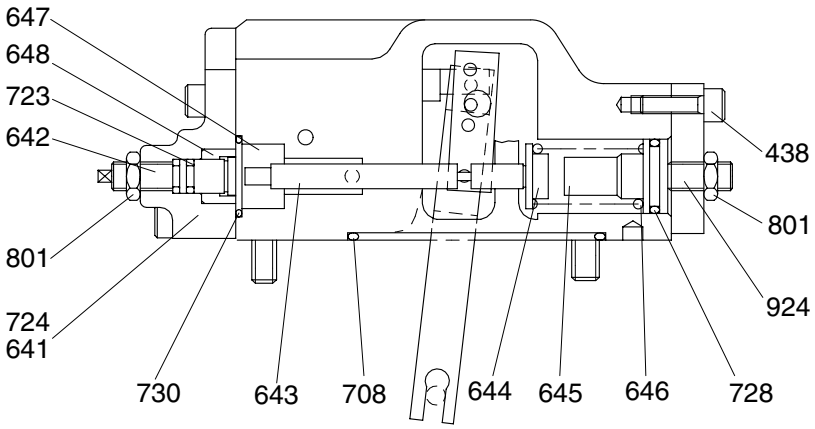
29072MP03

079	Proportional reducing valve	490	Plug	545	Steel ball
325	Casing assy	541	Seat	725	O-ring
407	Hexagon screw	543	Stopper 1	732	O-ring
466	Plug	544	Stopper 2		

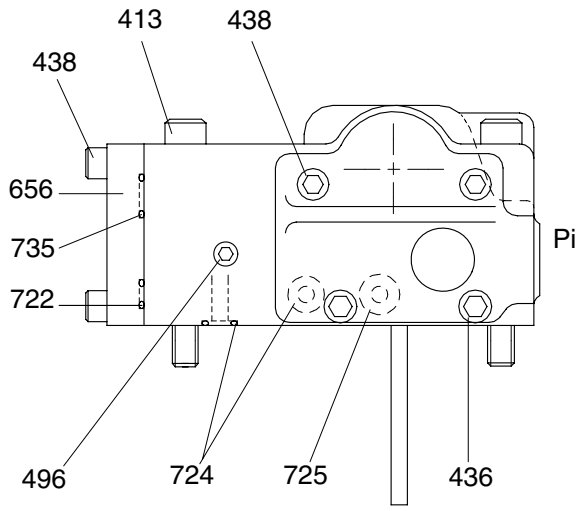
2) REGULATOR(1/2)







SECTION B-B

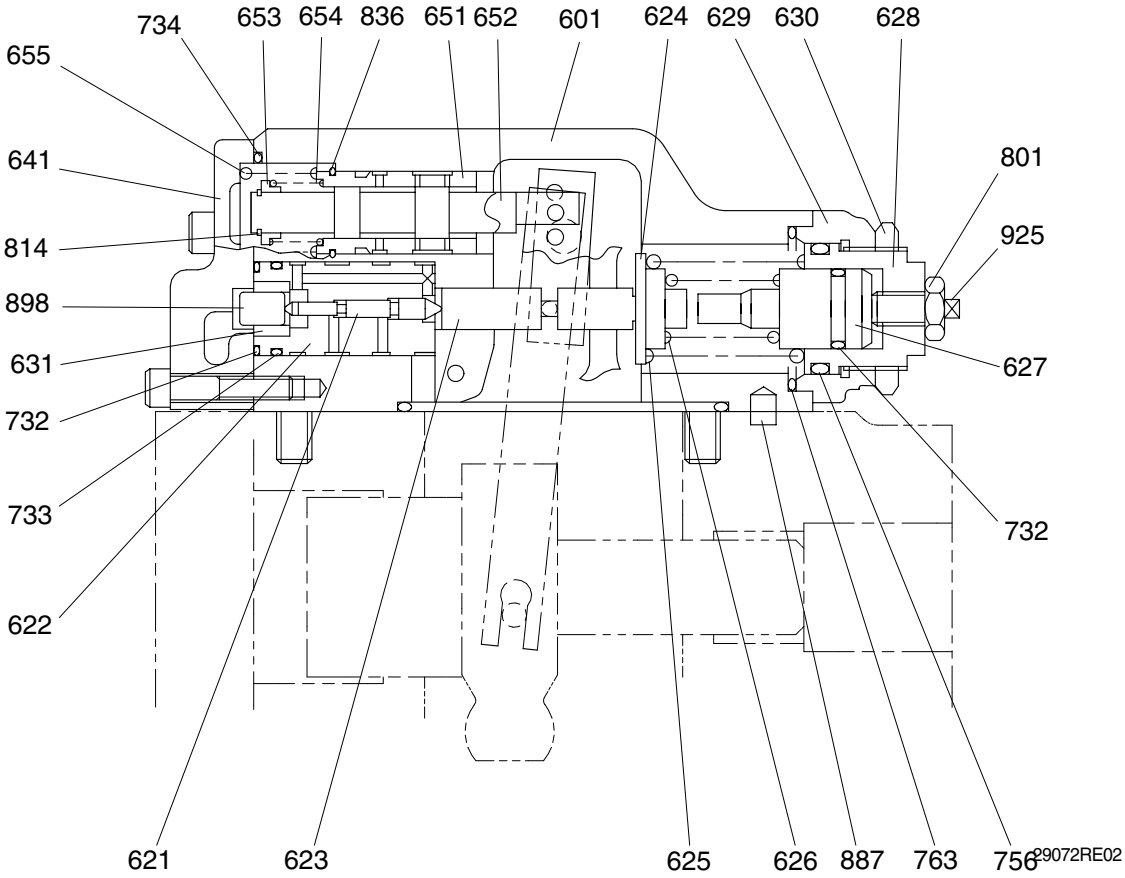


VIEW C

32072ST04

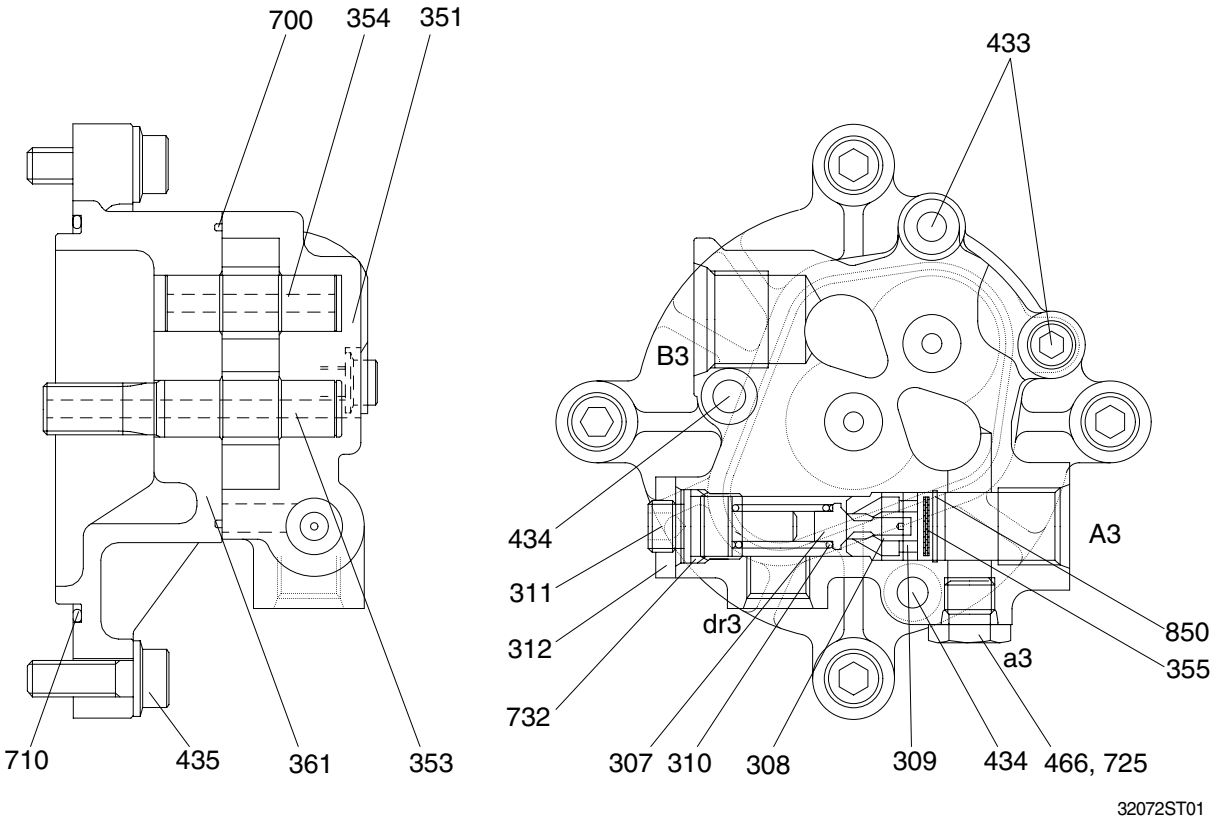
Port	Port name	port size
A	Delivery port	SAE6000psi 1"
B	Suction port	SAE2000psi 3"
Pi	Pilot port	PF 1/4-15
Pm	Qmax cut port	PF 1/4-15

REGULATOR(2/2)



412	Hexagon socket screw	631	Sleeve, pf	730	O-ring
413	Hexagon socket screw	641	Pilot cover	732	O-ring
436	Hexagon socket screw	642	Adjust screw(QMC)	733	O-ring
438	Hexagon socket screw	643	Pilot piston	734	O-ring
496	Plug	644	Spring seat(Q)	735	O-ring
601	Casing	645	Adjust stem(Q)	755	O-ring
611	Feed back lever	646	Pilot spring	756	O-ring
612	Lever(1)	647	Stopper	763	O-ring
613	Lever(2)	648	Piston(QMC)	801	Nut
614	Fulcrum plug	651	Sleeve	814	Snap ring
615	Adjust plug	652	Spool	836	Snap ring
621	Compensator piston	653	Spring seat	858	Snap ring
622	Piston case	654	Return spring	874	Spring pin
623	Compensator rod	655	Set spring	875	Pin
624	Spring seat(C)	656	Block cover	887	Pin
625	Outer spring	708	O-ring	897	Pin
626	Inner spring	722	O-ring	898	Pin
627	Adjust stem(C)	723	O-ring	924	Set screw
628	Adjust screw(C)	724	O-ring	925	Adjust screw(QI)
629	Cover(C)	725	O-ring		
630	Lock nut	728	O-ring		

3) GEAR PUMP



307	Poppet	353	Drive gear	466	Plug
308	Seat	354	Driven gear	700	Ring
309	Spring seat	355	Filter	710	O-ring
310	Spring	361	Front case	725	O-ring
311	Screw	433	Flange socket	732	O-ring
312	Nut	434	Flange socket	850	Snap ring
351	Gear case	435	Flange socket		

2. FUNCTION

1) MAIN PUMP

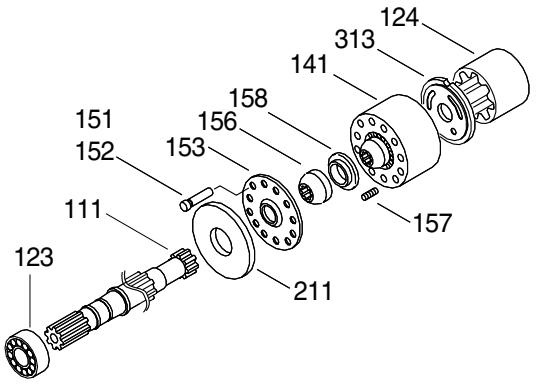
The pumps may classified roughly into the rotary group performing a rotary motion and working as the major part of the whole pump function: the swash plate group that varies the delivery rates: and the valve cover group that changes over oil suction and discharge.

(1) Rotary group

The rotary group consists of drive shaft (F)(111), cylinder block(141), piston shoes(151,152), set plate(153), spherical bush(156), spacer(158) and cylinder spring(157). The drive shaft is supported by bearing(123,124) at its both ends.

The shoe is caulked to the piston to from a spherical coupling. It has a pocket to relieve thrust force generated by loading pressure and the take hydraulic balance so that it slides lightly over the shoe plate(211). The sub group composed by a piston and a shoe is pressed against the shoe plate by the action of the cylinder spring via a retainer and a spherical bush.

Similarly, the cylinder block is pressed against valve plate(313) by the action of the cylinder spring.



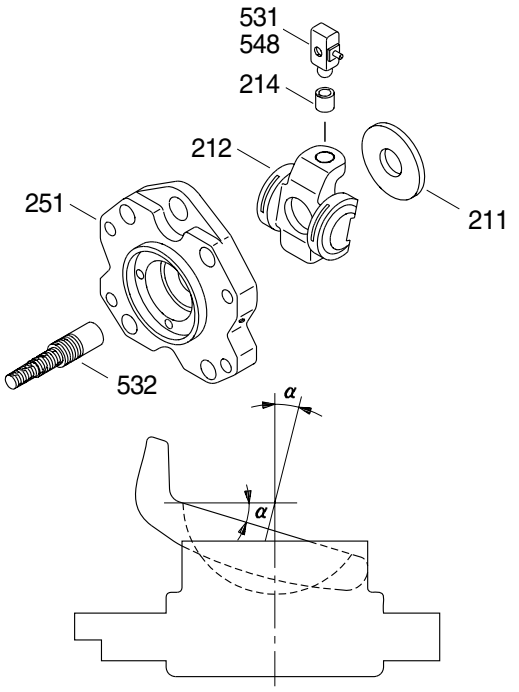
2-7(1) 210-7

(2) Swash plate group

The swash plate group consists of swash plate(212), shoe plate(211), swash plate support(251), tilting bush(214), tilting pin(531) and servo piston(532).

The swash plate is a cylindrical part formed on the opposite side of the sliding surface of the shoe and is supported by the swash support.

If the servo piston moves to the right and left as hydraulic force controlled by the regulator is admitted to hydraulic chamber located on both sides of the servo piston, the swash plate slides over the swash plate support via the spherical part of the tilting pin to change the tilting angle(α)



2-7 (210-7)

(3) Valve block group

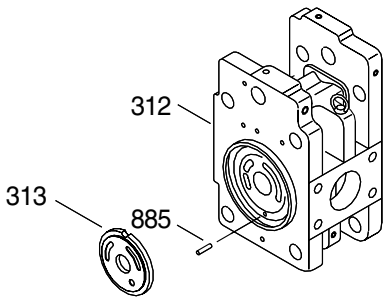
The valve block group consists of valve block(312), valve plate(313) and valve plate pin(885).

The valve plate having two melon-shaped ports is fixed to the valve block and feeds and collects oil to and from the cylinder block.

The oil changed over by the valve plate is connected to an external pipeline by way of the valve block.

Now, if the drive shaft is driven by a prime mover(electric motor, engine, etc), it rotates the cylinder block via a spline linkage at the same time. If the swash plate is tilted as in Fig(previous page) the pistons arranged in the cylinder block make a reciprocating motion with respect to the cylinder block, while they revolve with the cylinder block.

If you pay attention to a single piston, it performs a motion away from the valve plate(oil sucking process) within 180 degrees, and makes a motion towards the valve plate(or oil discharging process) in the rest of 180 degrees. When the swash plate has a tilting angle of zero, the piston makes no stroke and discharges no oil.



2-8 (210-7)

2) REGULATOR

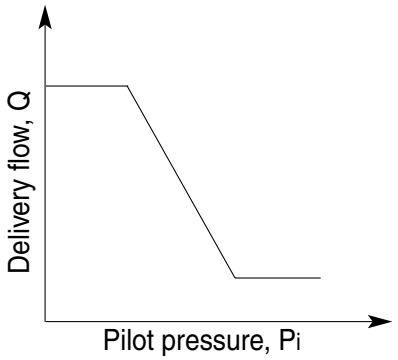
Regulator consists of the negative flow control, total horse power control and power shift control function.

(1) Negative flow control

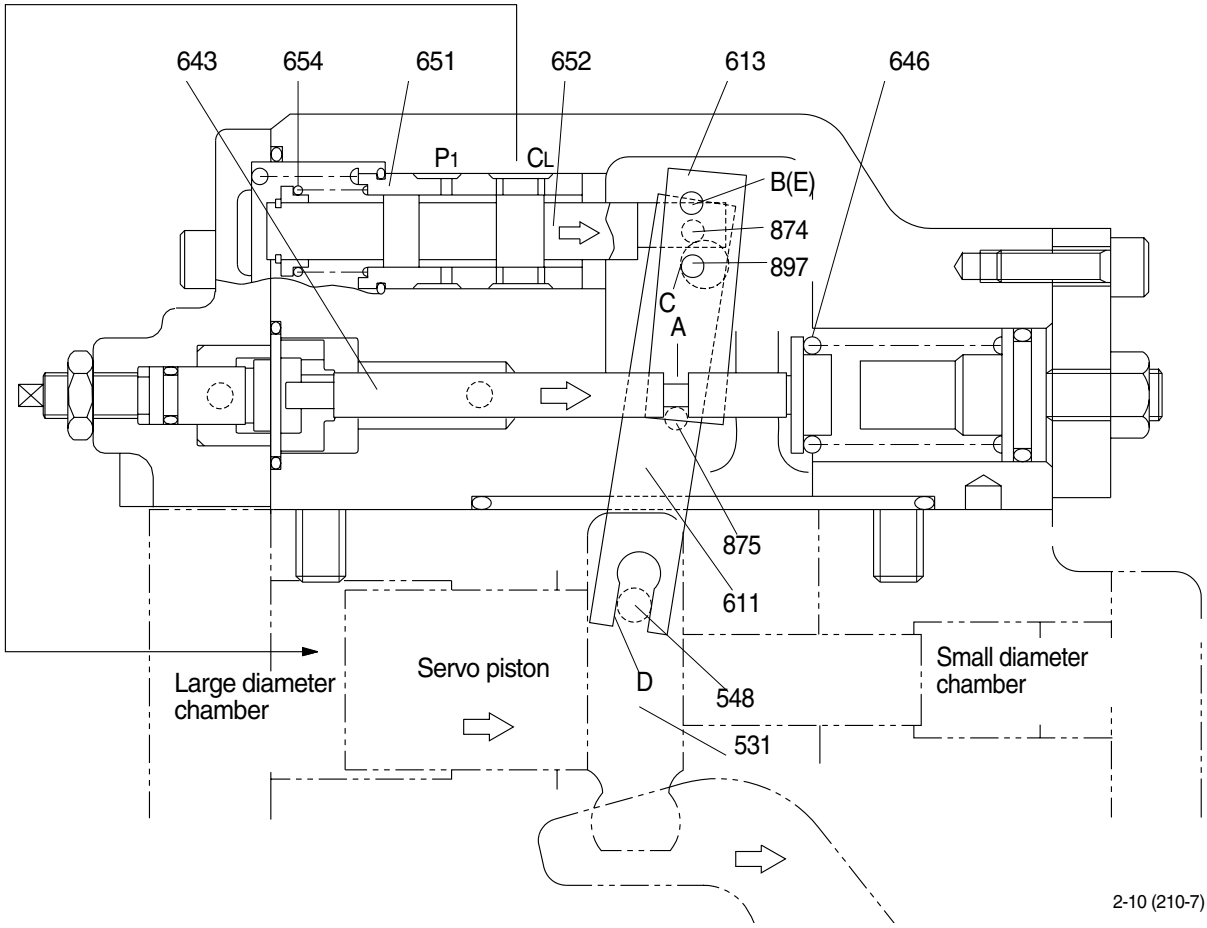
By changing the pilot pressure Pi, the pump tilting angle(delivery flow) is regulated arbitrarily, as shown in the figure.

This regulator is of the negative flow control in which the delivery flow Q decreases as the pilot pressure Pi rises.

With this mechanism, when the pilot pressure corresponding to the flow required for the work is commanded, the pump discharges the required flow only, and so it does not consume the power uselessly.



① Flow reducing function



As the pilot pressure Pi rises, the pilot piston(643) moves to the right to a position where the force of the pilot spring(646) balances with the hydraulic force.

The groove(A) in the pilot piston is fitted with the pin(875) that is fixed to lever 2(613). Therefore, when the pilot piston moves, lever 2 rotates around the fulcrum of point B [Fixed by the fulcrum plug(614) and pin(875)]. Since the large hole section(C) of lever 2 contains a protruding pin(897) fixed to the feedback lever(611), the pin(897) moves to the right as lever 2 rotates. Since the opposing-flat section(D) of the feedback lever is fitted with the pin(548) fixed by the tilting pin(531) that swings the swash plate, the feedback lever rotates around the fulcrum of point D, as the pin(897) moves.

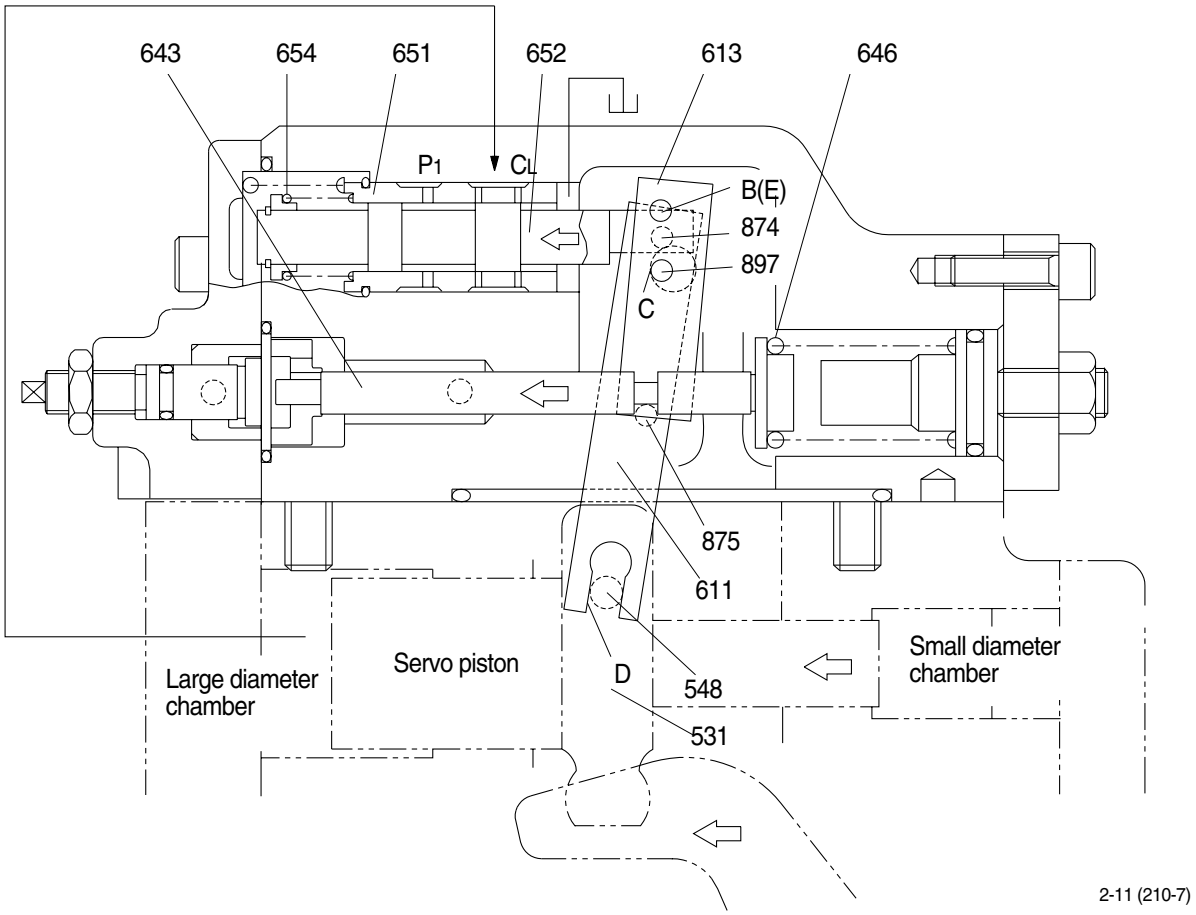
Since the feedback lever is connected with the spool(652) via the pin(874), the spool moves to the right.

The movement of the spool causes the delivery pressure P1 to connect to port CL through the spool and to be admitted to the large diameter section of the servo piston. The delivery pressure P1 that is constantly admitted to the small diameter section of the servo piston moves the servo piston to the right due to the area difference, resulting in decrease of the tilting angle.

When the servo piston moves to the right, point D also moves to the right. The spool is fitted with the return spring(654) and is tensioned to the left at all times, and so the pin(897) is pressed against the large hole section(C) of lever 2.

Therefore, as point D moves, the feedback lever rotates around the fulcrum of point C, and the spool is shifted to the left. This causes the opening between the sleeve(651) and spool(652) to close slowly, and the servo piston comes to a complete stop when it closes completely.

② **Flow increasing function**



As the pilot pressure Pi decreases, the pilot piston(643) moves to the left by the action of the pilot spring(646) and causes lever 2(613) to rotate around the fulcrum of point B. Since the pin(897) is pressed against the large hole section(C) of lever 2 by the action of the return spring(654) via the spool(652), pin(874), and feedback lever(611), the feedback lever rotates around the fulcrum of point D as lever 2 rotates, and shifts the spool to the left. Port CL opens a way to the tank port as the spool moves. This deprives the large diameter section of the servo piston of pressure, and shifts the servo piston to the left by the discharge pressure P1 in the small diameter section, resulting in an increase in the flow rate.

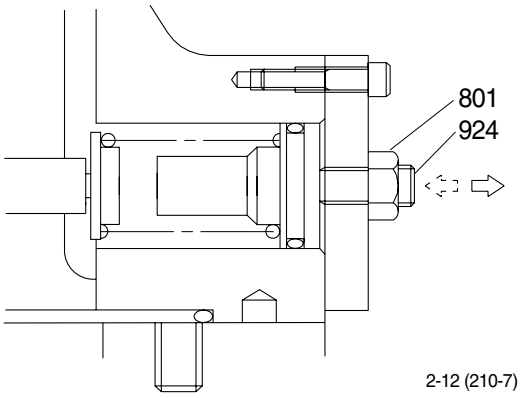
As the servo piston moves, point D also moves to the left, the feedback lever rotates around the fulcrum of point C, and the spool moves to the right till the opening between the spool and sleeve is closed.

③ **Adjustment of flow control characteristic**

The flow control characteristic can be adjusted with the adjusting screw.

Adjust it by loosening the hexagon nut (801) and by tightening(or loosening) the hexagonal socket head screw(924).

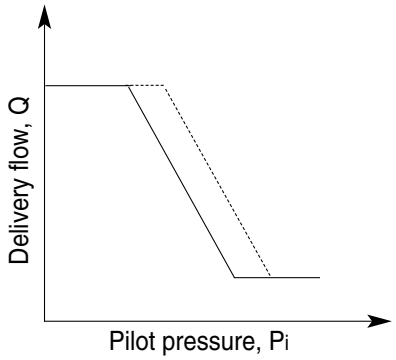
Tightening the screw shifts the control chart to the right as shown in the figure.



2-12 (210-7)

※ **Adjusting values are shown in table.**

Speed	Adjustment of flow control characteristic		
	Tightening amount of adjusting screw(924)	Flow control starting pressure change amount	Flow change amount
(min $^{-1}$)	(Turn)	(kgf/cm^2)	(ℓ /min)
1750	+1/4	+1.6	+18.4



(2) Total horsepower control

The regulator decreases the pump tilting angle(delivery flow) automatically to limit the input torque within a certain value with a rise in the delivery pressure P1 of the self pump and the delivery pressure P2 of the companion pump.

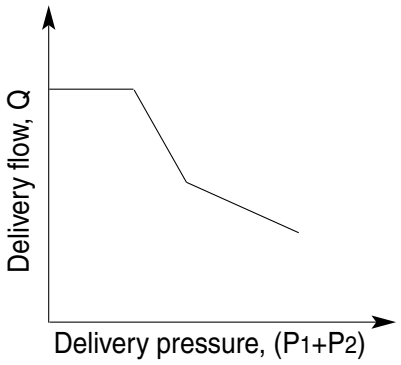
(The input horsepower is constant when the speed is constant.)

Since the regulator is of the simultaneous total horsepower type that operates by the sum of load pressures of the two pumps in the tandem double-pump system, the prime mover is automatically prevented from being overloaded, irrespective of the load condition of the two pumps, when horsepower control is under way.

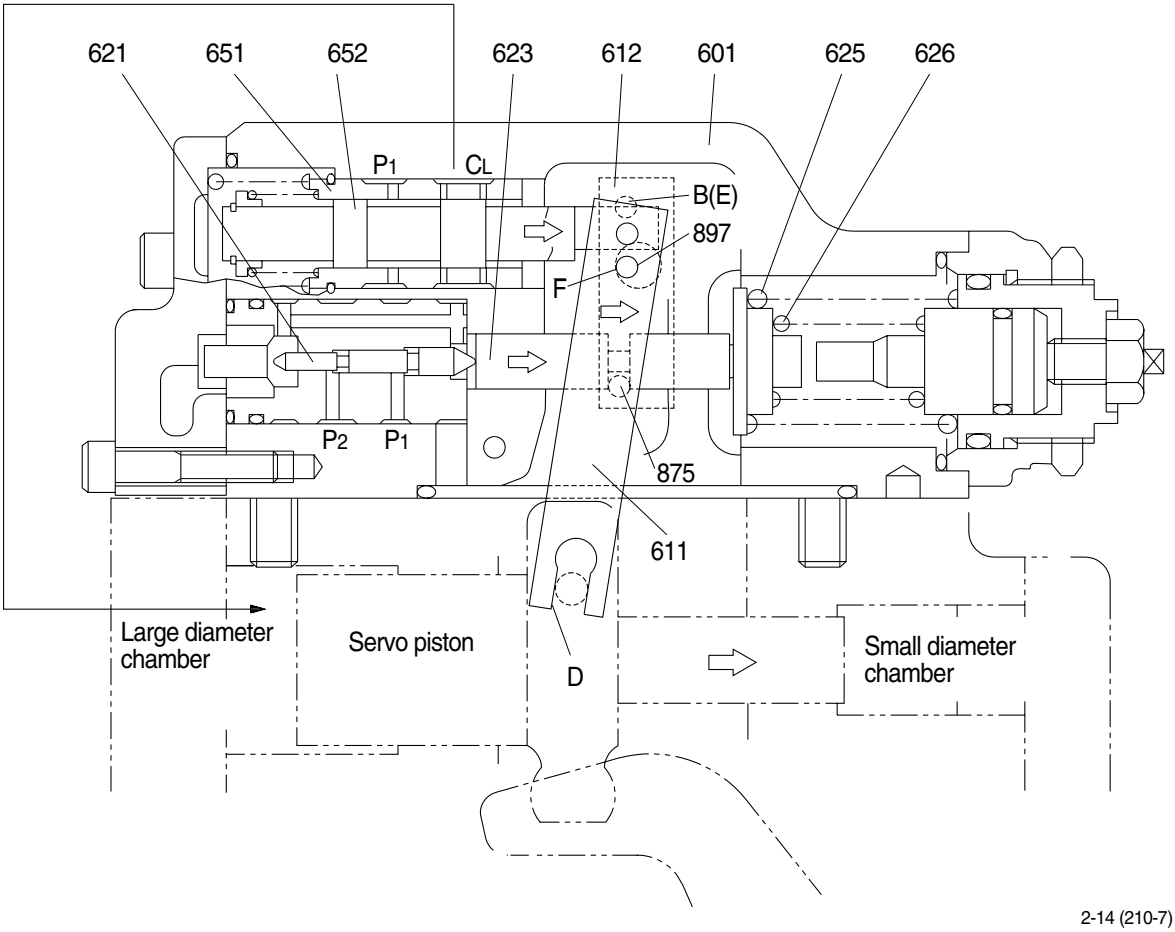
Since this regulator is of the simultaneous total horsepower type, it controls the tilting angles(displacement volumes) of the two pumps to the same value as represented by the following equation :

$$Tin = P1 \times q/2\pi + P2 \times q/2\pi$$
$$= (P1+P2) \times q/2\pi$$

The horsepower control function is the same as the flow control function and is summarized in the following.(For detailed behaviors of respective parts, refer to the section of flow control).



① Overload preventive function



2-14 (210-7)

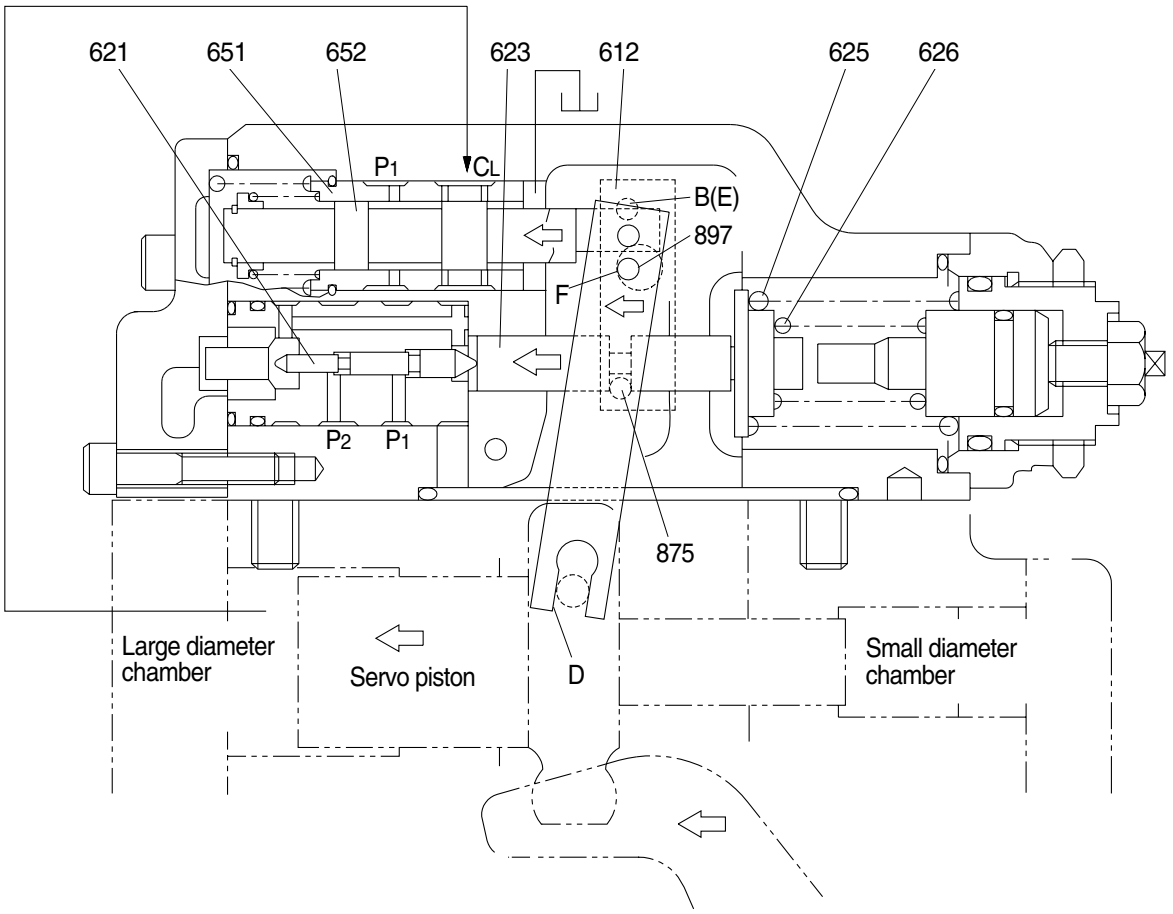
When the self pump delivery pressure P1 or the companion pump delivery pressure P2 rises, it acts on the stepped part of the compensating piston(621). It presses the compensating rod(623) to the right till the force of the outer spring(625) and inner spring(626) balances with the hydraulic force. The movement of the compensating rod is transmitted to lever 1(612) via pin(875).

Lever 1 rotates around the pin(875) (E) fixed to the casing(601).

Since the large hole section(F) of lever 1 contains a protruding pin(897) fixed to the feedback lever(611), the feedback lever rotates around the fulcrum of point D as lever 1 rotates, and then the spool(652) is shifted to the right. As the spool moves, the delivery pressure P1 is admitted to the large diameter section of the servo piston via port CL, causes the servo piston move to the right, reduces the pump delivery, flow rate, and prevents the prime mover from being overloaded.

The movement of the servo piston is transmitted to the feedback lever via point D. Then the feedback lever rotates around the fulcrum of point F and the spool is shifted to the left. The spool moves till the opening between the spool(652) and sleeve(651) is closed.

② **Flow reset function**



2-15 (210-7)

As the self pump delivery pressure P1 or the companion pump delivery pressure P2 decreases, the compensating rod(623) is pushed back by the action of the springs(625 & 626) to rotate lever 1(612) around point E. Rotating of lever 1 causes the feedback lever(611) to rotate around the fulcrum of point D and then the spool(652) to move to the left. As a result, port CL opens a way to the tank port.

This causes the servo piston to move to the left and the pump's delivery rate to increase.

The movement of the servo piston is transmitted to the spool by the action of the feedback mechanism to move it till the opening between the spool and sleeve is closed.

③ **Low tilting angle(low flow) command preferential function**

As mentioned above, flow control and horsepower control tilting angle commands are transmitted to the feedback lever and spool via the large-hole sections(C & F) of levers 1 and 2. However, since sections C and F have the pins(ø 4) protruding from the large hole(ø 8), only the lever lessening the tilting angle contacts the pin(897) ; the hole(ø 8) in the lever of a larger tilting angle command is freed without contacting the pin(897). Such a mechanical selection method permits preference of the lower tilting angle command of the flow control and horsepower control.

④ **Adjustment of input horsepower**

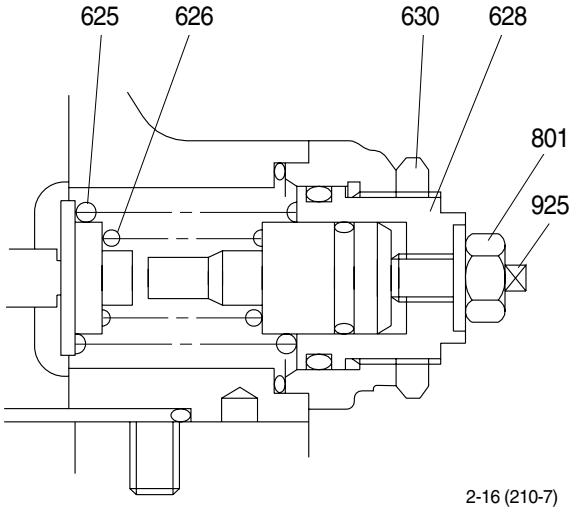
Since the regulator is of total cumulative horsepower type, adjust the adjusting screws of both the front and rear pumps, when changing the horsepower set values. The pressure change values by adjustment are based on two pumps pressurized at the same time, and the values will be doubled when only one pump is loaded.

a. Adjustment of outer spring

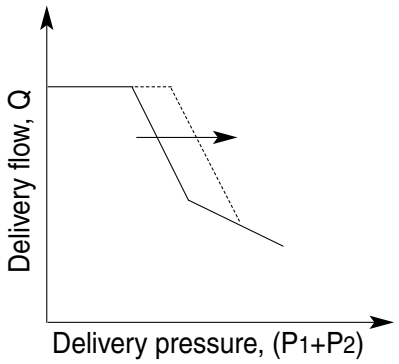
Adjust it by loosening the hexagon nut(630) and by tightening(or loosening) the adjusting screw C(628). Tightening the screw shifts the control chart to the right and increases the input horsepower as shown in the figure. Since turning the adjusting screw C by N turns changes the setting of the inner spring(626), return the adjusting screw QI(925) by N × A turns at first.(A=1.59)

※ **Adjusting values are shown in table.**

Speed	Adjustment of outer spring		
	Tightening amount of adjusting screw(628)	Compensating control starting pressure change amount	Input torque change amount
(min^{-1})	(Turn)	(kgf/cm^2)	(kgf · m)
1750	+1/4	+19.2	+8.3



2-16 (210-7)

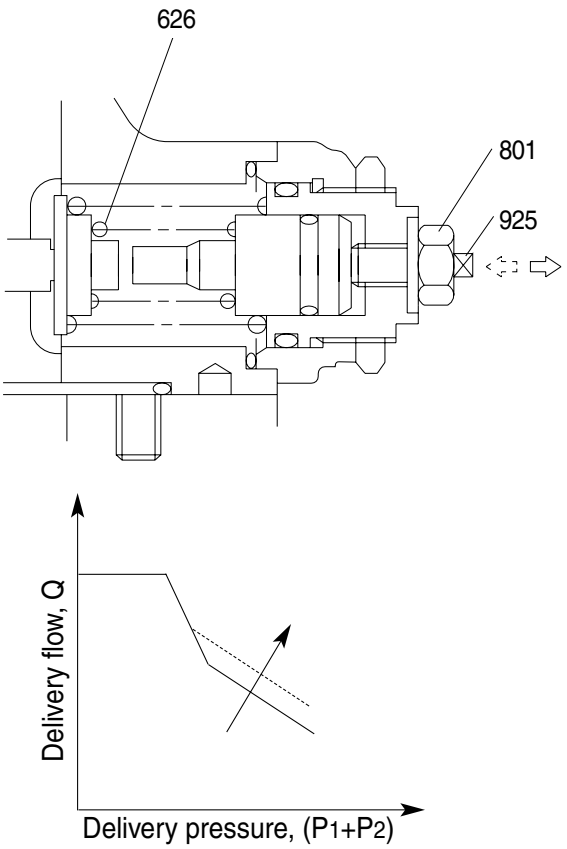


b. Adjustment of inner spring

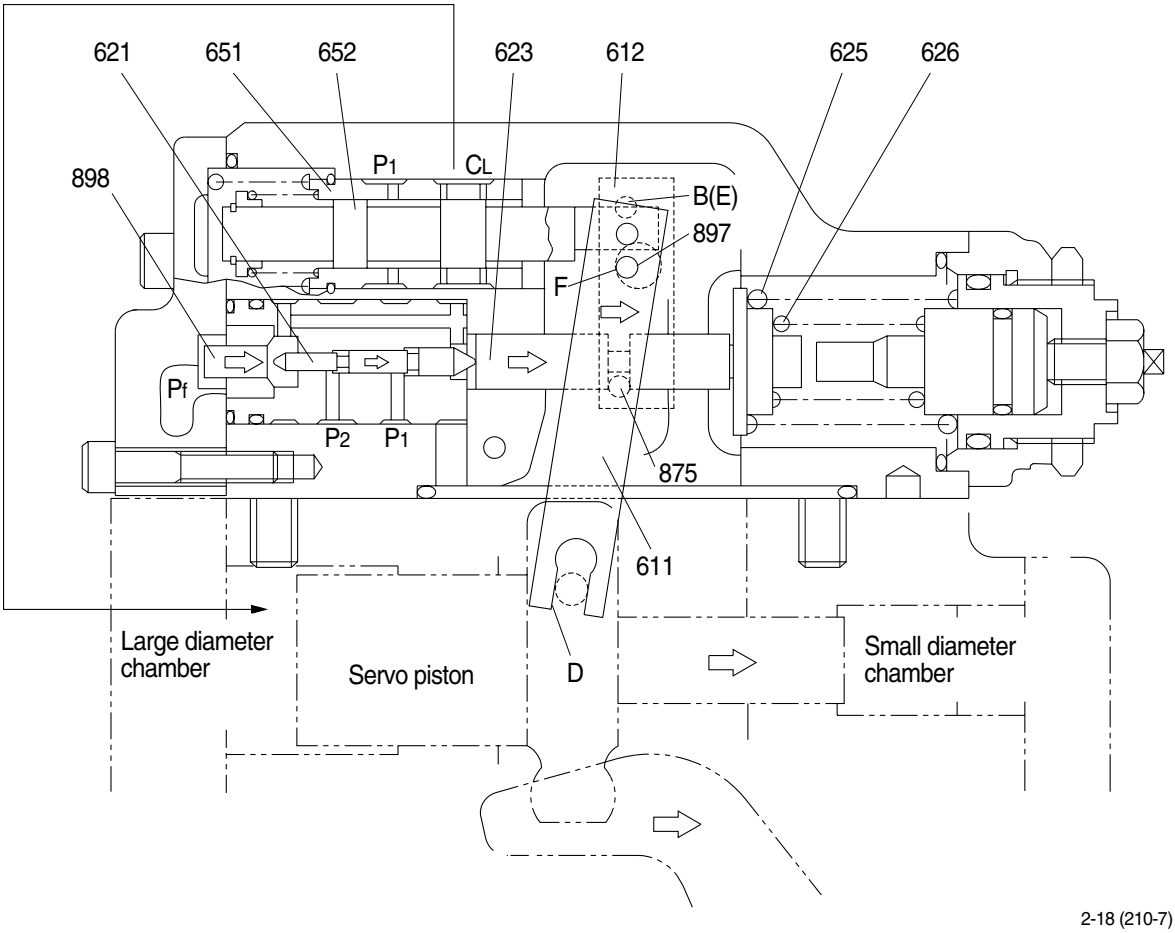
Adjust it by loosening the hexagon nut (801) and by tightening(or loosening) the adjusting screw QI(925). Tightening the screw increases the flow and then the input horsepower as shown in the figure.

※ Adjusting valves are shown in table.

Speed	Adjustment of inner spring		
	Tightening amount of adjusting screw(QI) (925)	Flow change amount	Input torque change amount
(min $^{-1}$)	(Turn)	(lpm)	(kgf · m)
1750	+1/4	+15.9	+8.5



(3) Power shift control

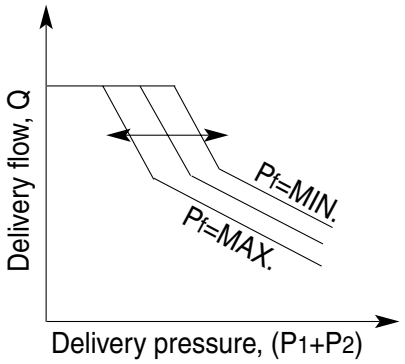


2-18 (210-7)

The set horsepower valve is shifted by varying the command current level of the proportional pressure reducing valve attached to the pump.

Only one proportional pressure reducing valve is provided.

However, the secondary pressure Pf (power shift pressure) is admitted to the horsepower control section of each pump regulator through the pump's internal path to shift it to the same set horsepower level.



This function permits arbitrary setting of the pump output power, thereby providing the optimum power level according to the operating condition.

The power shift pressure Pf controls the set horsepower of the pump to a desired level, as shown in the figure.

As the power shift pressure Pf rises, the compensating rod(623) moves to the right via the pin(898) and compensating piston(621).

This decreases the pump tilting angle and then the set horsepower in the same way as explained in the overload preventive function of the horsepower control. On the contrary, the set horsepower rises as the power shift pressure Pf falls.

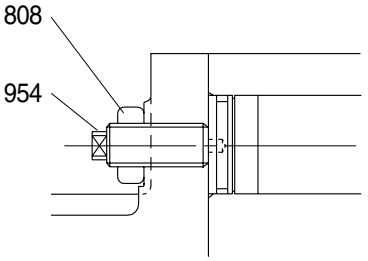
(4) Adjustment of maximum and minimum flows

① Adjustment of maximum flow

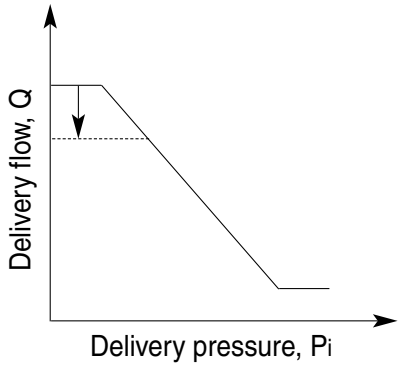
Adjust it by loosening the hexagon nut(808) and by tightening(or loosening) the set screw(954).

The maximum flow only is adjusted without changing other control characteristics.

Speed	Adjustment of max flow	
	Tightening amount of adjusting screw (954)	Flow change amount
(min^{-1})	(Turn)	(ℓ /min)
1750	+1/4	-6.7



2-19(1) 210-7

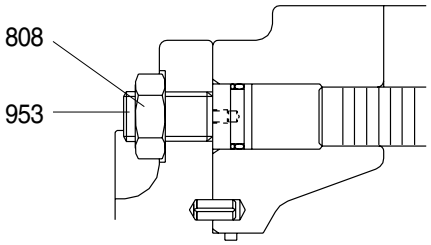


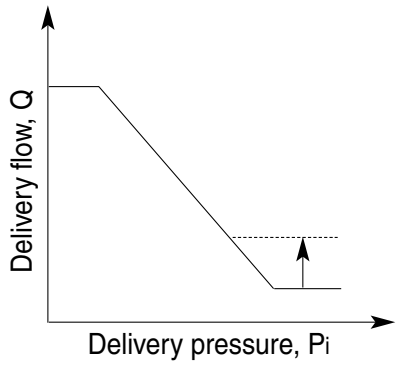
② Adjustment of minimum flow

Adjust it by loosening the hexagon nut(808) and by tightening(or loosening) the hexagonal socket head set screw (953). Similarly to the adjustment of the maximum flow, other characteristics are not changed.

However, remember that, if tightened too much, the required horsepower during the maximum delivery pressure(or during relieving) may increase.

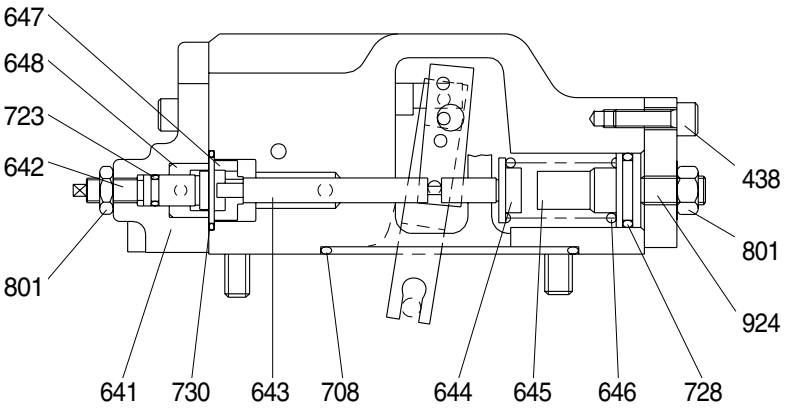
Speed	Adjustment of min flow	
	Tightening amount of adjusting screw (953)	Flow change amount
(min^{-1})	(Turn)	(ℓ /min)
1750	+1/4	+6.7





(5) Qmax cut control

The regulator regulates the maximum delivery flow by inputting the pilot pressure Pm. Since this is a 2-position control method, the maximum delivery flow may be switched in two steps by turning on/off the pilot pressure Pm.(The maximum control flow cannot be controlled in intermediate level.)

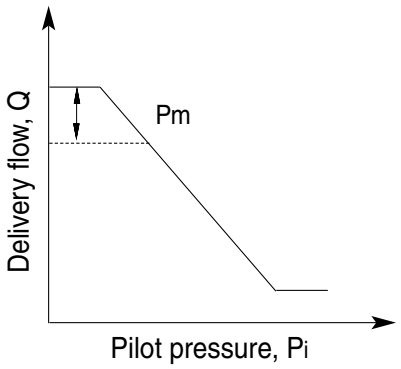


① Functional explanation

As shown in the figure, the pilot pressure Pm switches the maximum flow in two steps.

When the pilot pressure Pm is given, it is admitted to the lefthand side of the piston QMC(648). The piston QMC moves the stopper(647) and pilot piston(643) to the right, overcoming the force of the pilot spring(646), thereby reducing the delivery flow of the pump.

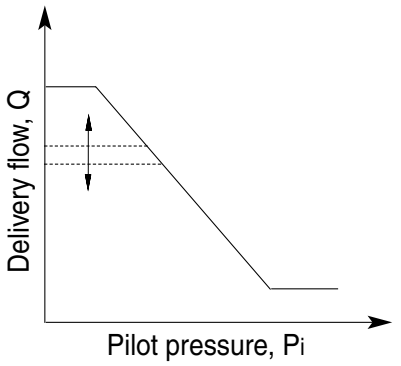
Since the adjusting screw QMC(642) is provided with a flange, the piston QMC stops upon contact with the flange, and the position of the pilot piston at this time determines the maximum flow of the pump.



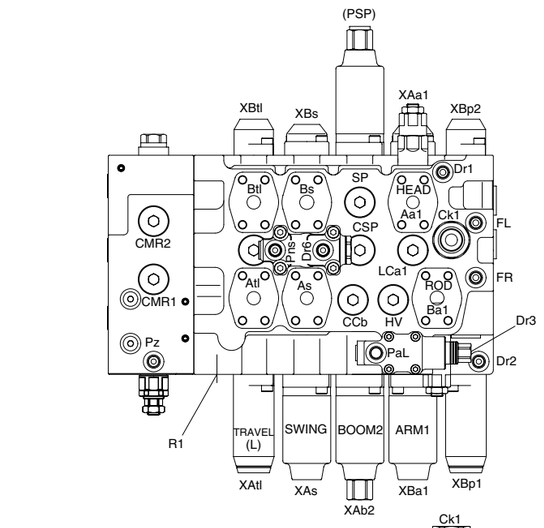
② Adjustment of Qmax cut flow

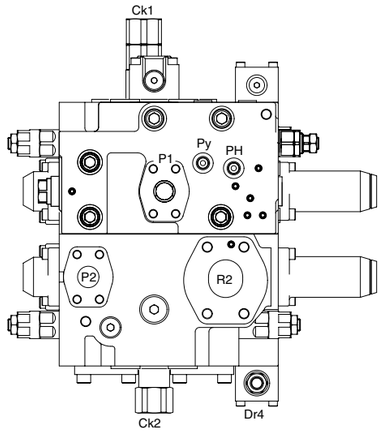
Adjust it by loosening the hexagon nut(801) and by tightening(or loosening) the adjusting screw QMC(642).

Tightening the screw decreases the Qmax cut flow as shown in the figure.

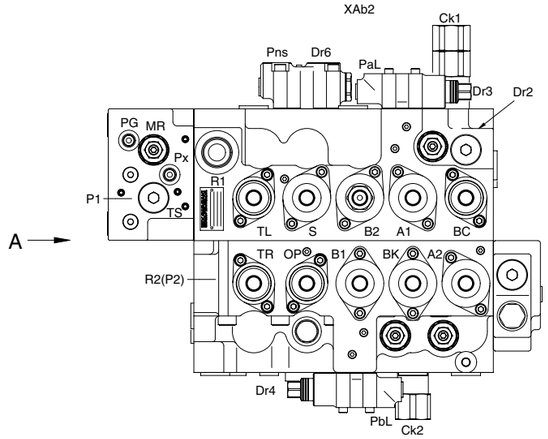


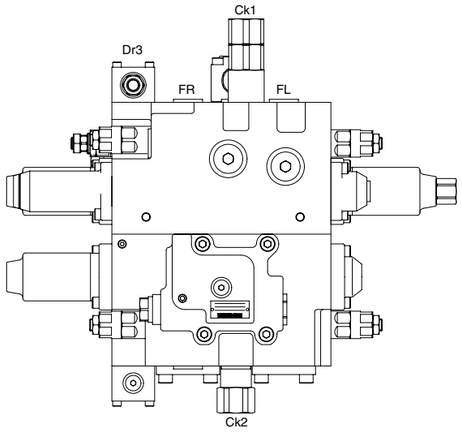
1. STRUCTURE

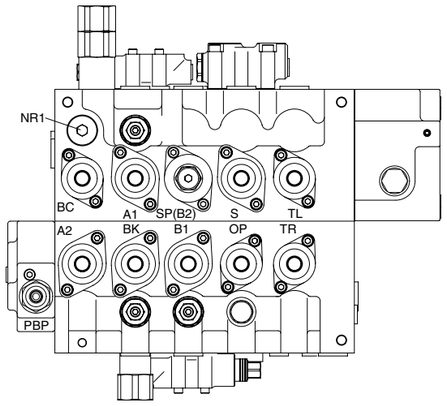


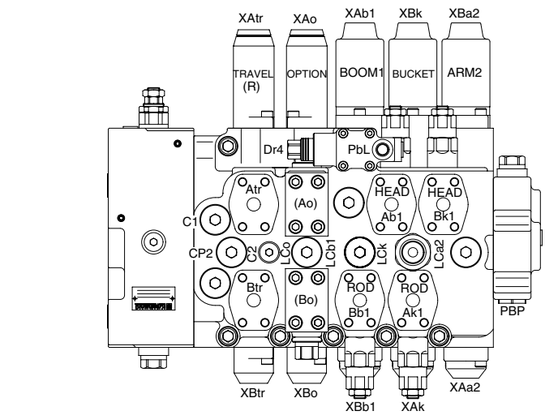


VIEW A



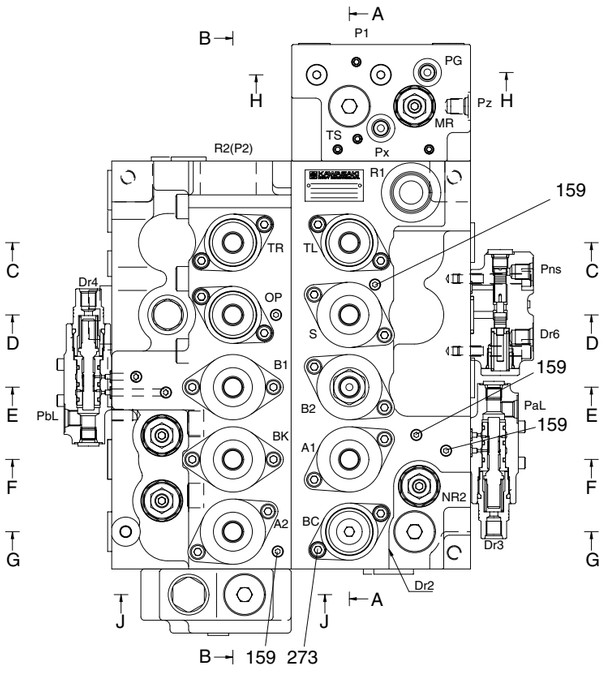


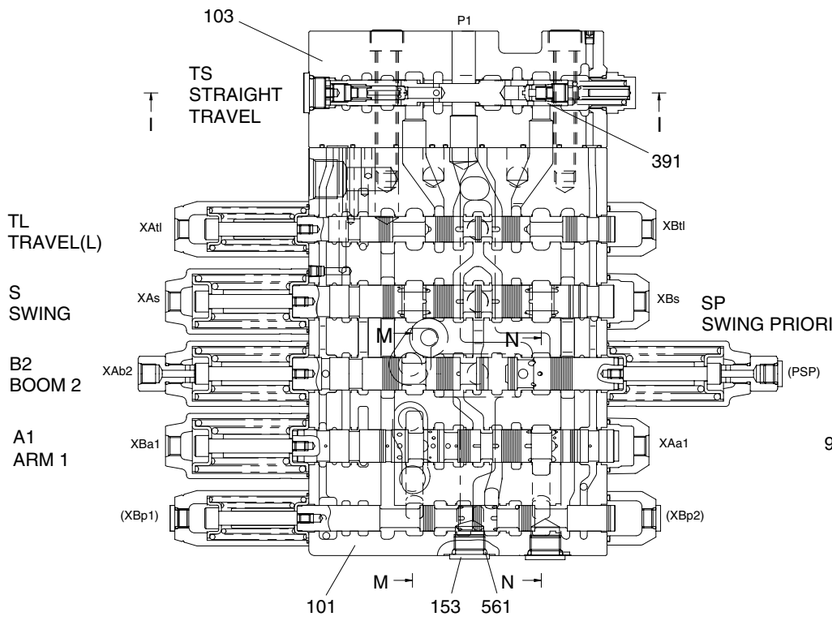




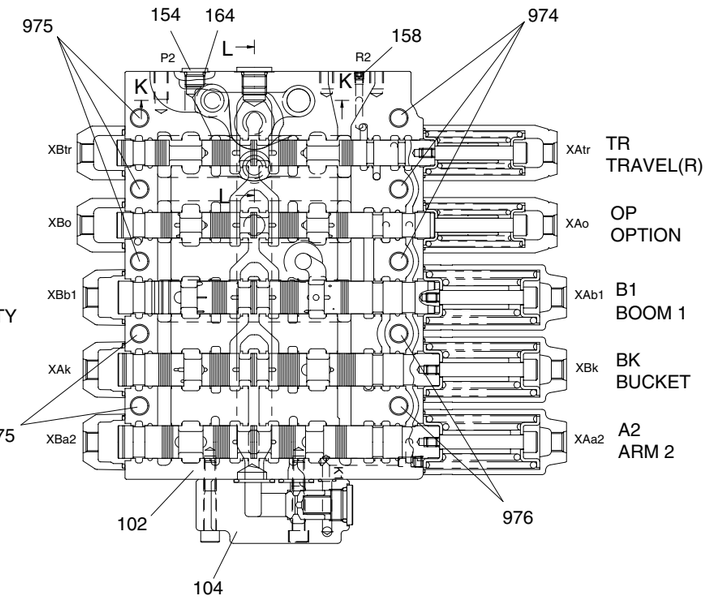
29072MC01

Mark	Port name	Port size	Tightening torque
R1	Make up port for swing	PF 1	20~25kgf · m (115~180lbf · ft)
Ck1 Ck2	Bucket in confluence port Bucket in confluence port	PF 3/4	15~18kgf · m (109~130lbf · ft)
XAtr XBtr XAo XBo XAk XBk XAb1 XBb1 XAa2 XBa2 XAtl XBtl XAs XBs XAa1 XBa1 XAb2 (Psp) (XBp1) (XBp2)	Travel right pilot port Travel right pilot port Option pilot port Option pilot port Bucket out pilot port Bucket in pilot port Boom up pilot port Boom down pilot port Arm in confluence pilot port Arm out confluence pilot port Travel left pilot port Travel left pilot port Swing pilot port Swing pilot port Arm in pilot port Arm out pilot port Boom up confluence pilot port (Swing priority pilot port) (Bucket in confluence pilot port) (Drain port)	PF 3/8	7~8kgf · m (50.6~57.8lbf · ft)
Pz Py PG PH Px Dr1 Dr2 Dr3 Dr4 Dr6 FL FR Pns PaL PbL PBP	Main relief pilot pressure Signal port for travel Pilot pressure port Pilot pressure port Signal for other acutuators Drain port Drain port Drain port Drain port Drain port Negative control signal port(P1 port side) Negative control signal port(P2 port side) Swing logic valve pilot port Lock valve pilot port Lock valve pilot port Drain port	PF 1/4	3.5~3.9kgf · m (25.3~28.2lbf · ft)
Atr Btr (Ao) (Bo) Ak1 Bk1 Ab1 Bb1 Atl Btl As Bs Aa1 Ba1 P1 P2	Travel motor right side port Travel motor right side port Option port Option port Bucket rod side port Bucket head side port Boom head side port Boom rod side port Travel motor left side port Travel motor left side port Swing motor port Swing motor port Arm head side port Arm rod side port Pump port(P1 side) Pump port(P2 side)	M10	5~6.6kgf · m (36.1~47.7lbf · ft)
R2	Return port	M12	8.5~11.5kgf · m (61.5~83.1lbf · ft)

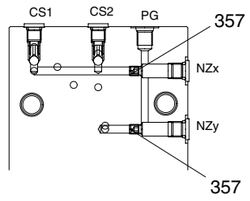




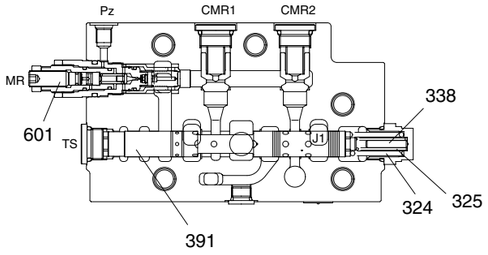
SECTION A-A



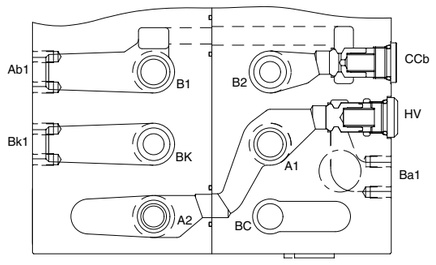
SECTION B-B



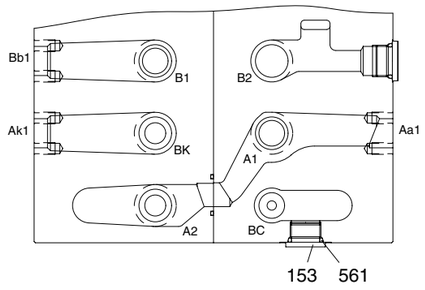
SECTION H-H



SECTION I-I



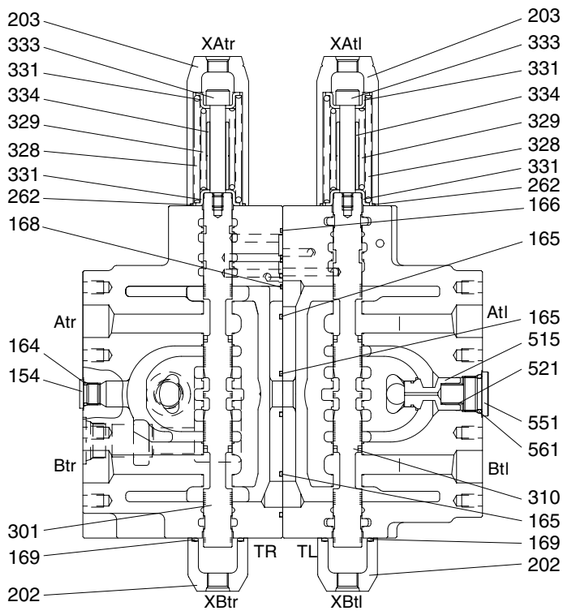
SECTION M-M



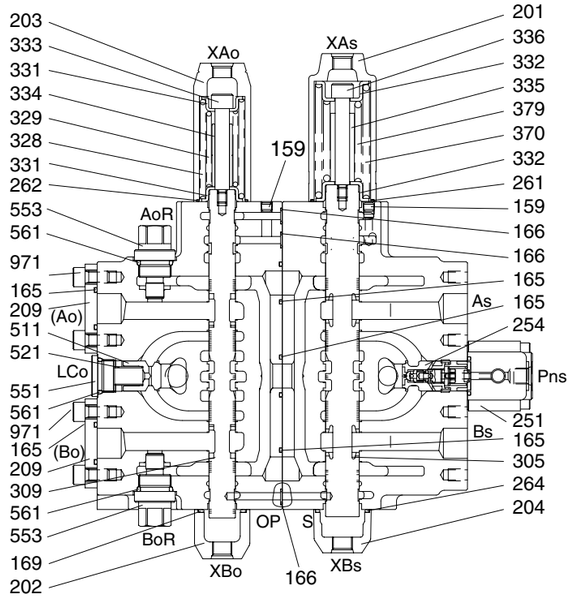
SECTION N-N

29072MC07

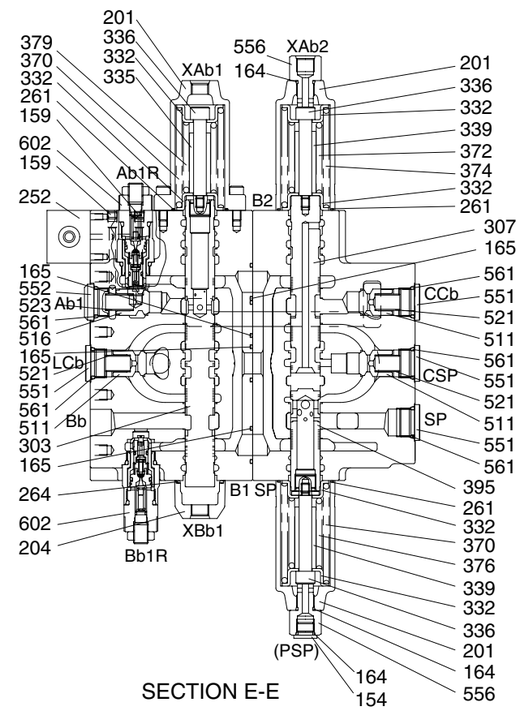
101	Casing A	301	Travel spool	379	Spring
102	Casing B	302	Arm 1 spool assy	391	Straight travel spool
103	Straight travel valve	303	Boom 1 spool assy	392	Bypass cut spool
104	Boom priority valve	304	Bucket spool	395	Swing priority spool
153	Plug	305	Swing spool	401	Spool
154	Plug	306	Arm 2 spool	424	Spring
155	Plug	307	Boom 2 spool	425	Spring
156	Plug	309	Spool(Option)	438	Rod
158	Plug	310	Travel spool	511	Poppet
159	Plug	324	Spring	513	Poppet
163	O-ring	325	Spring	515	Poppet
164	O-ring	328	Spring	516	Poppet
165	O-ring	329	Spring	521	Spring
166	O-ring	331	Spring seat	522	Spring
167	O-ring	332	Spring seat	523	Spring
168	O-ring	333	Bolt	551	Plug
169	O-ring	334	Stopper	552	Plug
201	Cover	335	Stopper	553	Plug
202	Cover	336	Bolt	554	Plug
203	Cover	337	Stopper	555	Check valve assembly
204	Cover	338	Stopper	556	Plug
209	Flange	339	Stopper	561	O-ring
251	Control valve assy	357	Orifice	601	Main relief valve
252	Lock valve assy	370	Spring	602	Port relief valve
254	Swing logic valve assy	372	Spring	611	Nega-con relief valve
261	O-ring	374	Spring	971	Socket screw
262	O-ring	376	Spring	974	Socket screw
264	O-ring	377	Spring	975	Socket screw
273	Socket screw	378	Spring	976	Socket screw



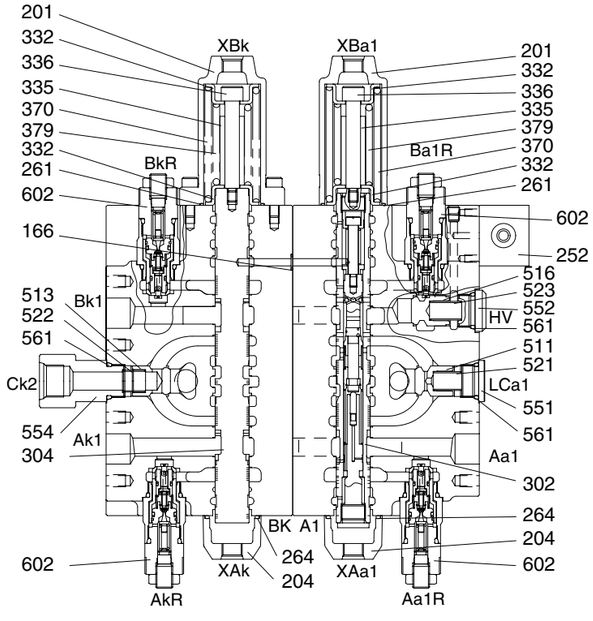
SECTION C-C



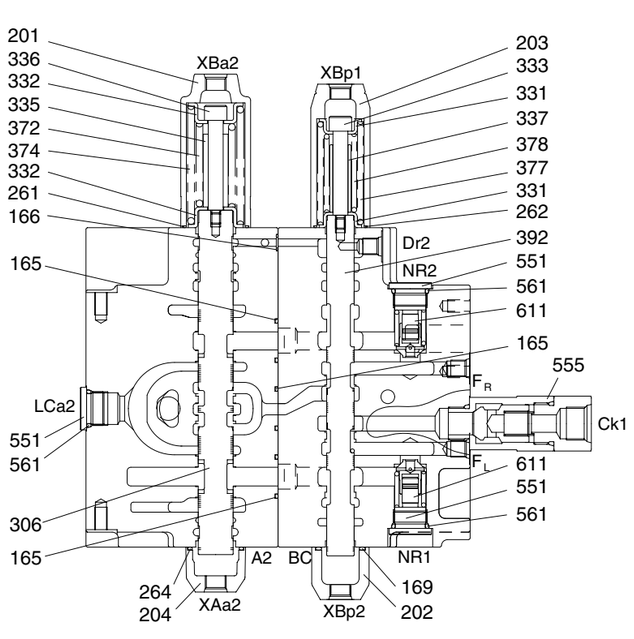
SECTION D-D



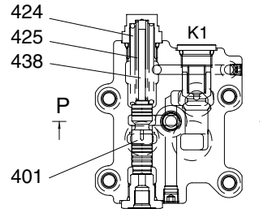
SECTION E-E



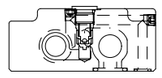
SECTION F-F



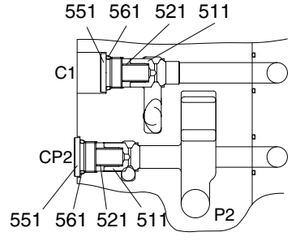
SECTION G-G



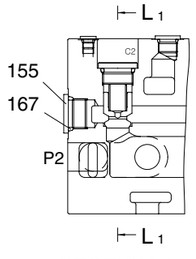
SECTION J-J



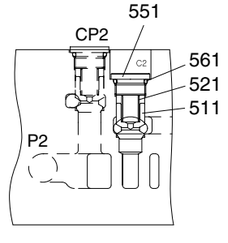
SECTION P-P



SECTION K-K



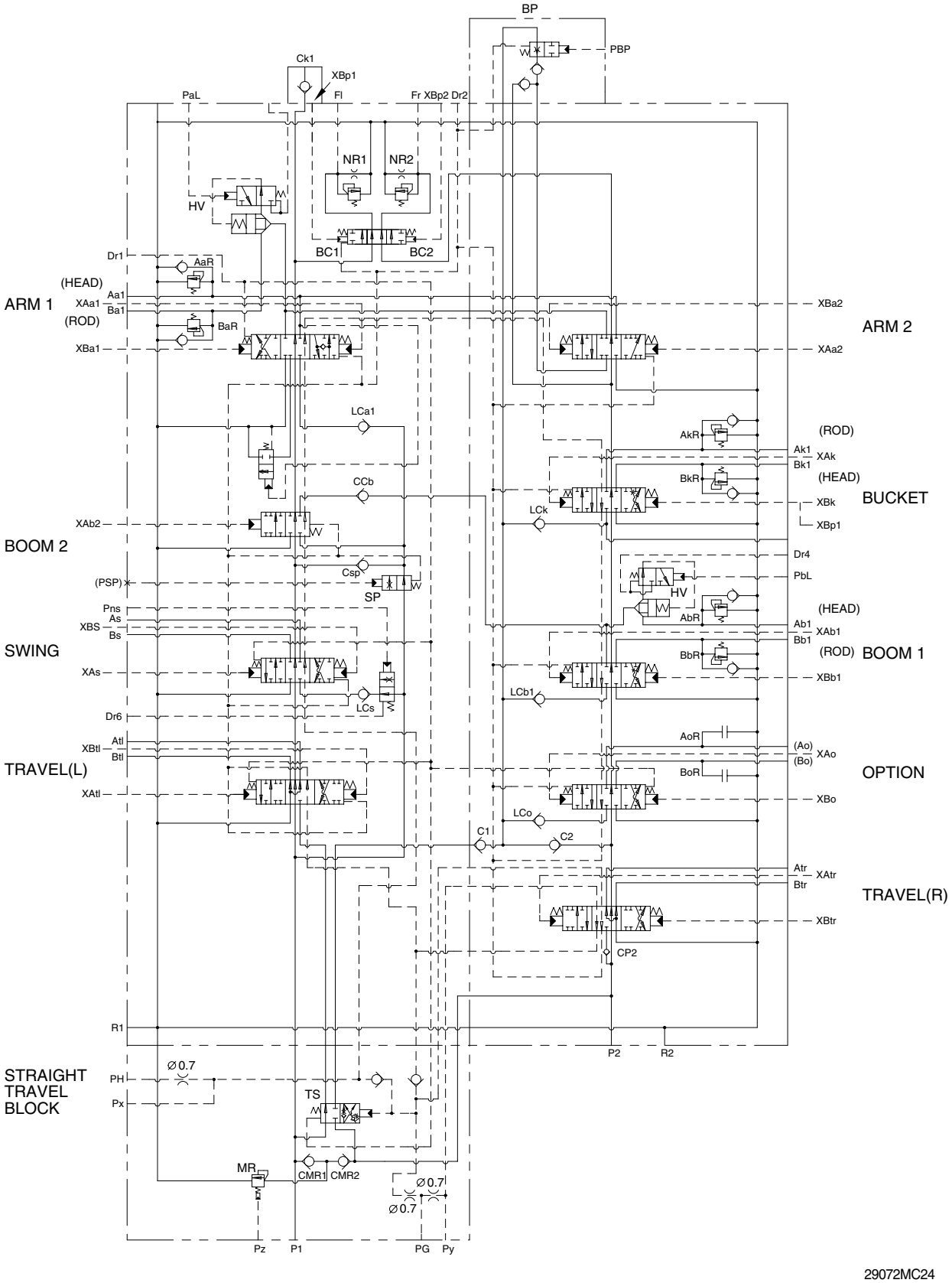
SECTION L-L



SECTION L 1 -L 1

29072MC14

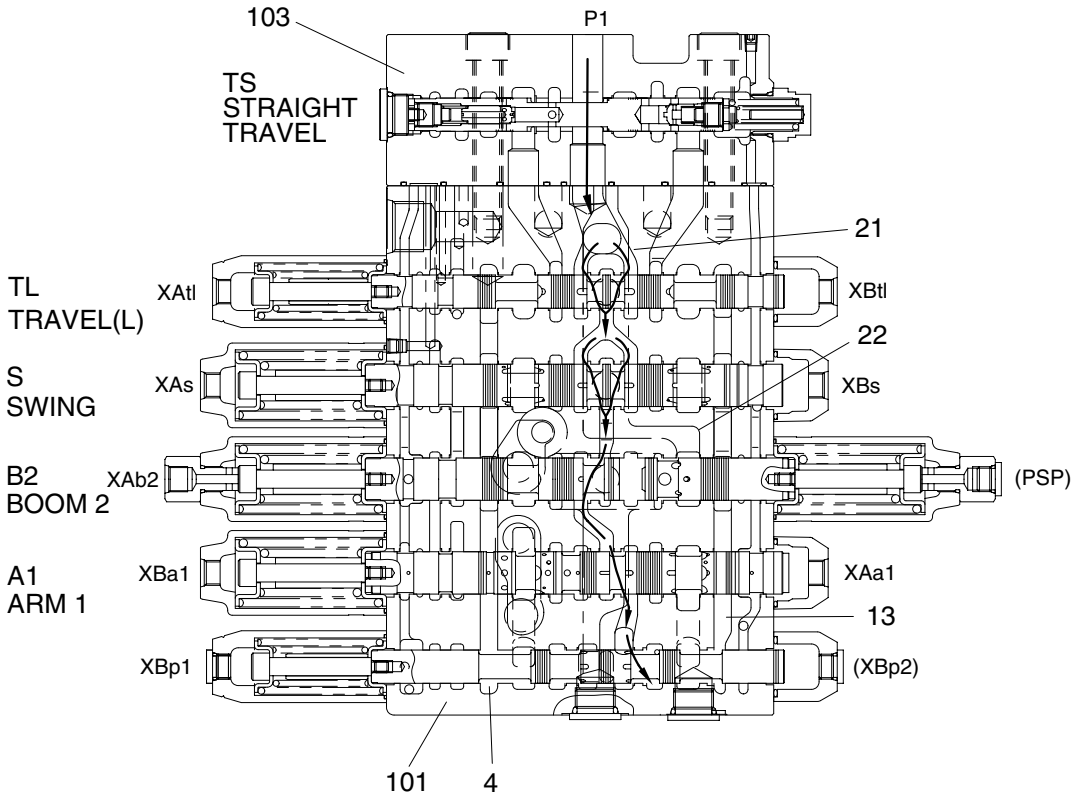
2. HYDRAULIC CIRCUIT



29072MC24

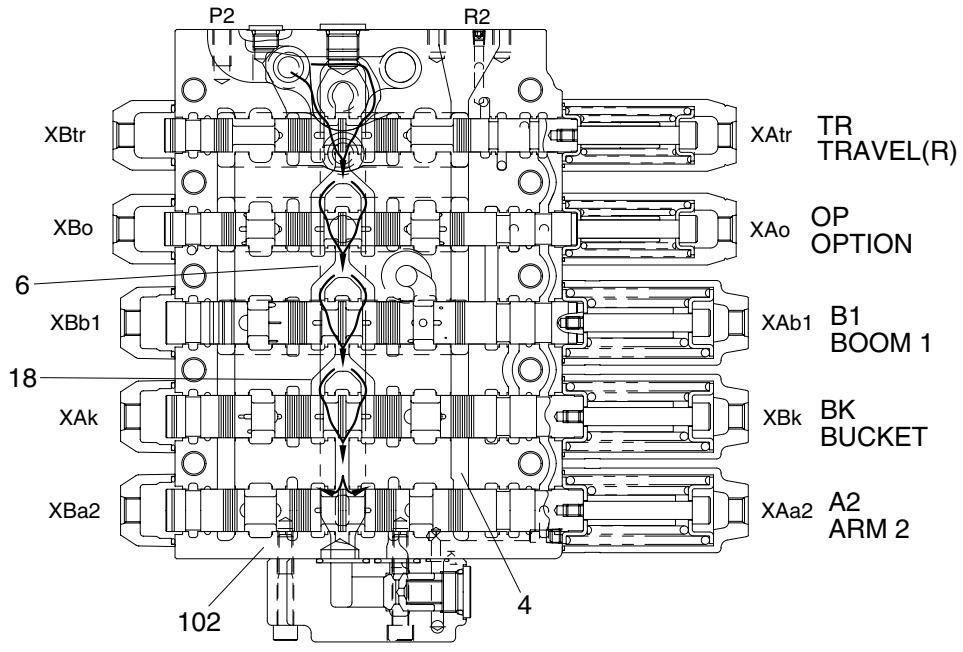
3. FUNCTION

1) CONTROL IN NEUTRAL POSITION



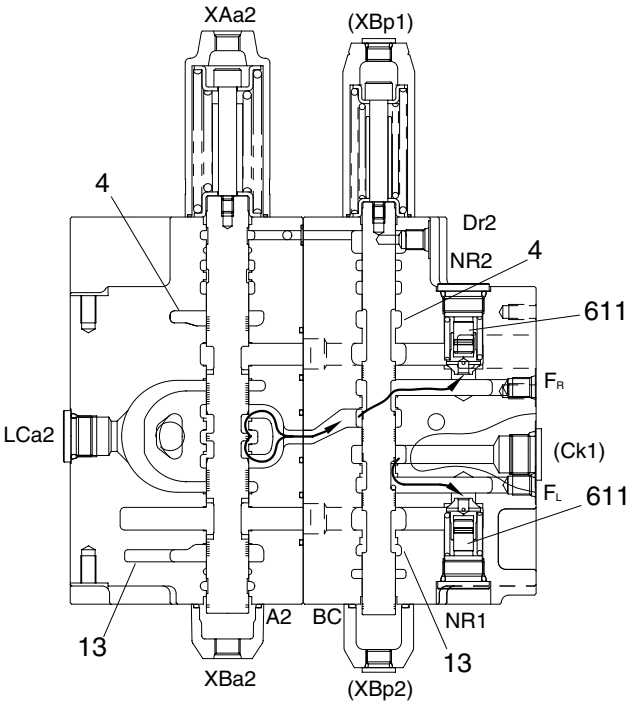
29072MC25

SECTION A-A



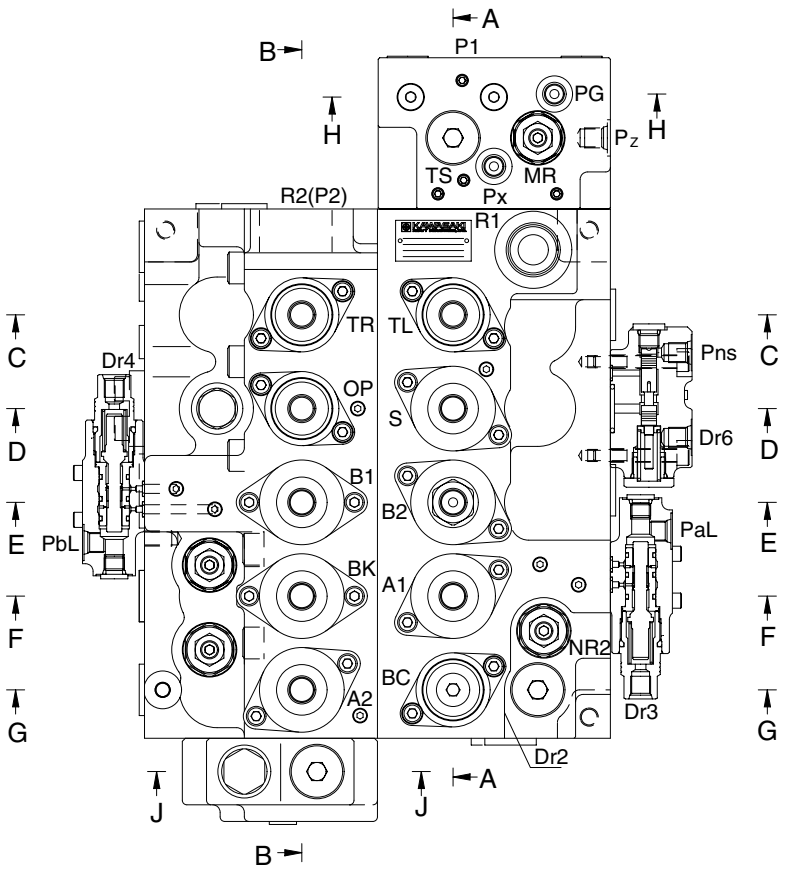
29072MC26

SECTION B-B



29072MC27

SECTION G-G

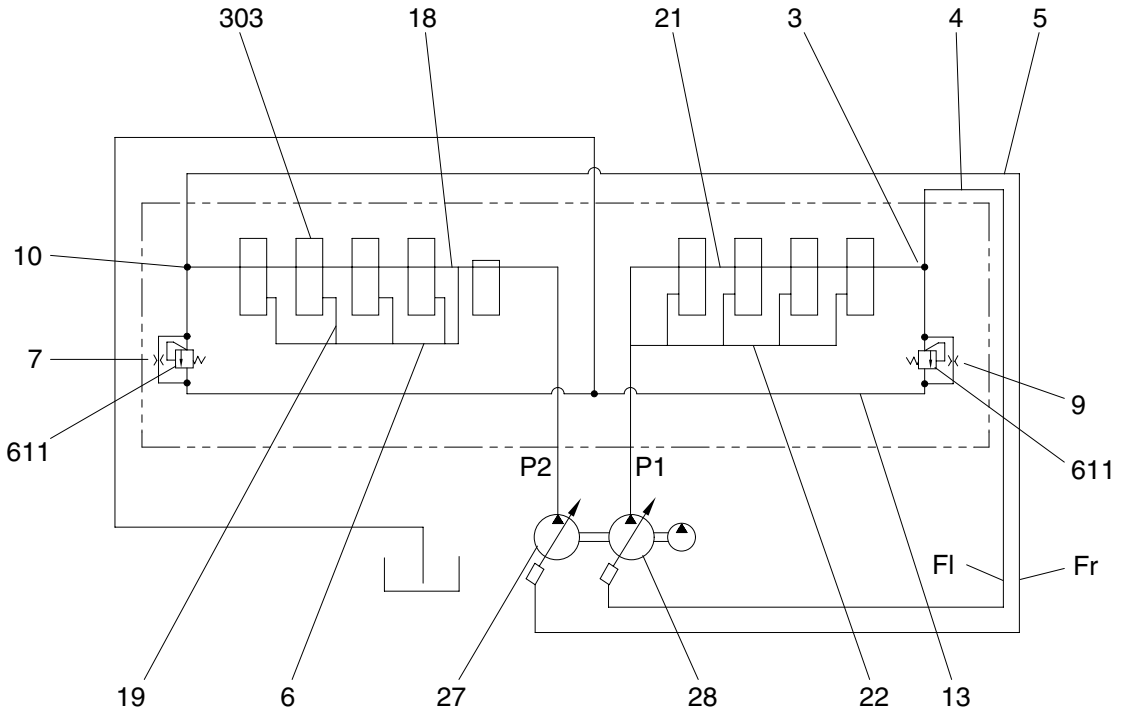


29072MC28

The hydraulic fluid from the pump P1 flows into casing A(101) through the inlet port(P1), through the center bypass(21) and the parallel path(22). The hydraulic fluid from the pump P2 flows into casing B(102) through the inlet port(P2) through the center bypass(18) and the parallel path(6).
The hydraulic fluid from the pump P1 is directed to the tank through the center bypass(21), negative control orifice(NR1), the return path(13) and the return port(R2). The hydraulic fluid from the pump P2 also flows to the tank through the center bypass(18), negative control orifice(NR2), return path (4) and return port(R2). The hydraulic fluid in paths (6) and (22) is blocked and cannot return to the tank.
In case a control lever is operated, the hydraulic fluid from the pump P2 is supplied to the travel right spool(301) from path(18) and to the spools: option(309), boom1(303), bucket(304) and arm2(306) from path(6). Additionally, the hydraulic fluid from the pump P1 is supplied to the travel left spool (310) from path(7) while the swing(305), boom2(307) and arm(302) spools are supplied from path(22).

2) NEGATIVE CONTROL

(1) General operation

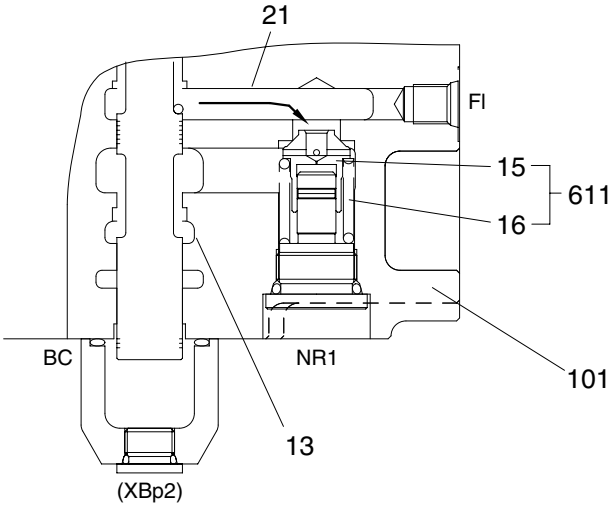


29072MC29

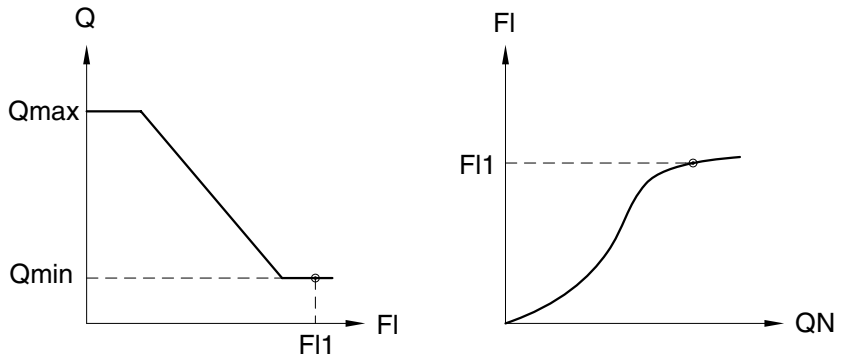
The negative control signal pressure from the center bypass(18, 21) occurs in the following cases and controls the discharge of the pump.

1. Neutral condition when no function is being actuated.
2. The pilot control lever is partially operated.

The hydraulic fluid of the pump P1(28) flows into the return passage(13) through the center bypass (21), the path(3) and orifice(9)(Within the poppet(15)). The restriction caused by this orifice thereby pressurizes path(3). This pressure is transferred as the negative control signal pressure Fl to the pump P1 regulator through the negative control line(4). It controls the pump regulator so as to decrease the discharge of the pump P1(28).



29072MC30

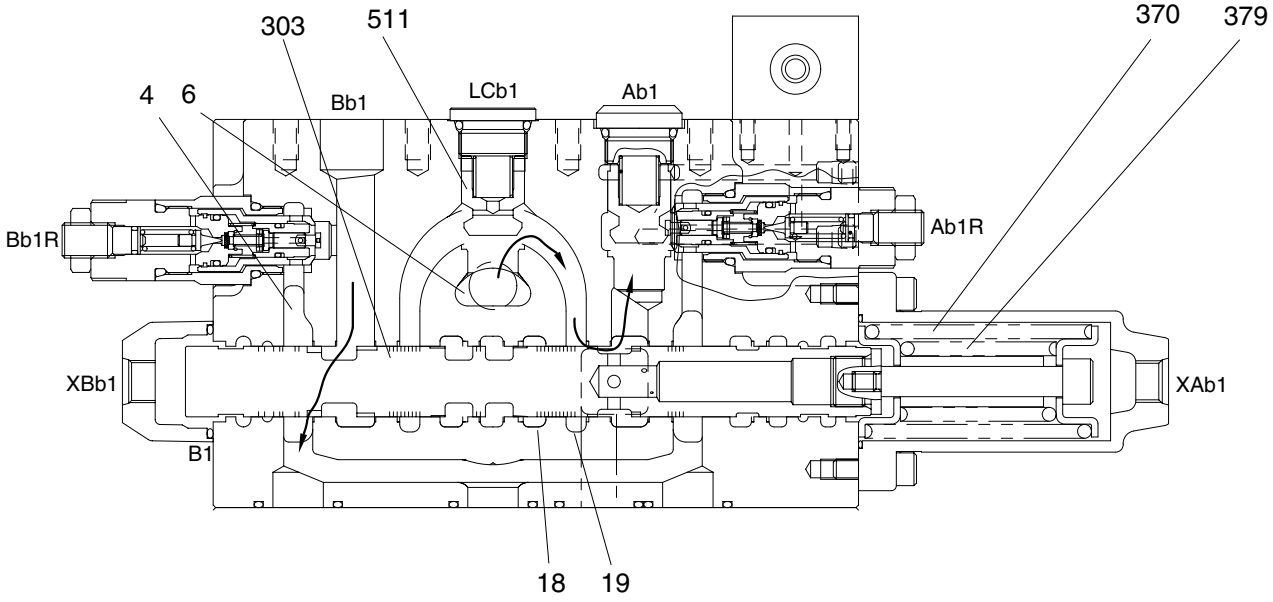


29072MC31

The negative control relief valve(611) consists of poppet(15), spring(16) and casing(101). When the hydraulic fluid in the center bypass increases to the level that the pressure in the path(3) reaches the set pressure of the spring(16), the hydraulic fluid in the path(3) pushes open the poppet (15) and escapes into the return path(13).

In the unloaded state, the hydraulic fluid of the pump P1(28) entirely flows to the tank through the path(21), orifice(9) and the return path(13). Therefore the pressure Fl in the path(3) becomes maximum(Fl1) because all the discharge is reduced by the orifice(9) which in turn destrokes the pump P1(28) so as to minimize the tilting angle and consequent discharge of the pump P1(28). (Qmin)

(2) Negative control(With fine metering)



29072MC32

In the case, for example, when the pilot control lever for main boom is slightly operated, the pilot pressure XAb1 shifts the main boom spool(303) partially in the left direction. So the path(19) is partially opened and the center bypass(18) is shut slightly. The hydraulic fluid thereby separates. One part flows via the orifice(7) through the path(18) and the other portion flows into the parallel path(6), the path(19) and the port Ab1. The flow from the path(18) through the orifice(7) decreases slightly and the pressure Fr in the path(10) thereby also slightly decreases. As the pressure Fr becomes lower, the discharge of the pump P2(27) increases. With the pilot control lever shifted even more the path(18) is shut off by the shifting of the spool(303) and then the flow through the bypass becomes zero. The pressure in the path(10) becomes zero and the discharge of the pump P2(27) becomes maximum.(Qmax)

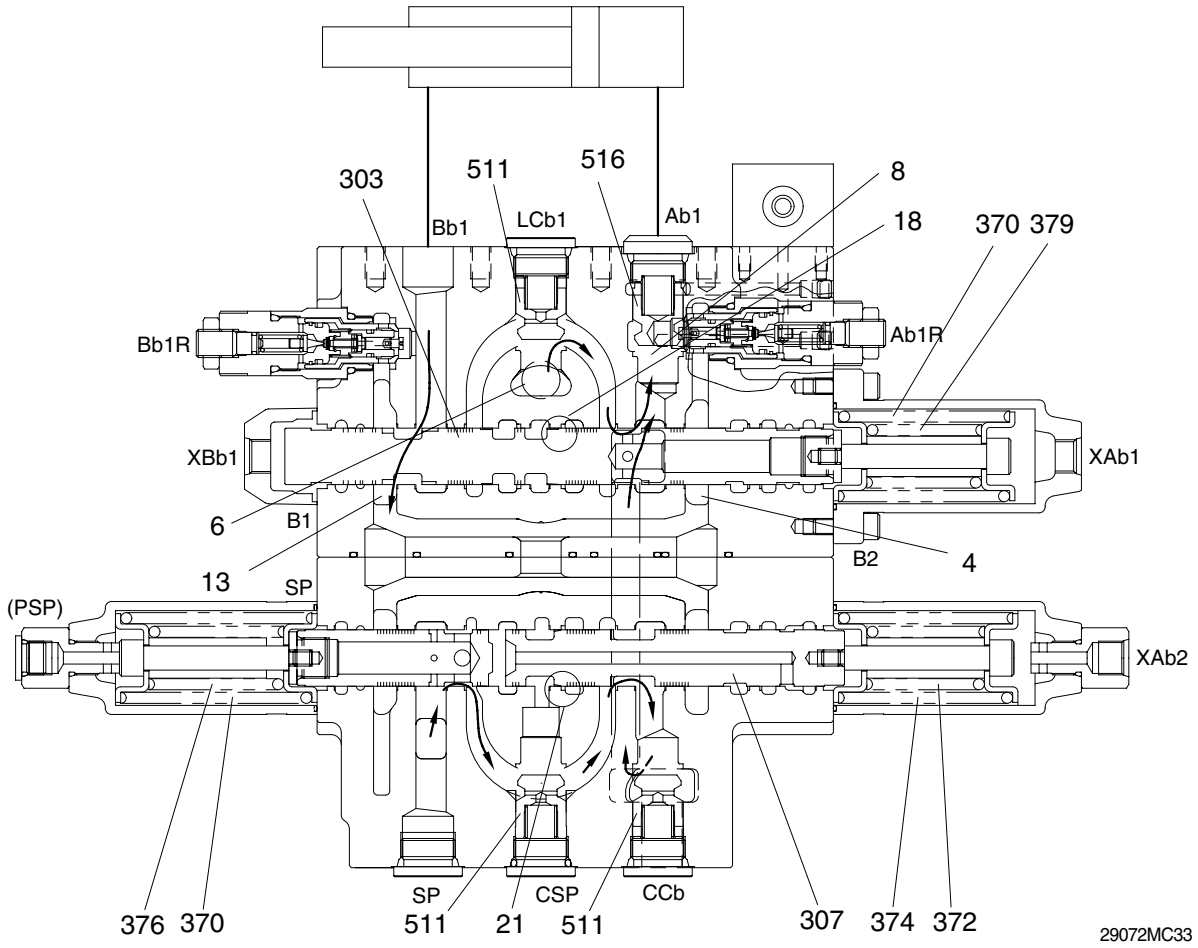
Because the discharge of the pump is adjusted by operating the pilot control lever slightly, the precise moving of the actuator is realized.

For the pump P1(28) the same negative control principle of operation occurs utilizing the orifice(9).

3) EACH SPOOL OPERATION

(1) Boom control

① Boom up operation

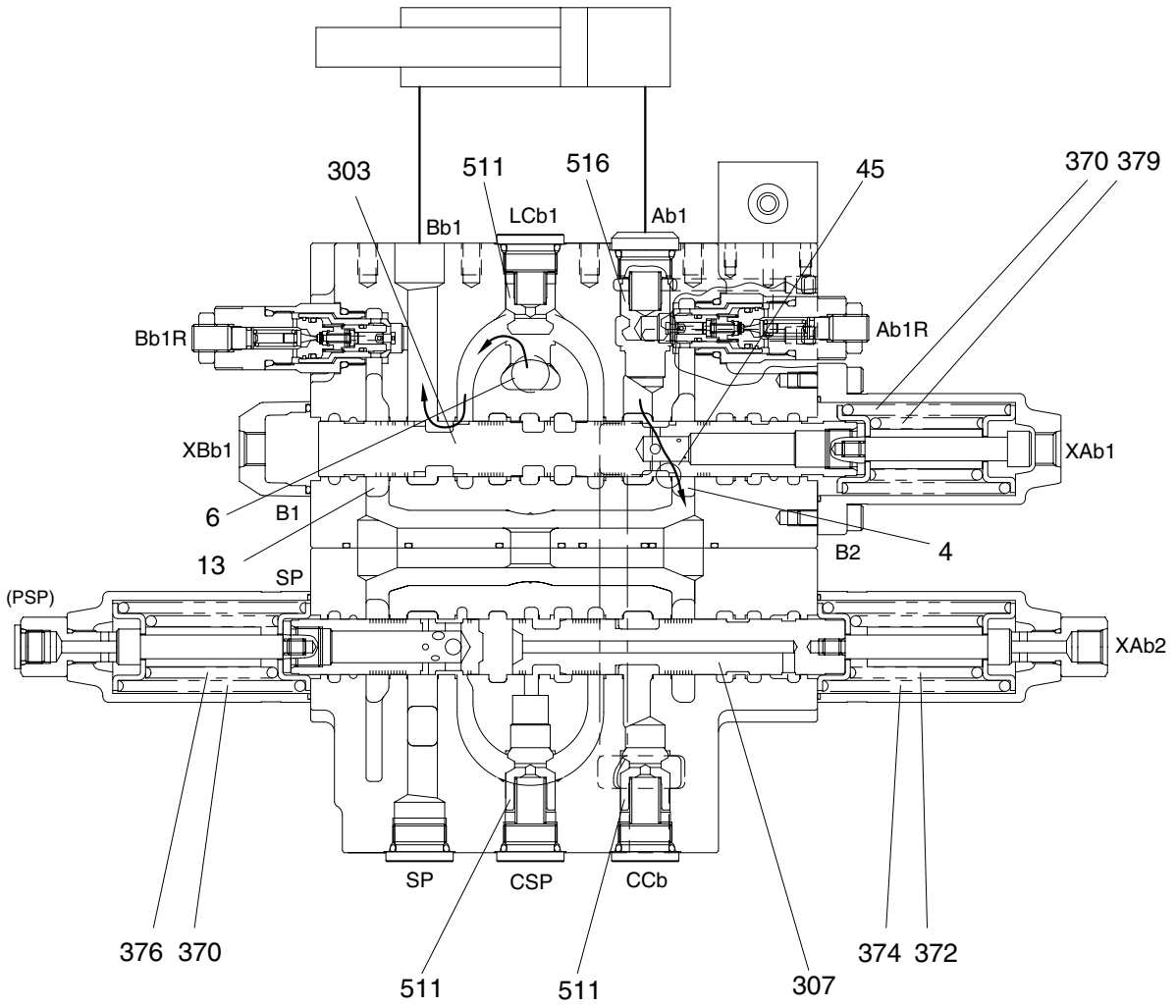


The main boom up operation becomes fast because the hydraulic fluid from the pump P2 that is directed to the port P2 is combined in the casing that of the pump P1 which enters port P1. The confluence flow is supplied to the head side of the boom cylinder. In low speed operation, only the boom1 spool(303) operates and is supplied with hydraulic fluid from the pump P2.

The hydraulic fluid from the pump P2 flows into the boom1 spool(303) through port P2 and parallel path(6). The hydraulic fluid from the pump P1 flows to the boom2 spool(307) through pump port P1 and the parallel path(22).
During the boom up operation, the pilot pressure from the pilot control valve is supplied into the port XAb1 and shifts the boom1 spool(303) in the left direction against the springs (370) and (370). The hydraulic fluid from the pump P2 enters the parallel path(6) and then passes through the load check valve LCb1(511) and boom1 spool(303) and check valve HV(516) then flows into the port Ab1. Following this it flows into the head side of the boom cylinder.

At the same time, the pilot pressure through the port XAb2 shifts the boom2 spool(307) in the left direction against the springs (374) and (372). The hydraulic fluid from the pump P1 enters via the parallel path(22) and center bypass(21), then passes through the load check valve CSP(511), boom2 spool(307) and the load check valve CCb(511). Then flows combine in path(8) and are directed to port Ab1 and the head side of the boom cylinder.
The flow from the rod side of the boom cylinder returns to the boom1 spool(303) through the port Bb1. Thereafter it is directed to the return port R2 through path(13).

② Boom down operation



During the boom down operation, the pilot pressure from the pilot control valve is supplied to port XBb1 and PbL and shifts the boom1 spool(303) in the right direction against the springs (370) and (379).

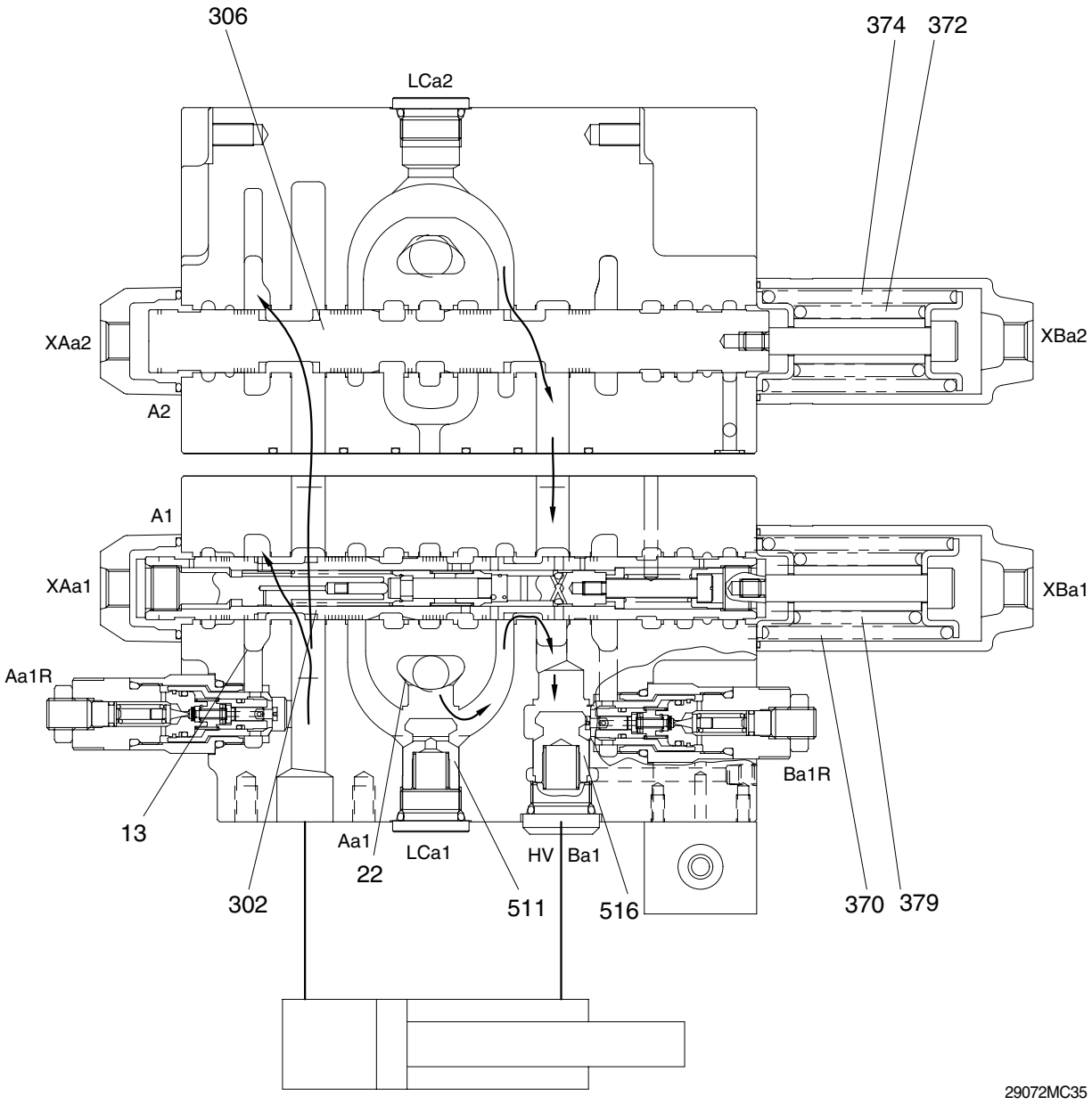
The hydraulic fluid from the pump P2 enters the parallel path(6) and is directed to the port Bb1 through the load check valve LCb1(511). Following this is flows into the rod side of the boom cylinder.

The return flow from the head side of the boom cylinder returns to the boom1 spool(303) through the port Ab1. Thereafter it is directed to the return port R2 through path(4).

Additionally, the return flow is restricted in path(45), which lowers the boom cylinder at a suitable speed.

(2) Arm control

① Arm roll out operation

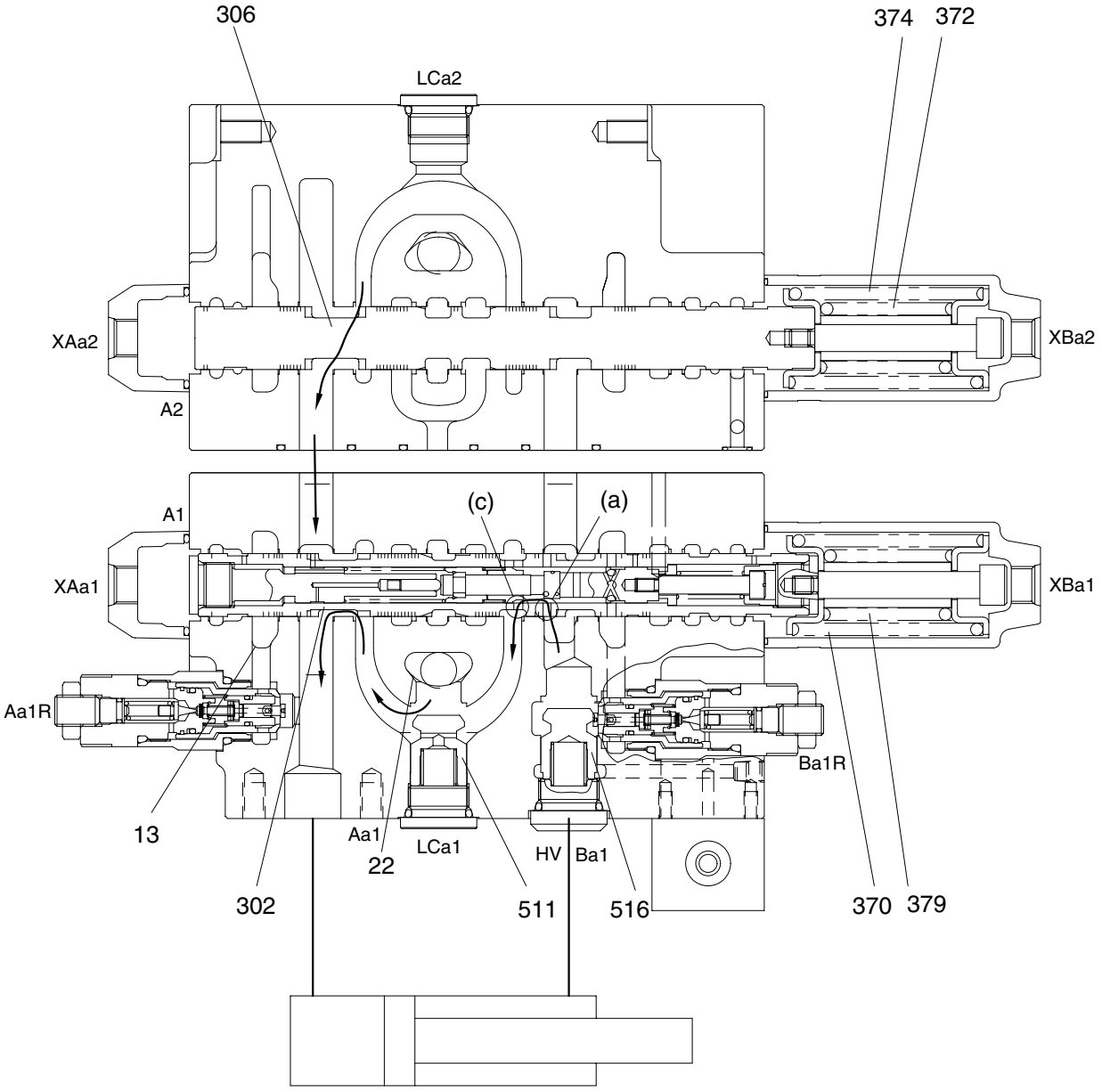


During the arm roll out operation, the pilot pressure from the pilot control valve is supplied to the pilot ports(XBa1& XBa2) and shifts the arm1 spool(302) in the left direction against the springs (370) and (379) and shifts the arm2 spool(306) in the left direction against the springs (374) and (372). The hydraulic fluid from the pump P1 flows through the load check valve LCa1(511), lock valve HV(516), and then through parallel path(22). It is then directed to the rod side of the arm cylinder through the port Ba1.

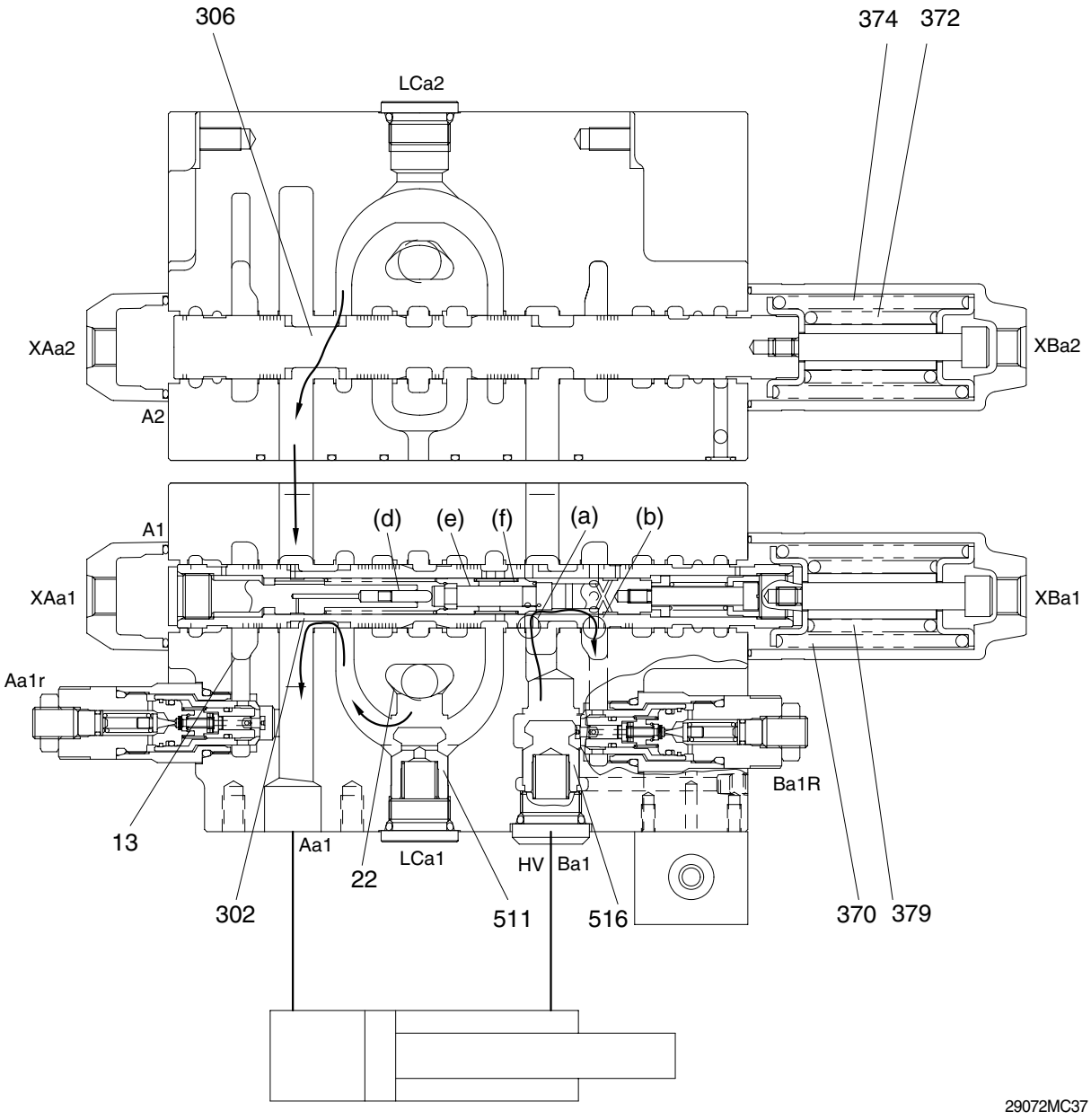
At the same time, the pilot pressure through the port XBa2 shifts the arm2 spool(306) in the left direction against the springs (374) and (372). The hydraulic fluid from the pump P2 enters via the parallel path(22) and center bypass(21), then passes through the check valve of the boom priority valve(104), arm2 spool(306). The flows are combined and directed to port Ba1 and the rod side of the arm cylinder. The flow from the head side of the arm cylinder returns to the arm1 spool(302) through the port Aa1. Thereafter it is directed to the return port R2 through path(13).

② Arm roll in operation

· During light load only



· The pressure in the arm cylinder head side increases

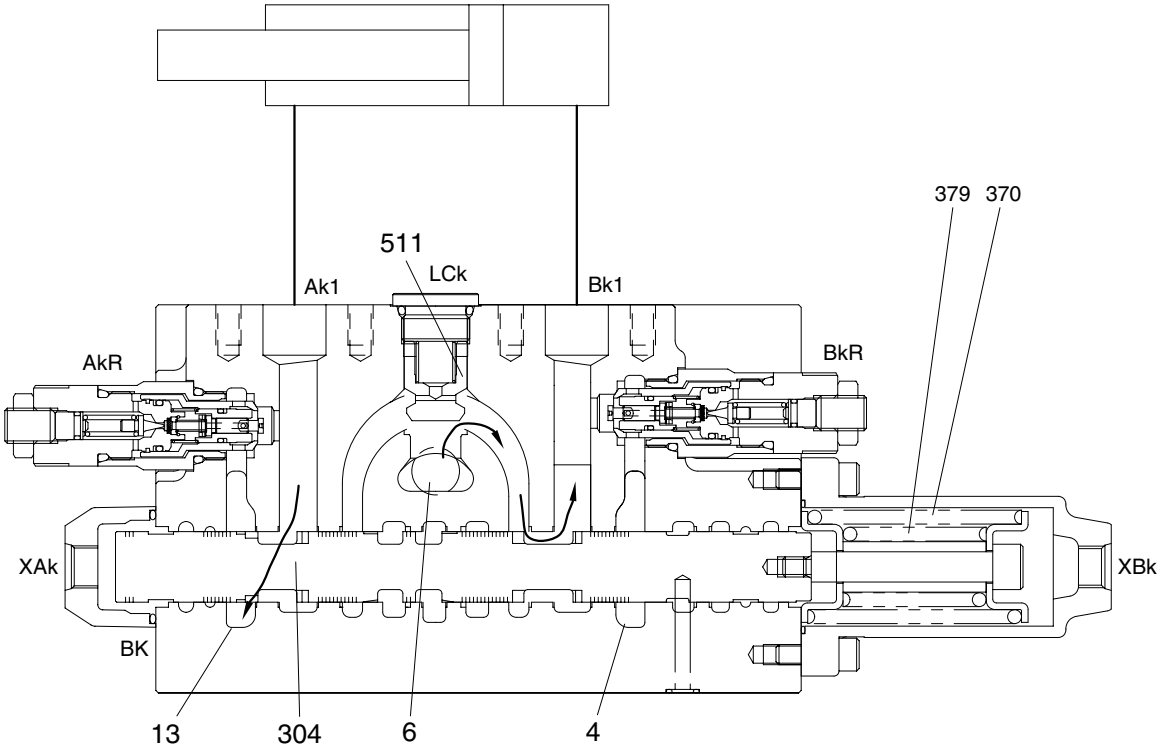


During the arm roll in operation, the pilot pressure from the pilot control valve is supplied to the ports XAa1, XAa2 and PaL and shifts the arm1 spool(302) in the right direction against the springs (370) and (379) and shifts the arm2 spool(306) in the right direction against the springs (384) and (372).
During the arm roll in operation, the hydraulic fluid from the pump P1 flows into the arm1 spool(302) through the parallel path(22). Then it enters into the head side of the arm cylinder through the load check valve LCa1(511), check valve HV(516) and port Aa1.
At the same time, the hydraulic fluid from the pump P2 flows into the arm2 spool(306) through the parallel path(22). Then it enters into the head side of the arm cylinder through the check valve of boom priority valve(104) and port Aa1.

The return flow from the rod side of the arm cylinder is pressurized by self-weight of arms and so on, and returns to port Ba1. The pressurized oil returning to port Ba1 enters into the arm1 spool through the outside of the arm1 spool. During a light load only, it pushes open the sleeve check valve, flows the parallel path reversely from spool hole(c), and joints into port Aa1. This is called the arm regeneration function.
When the pressure in the arm cylinder head side increases, the piston(d) and sub spool(e) are transferred in the right direction, and at the same time the sleeve check valve(f) is from the arm cylinder rod side enters flow port Ba1 through the periphery hole(a) of the arm1 spool into the spool, flows out through the periphery hole(b) of the spool, and returns through the tank port R2 to the hydraulic oil tank.

(3) Bucket control

① Bucket roll in operation



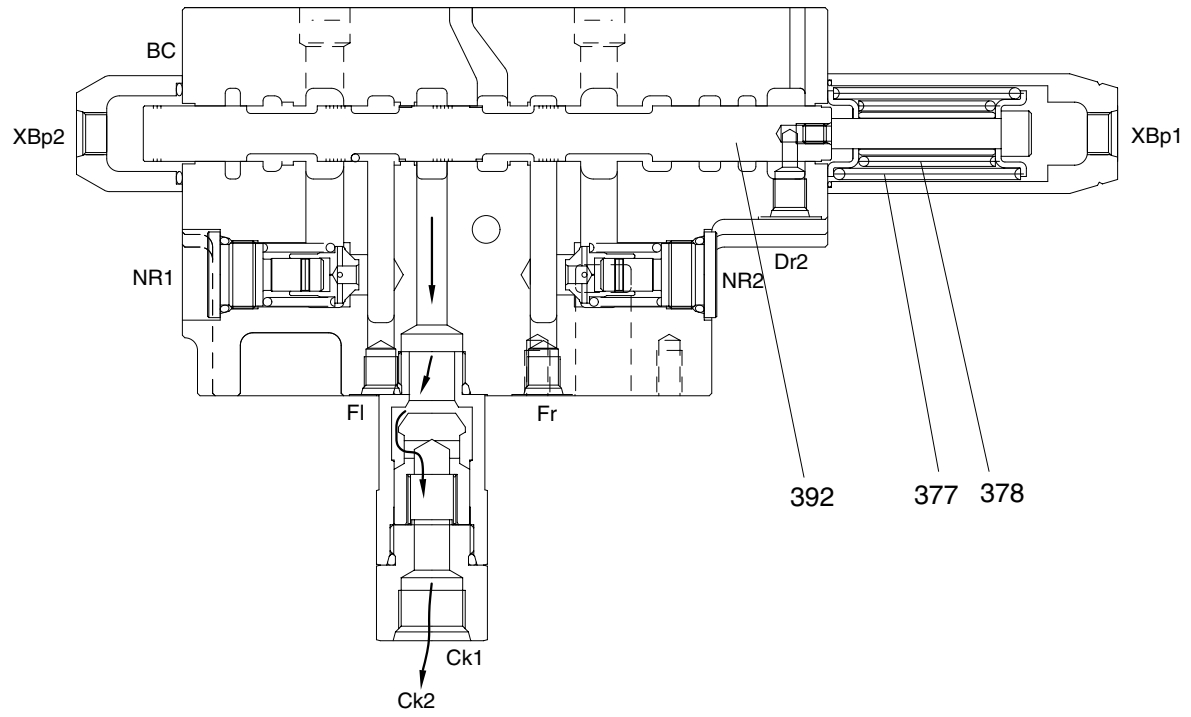
29072MC38

During the bucket roll in operation, the pilot pressure from the pilot control valve is supplied to port XBk and shifts the bucket spool(304) in the left direction against the springs (370) and (379).

The hydraulic fluid from the pump P2 enters the parallel path(6) and is directed to the port Bk1 through the load check valve LCk(511). Following this it flows into the head side of the bucket cylinder.

The retutn flow from the rod side of the bucket cylinder returns to the bucket spool(304) through the port Ak1. Thereafter it is directed to the return port R2 through path(13).

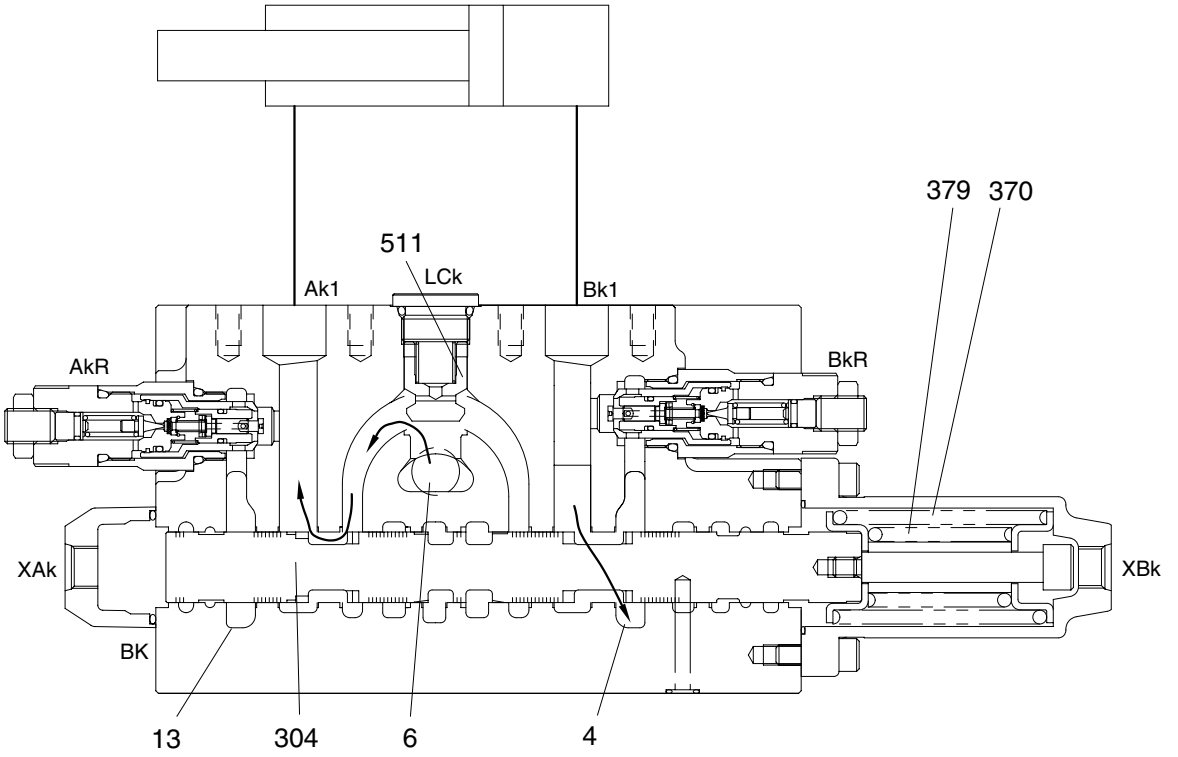
② **Bucket confluence operation**



29072MC40

During the bucket roll in operation, the pilot pressure from the pilot control valve is supplied to port XBp1 and shifts the bypass cut spool(392) in the left direction against the springs(392) in the left direction against the springs (377) and (378).
The hydraulic fluid from the pump P1 enters the center bypass path(21).
But bypass path is shut off by the bypass cut spool. Therefore the hydraulic fluid is directed to port Ck2 after passing through the check valve Ck1.
Then the fluid is directed to the bucket spool(304).

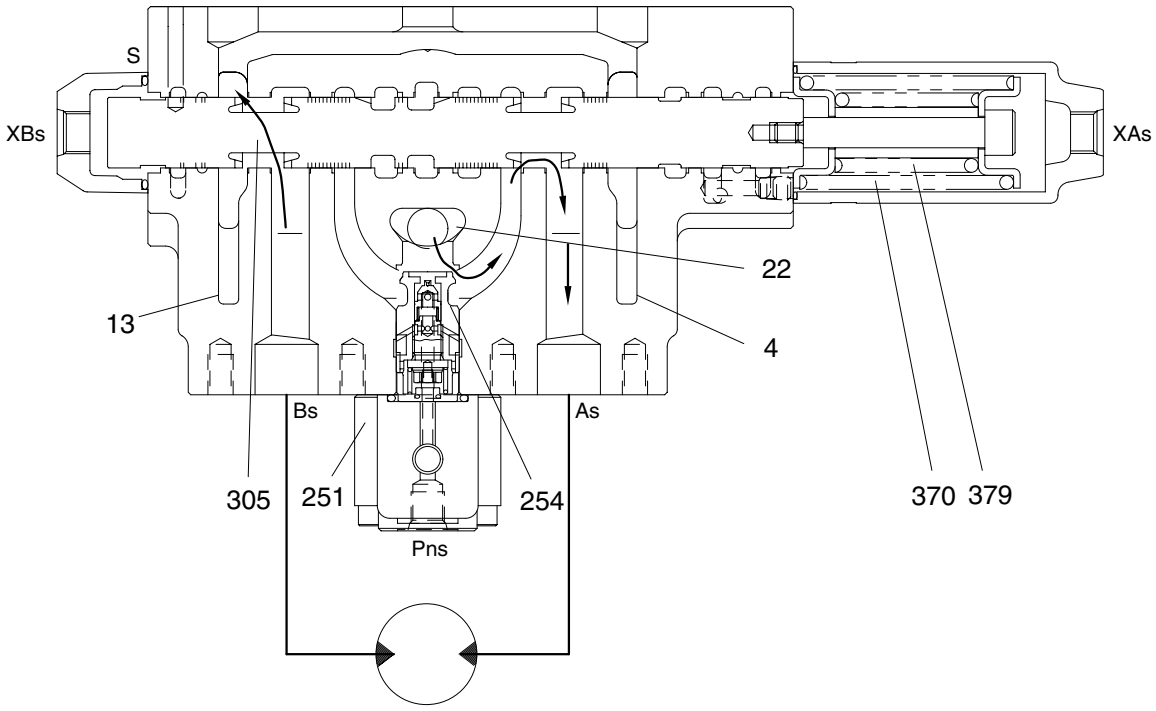
③ Bucket out operation



29072MC39

During the bucket roll out operation, the pilot pressure from the pilot control valve is supplied to port XAk and shifts the bucket spool(304) in the right directed agains the springs (370) and (370).
The hydraulic fluid from the pump P2 enters the parallel path(6) and is directed to the port AK1 through the load check valve LCk(511). Following this it flows into the rod side of the bucket cylinder.
The return flow from the head side of the bucket cylinder returns to the bucket spool(304) through the port Bk1. Thereafter it is directed to the return port R2 through path(4).

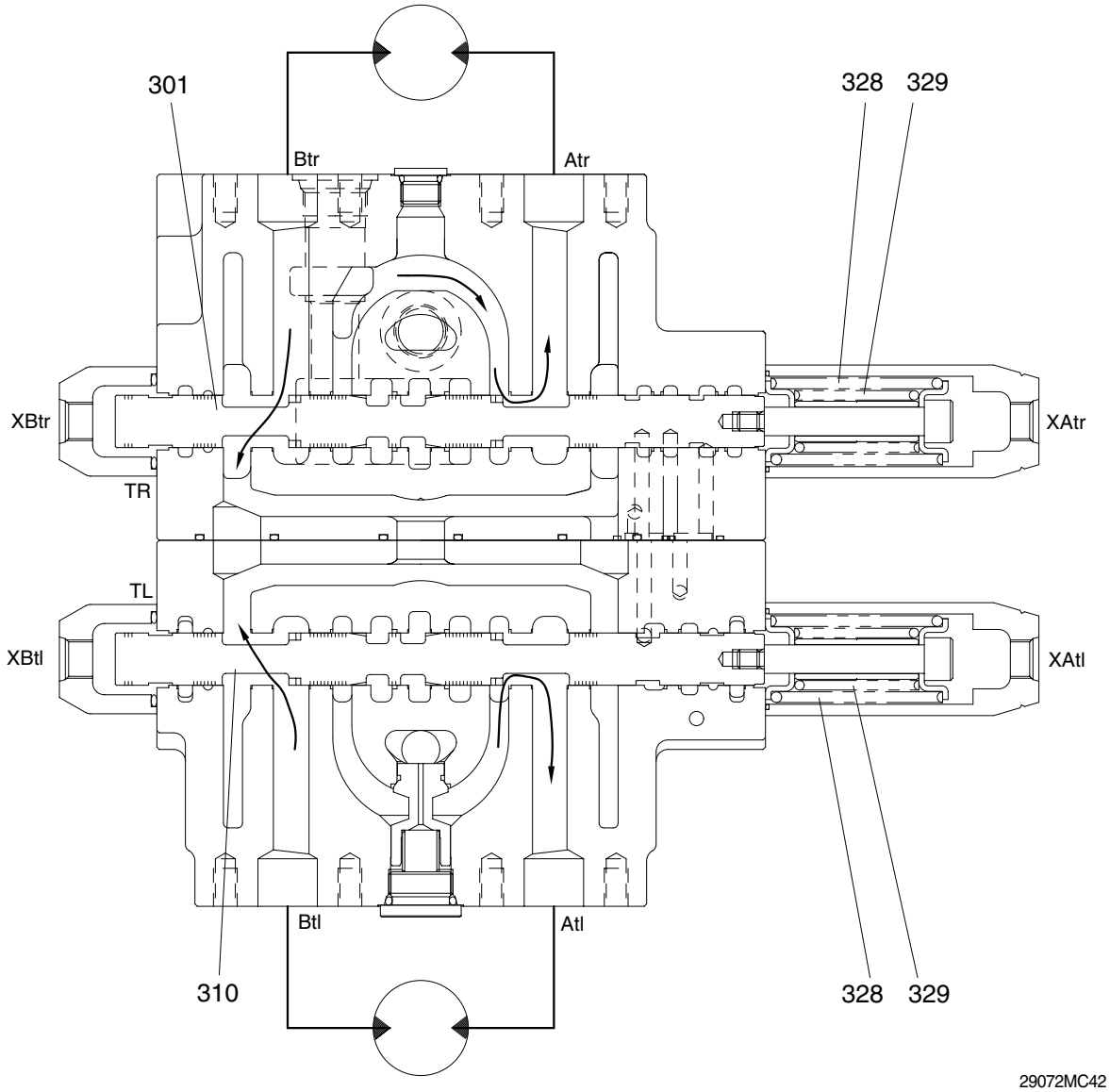
(4) Swing control



During the swing right or left operation, only the hydraulic fluid of the pump P1 is supplied to the swing motor.

The pilot pressure from the pilot control valve is supplied to the port XAs and shifts the swing spool (305) in the left direction against springs (370) and (379). Hydraulic fluid from the pump P1 flows into the swing spool(305) through the parallel path(22). Then it is directed to the swing motor through the check valve LCs(254) and the port As. As a result, the swing motor turns and the return flow from the swing motor enters port Bs. The flow from the motor returns to the tank port R2 through the swing spool(305) and path(13). In the case of the opposite operation, the operation is similar.

(5) Travel control



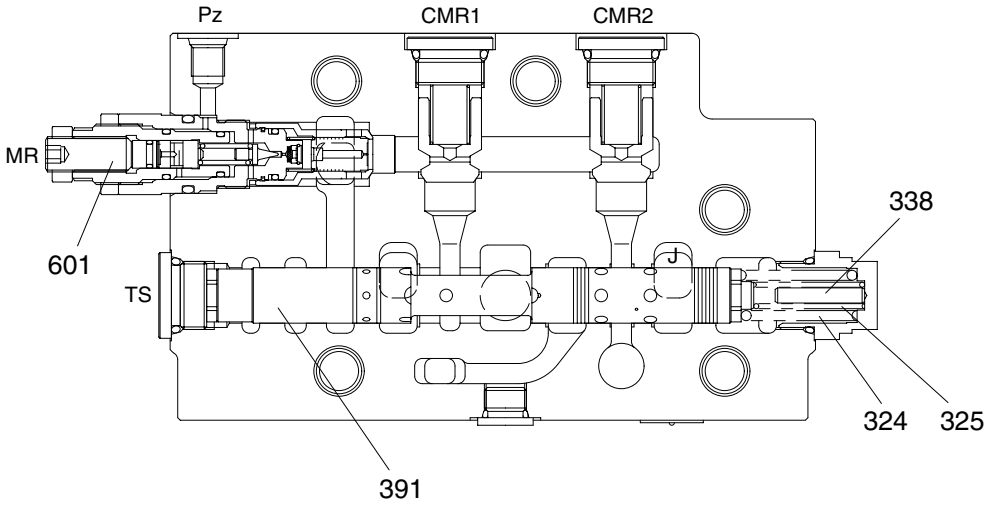
During the travel operation, the hydraulic fluid of the pump P1 is supplied to the travel motor and the hydraulic fluid of the pump P2 is supplied to the other travel motor.
The pilot pressure from the pilot control valve is supplied to the port XAtr and XAtl.
And it shifts the travel right spool(301) and travel left spool(310) in the left direction against springs (328) and (329). Hydraulic fluid from the pump P1 flows into the travel left spool(310) through the parallel path and hydraulic fluid from the pump P2 flows into the travel right spool(301). Then they are directed to the each travel motor through port Atl and Atr. As a result, the travel motors turn and the return flow from the travel motors enter port Btl and port Btr. The flow from the motors returns to the tank port R2 through the travel spools(310 and 301).
In the case of the opposite operation, the operation is similar.

4) CIRCUIT PRESSURE PROTENCTION

The control valve has two kinds of relief valve to limit the pressure in a circuit.

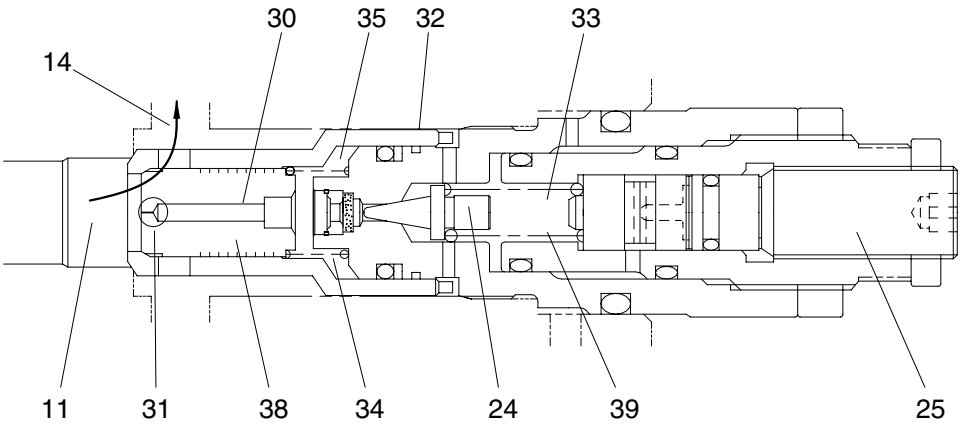
(1) Main relief valve

Limits the pressure of the main hydraulic system.



The hydraulic fluid from the pump P1 and the pump P2 enters the control valve through ports P1 and P2, respectively. From here the flow is directed to the main relief valve(601) through the check valve CMR1 or CMR2(511) and path(11). The pressure in path(11) is limited by the main relief valve(601) to its set pressure.

· Main relief operation while working

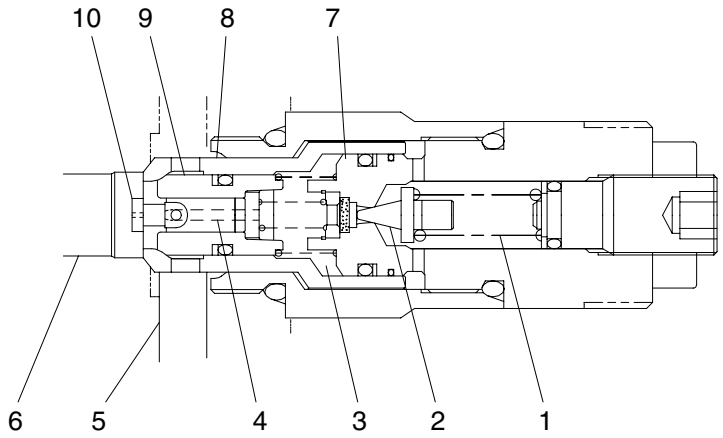


29072MC44

While the pressure in path(11) is lower than the set pressure of main relief valve(601), the poppet (24) is seated and the hydraulic fluid in path(11) can not escape to the return(14). When the pressure in path(11) approaches the pressure setting, poppet(24) opens against the spring(39). As the flow in chamber(33) escapes into the return(14) through path(32), its pressure decreases. At the same time, hydraulic fluid in path(11) flows into path(30) with a pressure drop across orifice(31). Then pressure in spring chamber(35) becomes lower because it bleeds off through path(30). The pressure from path(11) pushes the plunger(38) in the left direction against the spring(34). Then plunger(38) opens and hydraulic fluid in path(11) escapes into the return(14) and maintains the pressure setting. The pressure setting is adjusted with adjustment screw(25).

(2) Port relief valve

Limits the service pressure in a cylinder circuit.



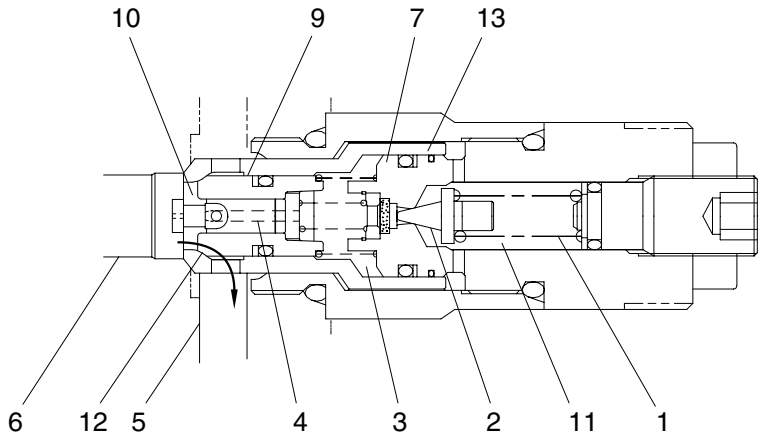
29072MC45

Port relief valves and make up valves are fitted between the cylinders of the working devices(Boom, arm, bucket) and their spools. In the case of an external force acting on the cylinder rod with its spool in neutral, the pressure in the cylinder could become excessive. The port relief valve(602) restricts this pressure to the set pressure of the valve.

Port relief valve(602) have also the additional function of a make up valve. It is possible, under the influence of an external force acting on a cylinder that a condition can occur where insufficient flow is available to match cylinder velocity. If this occurs then a vacuum and thereby cavitation could exist. To eliminate such an occurrence, a make up valve operates to break this vacuum by supplying the return flow into the cylinder.

The hydraulic fluid between the cylinder and its spool flows into the path(6) to pressurize the port relief valve(602). The hydraulic fluid in the path(6) flows into the spring chamber(3) through the path(4) in the piston(10). If the pressure is lower than the pressure setting, the poppet(2) is shut off because the force of the spring(1) overcomes the pressure. So the path(6) and the spring chamber(3) have the same pressure. Because the spring chamber(3) side pressured area of the seat(8) and the plunger(9) is larger than that of the path(6) side, seat(8) and the plunger(9) are pushed in the right direction to be seated securely and then the hydraulic fluid in the path(6) doesn't escape into the return path(5).

① **Port relief function**

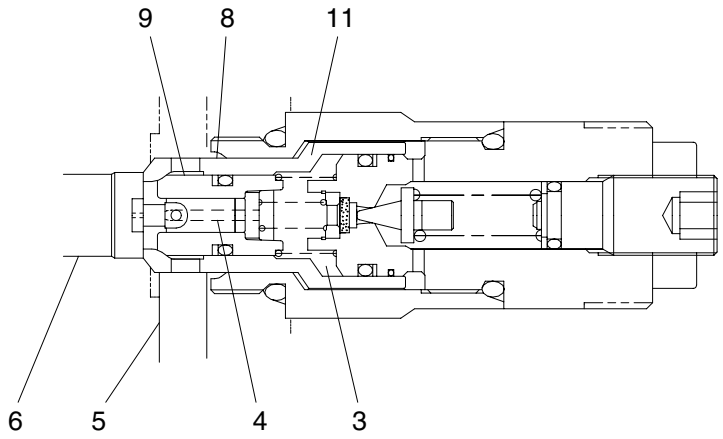


29072MC46

When the pressure in the path(6) is pressurized to the pressure setting, the poppet(2) is pushed open against the spring(1). The hydraulic fluid in the chamber(11) flows into the return path(5) through the path(13) with reducing its pressure. The piston(10) is shifted in the left direction by the pressure in the path(6) and stops on the end of the plug(7).

The hydraulic fluid in the path(6) flows into the chamber(11) through the path(4) in the piston(10) and the spring chamber(3). Because the differential pressure occurs between the pass(6) and the pass(4) by the orifice between the outernal diameter of the end of the piston(10) and the internal diameter of the plunger(9), the pressure in the spring chamber(3) becomes low and therefore the plunger(9) is pushed in the left direction with the path(12) opened so that the hydraulic fluid in the path(6) flows into the return path(5).

② **Make up function**



29072MC47

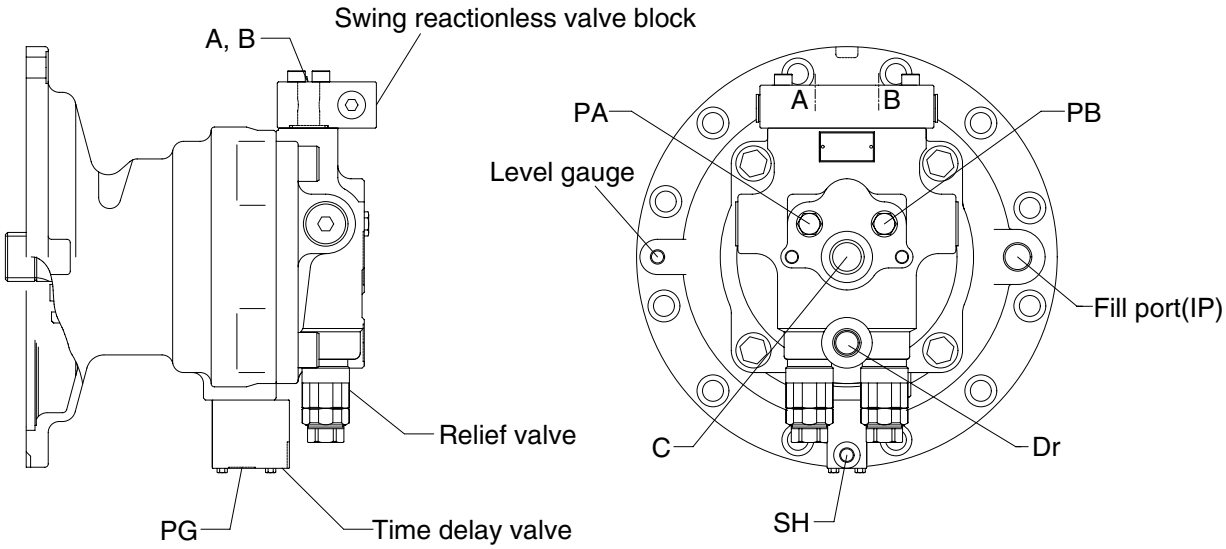
Following this then the case of a port relief valve operating as a make up valve is now explained. In the case that the hydraulic fluid in the cylinder rod(Head) side escapes from the port relief valve (602), then hydraulic fluid needs to be supplied because vacuum occurs in the head(Rod) side. When vacuum occurs in the side of the path(6), it also occurs in the spring chamber(3) through the path(4). The pressure in the side of the return path(5) acts on the seat(8). The seat(8) is shifted in the left direction by the return pressure because the spring chamber(3) sides of the seat(8) and the plunger(9) are under a vacuum. The hydraulic fluid in the return path(5) flows into the path(6) so as to break the vacuum in the path(6) side.

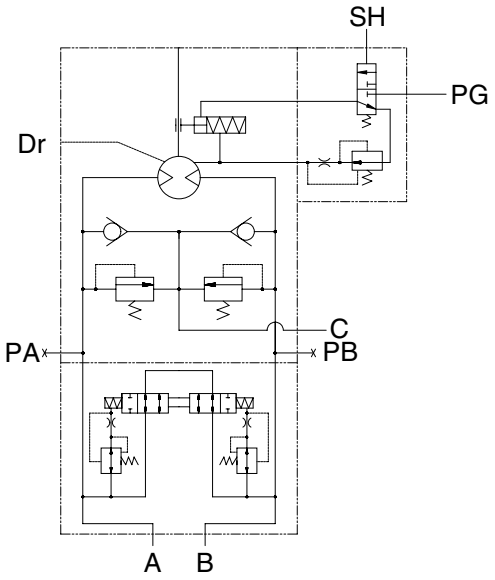
GROUP 3 SWING DEVICE

1. STRUCTURE

Swing device consists swing motor, swing reduction gear.

Swing motor include mechanical parking valve, relief valve, make up valve and time delay valve.

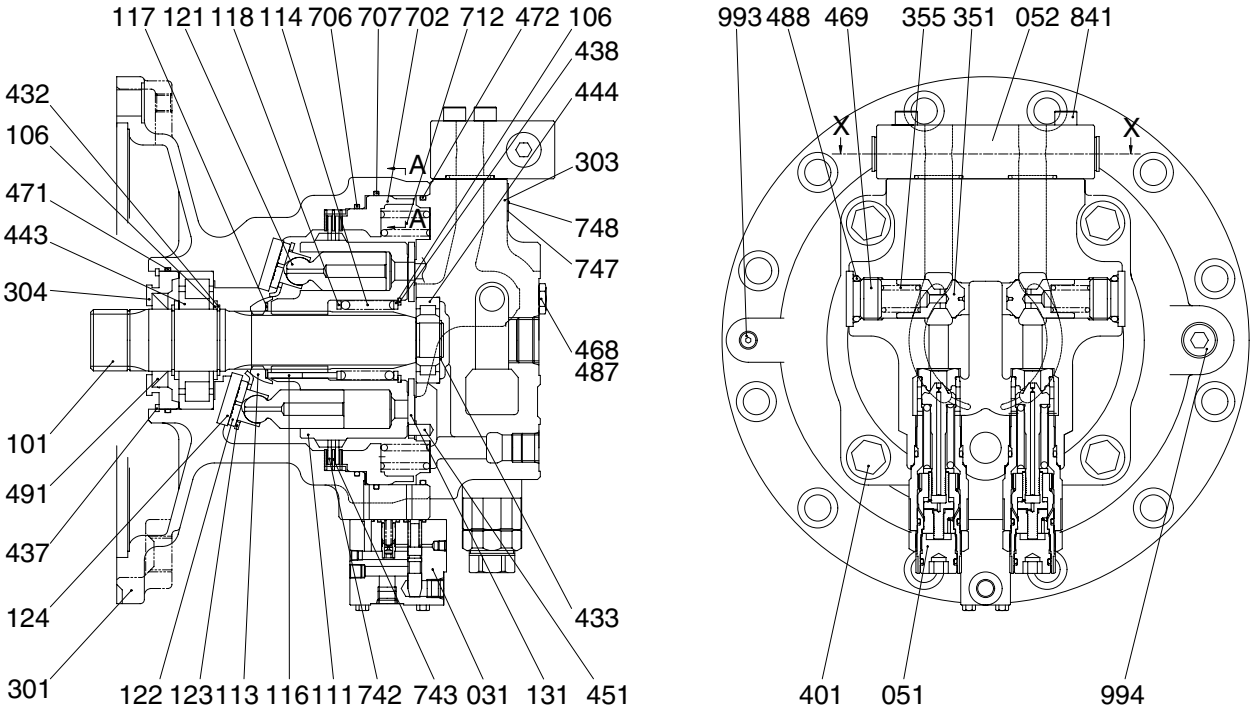




R32072SM04

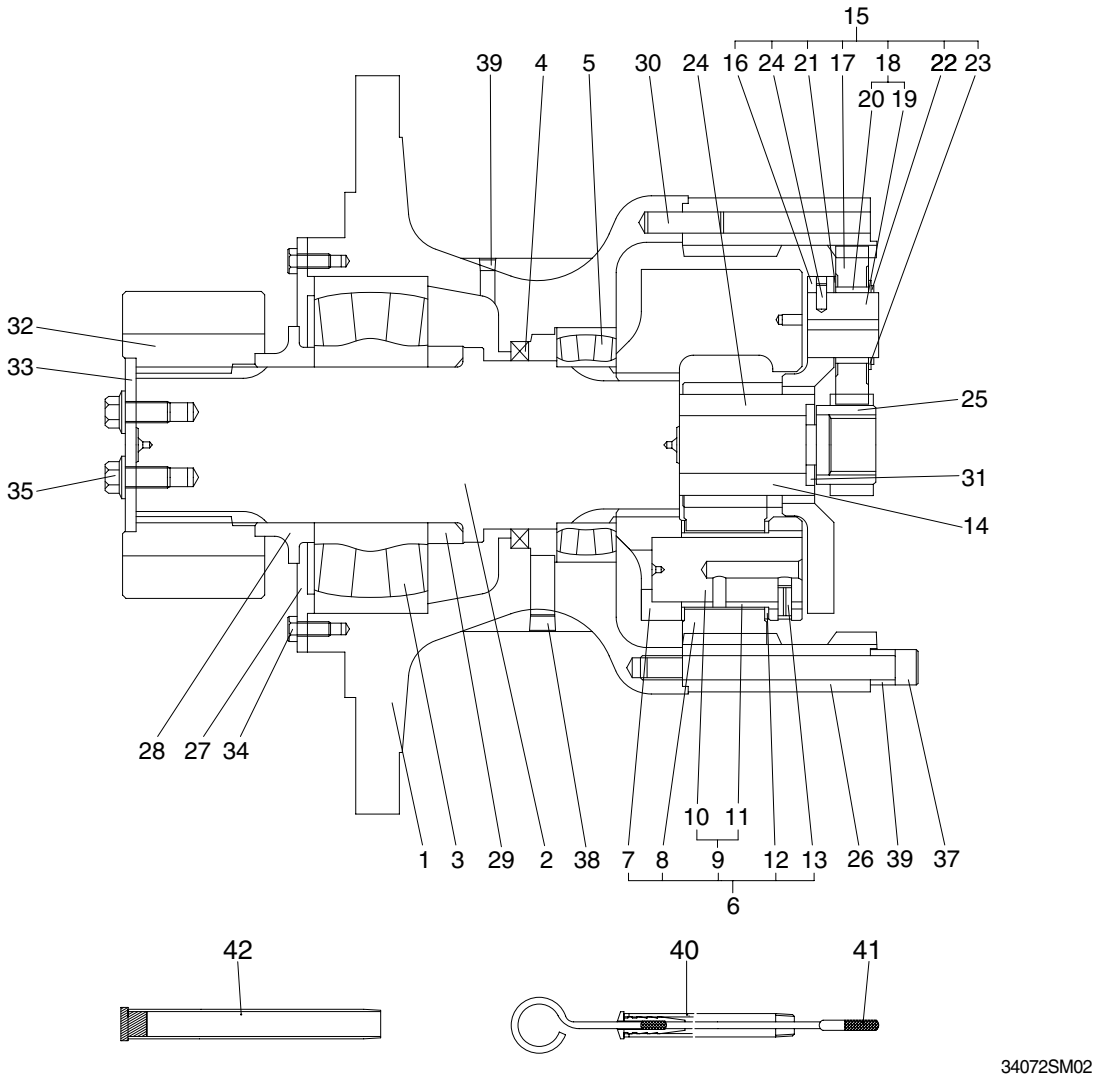
Port	Port name	Port size
A, B	Main port	ø 20
Dr	Drain port	PF 1/2-19
C	Make up port	PF 1-24
PA, PB	Gauge port	PF 1/4-12
PG	Brake release port	PF 1/4-12
SH	Brake pilot port	PF 1/4-12
IP	Gear oil inlet port	PT 3/4-19

1) SWING MOTOR



031	Time delay valve	162	O-ring	468	Plug
051	Relief valve	163	O-ring	469	Plug
052	Valve assy	171	Hexagon screw	471	O-ring
101	Drive shaft	301	Casing	472	O-ring
106	Spacer	303	Casing	485	O-ring
111	Cylinder	304	Front cover	487	O-ring
113	Spherical bush	351	Plunger	488	O-ring
114	Spring	355	Spring	491	Oil seal
116	Push rod	401	Socket bolt	702	Piston
117	Spacer	432	Snap ring	706	O-ring
118	Spacer	433	Snap ring	707	O-ring
121	Piston	437	Snap ring	712	Brake spring
122	Shoe	438	Snap ring	742	Friction plate
123	Retainer	443	Roller bearing	743	Separate plate
124	Shoe plate	444	Needle bearing	841	Socket bolt
131	Valve plate	451	Spring pin	993	Plug
161	O-ring	464	Plug	994	Plug

2) REDUCTION GEAR



1	Casing	15	Carrier assy 1	29	Spacer ring
2	Drive shaft	16	Carrier 1	30	Knock pin
3	Roller bearing	17	Planet gear 1	31	Thrust plate 3
4	Oil seal	18	Pin assy 1	32	Pinion gear
5	Roller bearing	19	Pin 1	33	Lock plate
6	Carrier assy 2	20	Bushing 1	34	Hexagon bolt
7	Carrier 2	21	Side plate 1	35	Hexagon flange bolt
8	Planet gear 2	22	Side plate 2	37	Socket bolt
9	Pin assy 2	23	Stop ring	38	Plug
10	Pin 2	24	Spring pin	39	Plug
11	Bushing 2	25	Sun gear 1	40	Gage pipe
12	Thrust washer	26	Ring gear	41	Gage bar
13	Spring pin	27	Cover plate	42	Air breather assy
14	Sun gear 2	28	Spacer	43	Bushing 1

2. FUNCTION

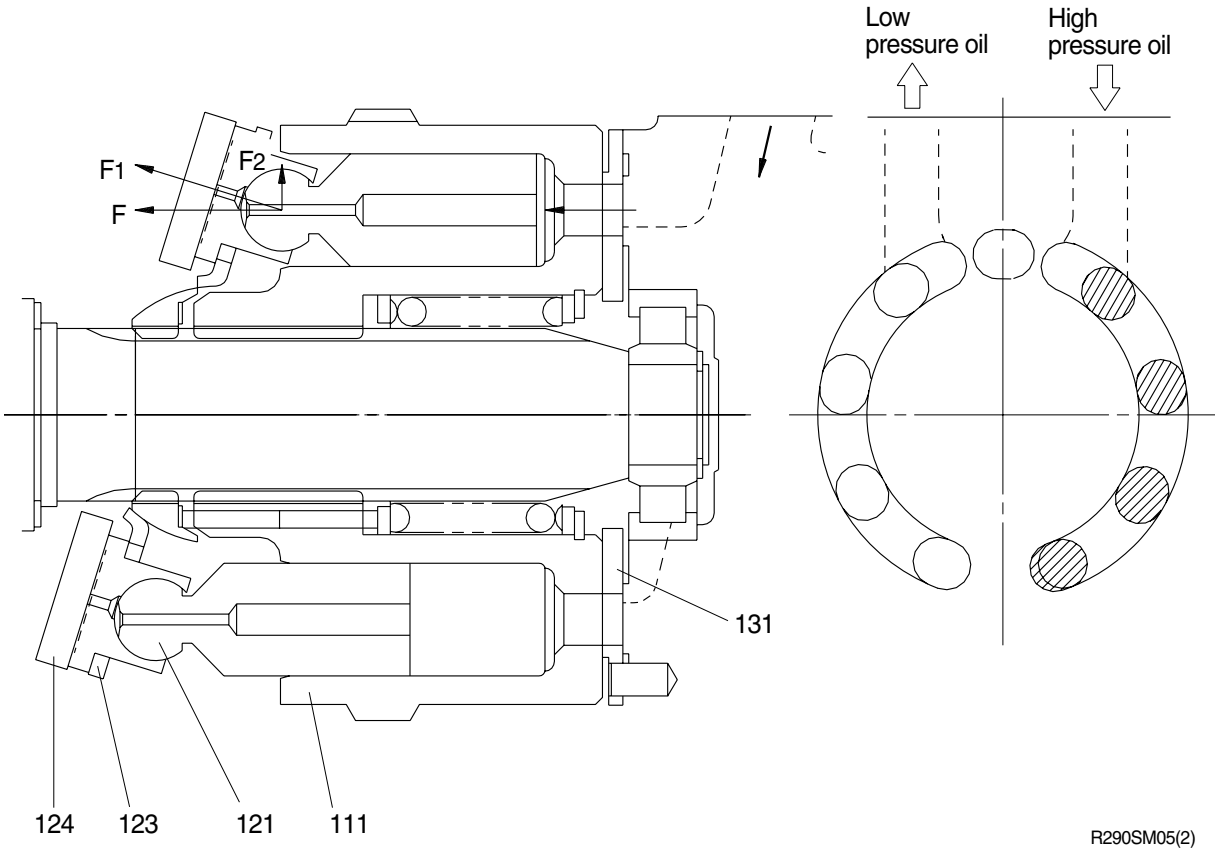
1) ROTARY PART

When high pressurized oil enters a cylinder through port(a), which is the inlet of balance plate(131), hydraulic pressure acting on the piston causes axial force F. The pressure force F works via the piston(121) upon the return plate(123) which acts upon the swash plate(124) via an hydrostatic bearing. Force F1 perpendicular to swash plate(124) and force F2 perpendicular to cylinder center. Being transferred to the cylinder block(111) through piston, force F2 causes rotational moment at surroundings of cylinder.

Since cylinder block has 9 equidistantly arrayed pistons, rotational torque is transmitted to cylinder shaft in order by several pistons connected to the inlet port of high pressurized oil. When the direction of oil flow is reversed, rotational direction of cylinder is also reversed. Output torque is given by the equation.

$$T = \frac{p \times q}{2\pi}, \quad q = Z \cdot A \cdot PCD \cdot \tan\theta, \quad F1 = \frac{F}{\cos\theta}, \quad F_2 = F\tan\theta, \quad S = PCD \times \tan\theta$$

Where p : Effective difference of pressure(kgf/cm^2)
q : Displacement(cc/rev)
T : Output torque(kgf · cm)
Z : Piston number(9EA)
A : Piston area(cm^2)
θ : Tilting angle of swash plate(degree)
S : Piston stroke(cm)



R290SM05(2)

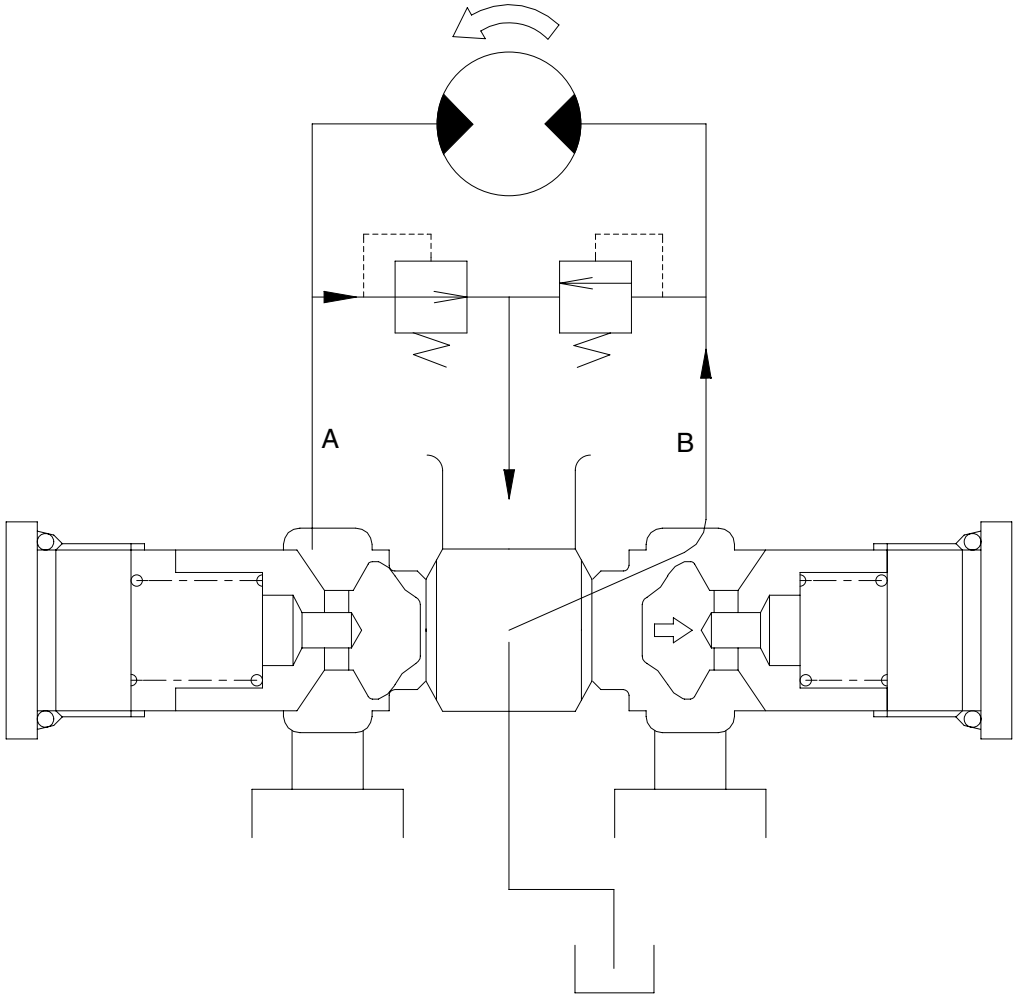
2) MAKE UP VALVE

In the system using this type of motor, there is no counter balance functioning valve and there happens the case of revolution exceeding hydraulic supply of motor. To prevent the cavitation caused by insufficient oil flow there is a make up valve to fill up the oil insufficiency.

A make up valve is provided immediately before the port leading to the hydraulic oil tank to secure feed pressure required when the hydraulic motor makes a pumping action. The boost pressure acts on the hydraulic motor's feed port via the make up valve.

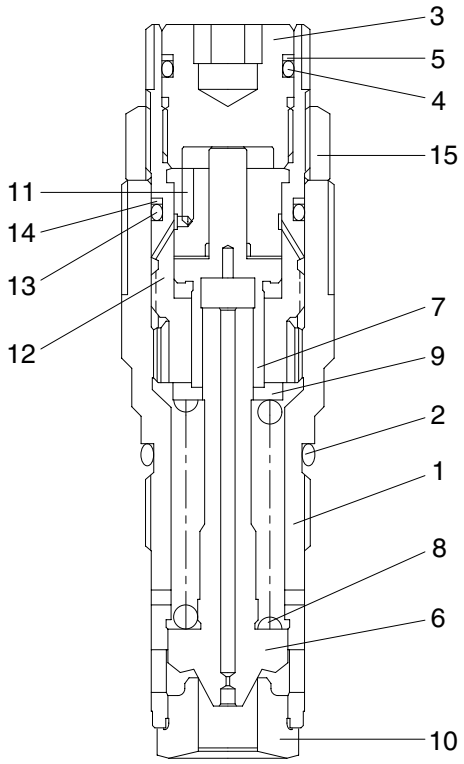
Pressurized oil into the port B, the motor rotate counterclockwise.

If the plunger of MCV moves neutral position, the oil in the motor is drain via left relief valve, the drain oil run into motor via right make up valve, which prevent the cavitation of motor.



29072SM09

3) RELIEF VALVE



1	Body
2	O-ring
3	Plug
4	O-ring
5	Back up ring
6	Plunger
7	Piston
8	Spring
9	Seat spring
10	Seat
11	Sleeve
12	Adjust plug
13	O-ring
14	Back up ring
15	Nut

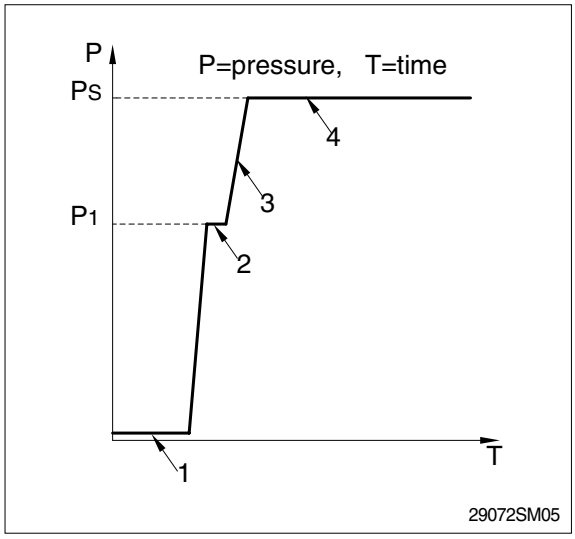
29072SM03

(1) Construction of relief valve

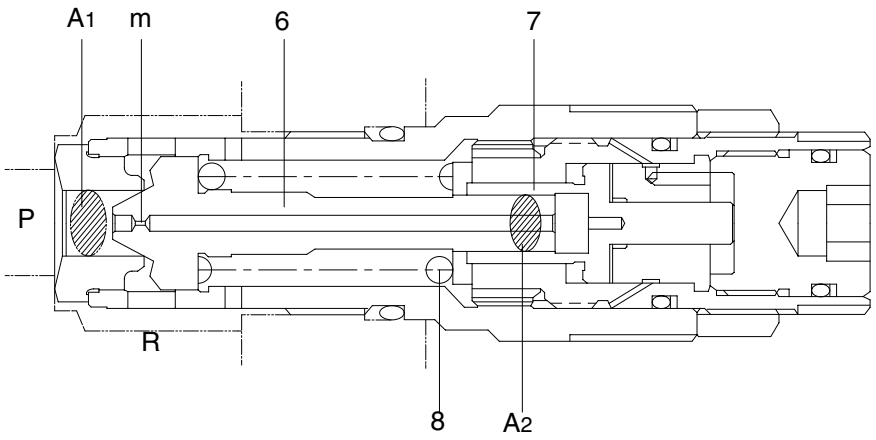
The valve casing contains two cartridge type relief valves that stop the regular and reverse rotations of the hydraulic motor. The relief valves relieve high pressure at start or at stop of swing motion and can control the relief pressure in two steps, high and low, in order to insure smooth operation.

(2) Function of relief valve

Figure illustrates how the pressure acting on the relief valve is related to its rising process. Here is given the function, referring to the figure following page.



① Ports (P,R) at tank pressure.

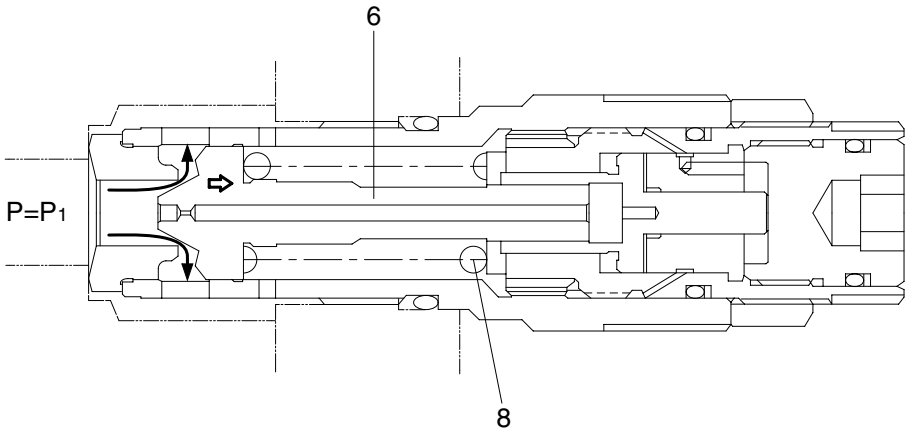


29072SM04

② When hydraulic oil pressure($P \times A_1$) reaches the preset force(F_{SP}) of spring(8), the plunger (6) moves to the right as shown.

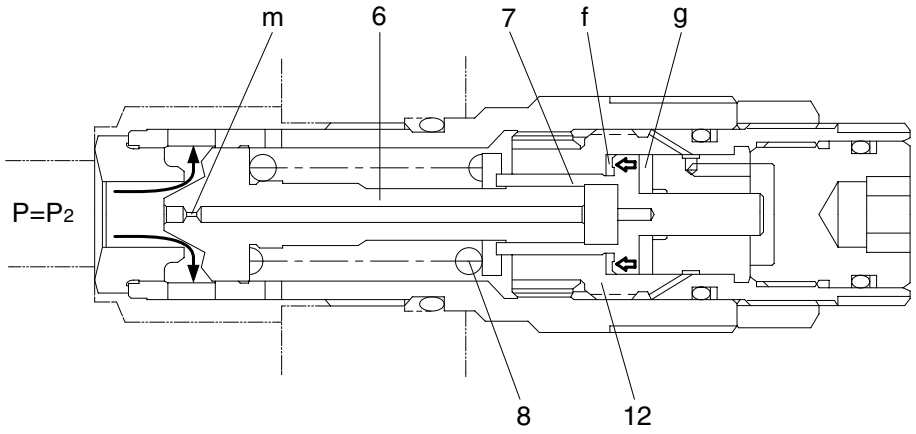
$P_1 \times A_1 = Fsp + P_1 \times A_2$

$$P_1 = \frac{Fsp}{A_1 - A_2}$$



29072SM06

③ When the pressure of chamber g reaches the preset force(Fsp) of spring(8), the piston(7) moves right and stop the piston(7) hits the end of body.

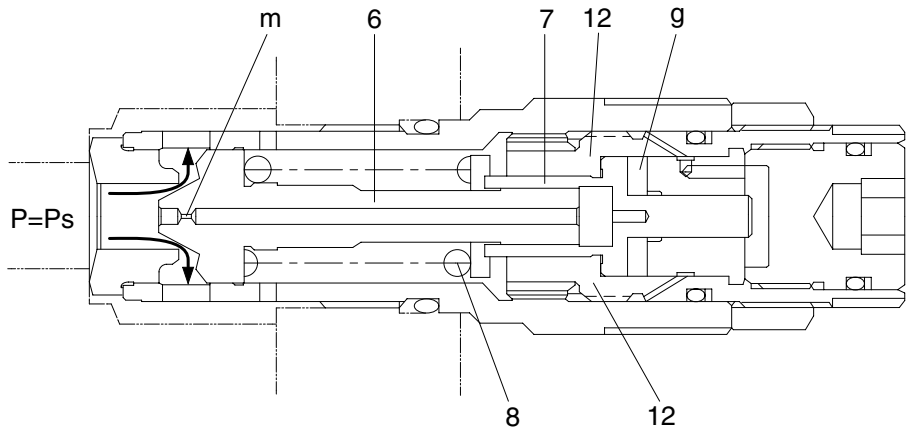


29072SM07

④ When piston(7) hits the end of body, it stops moving to the right any further. As the result, the pressure in chamber(g) equals(Ps).

$$Ps \times A_1 = Fsp + Ps \times A_3$$

$$Ps = \frac{Fsp}{A_1 - A_3}$$



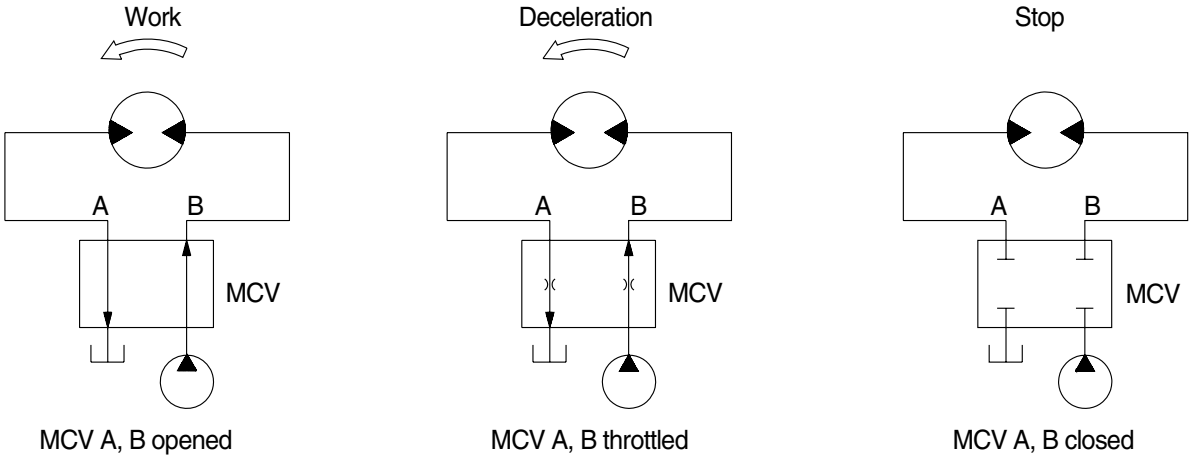
29072SM08

4) BRAKE SYSTEM

(1) Control valve swing brake system

This is the brake system to stop the swing motion of the excavator during operation.

In this system, the hydraulic circuit is throttled by the swing control valve, and the resistance created by this throttling works as a brake force to slow down the swing motion.



MCV A, B opened　　MCV A, B throttled　　MCV A, B closed

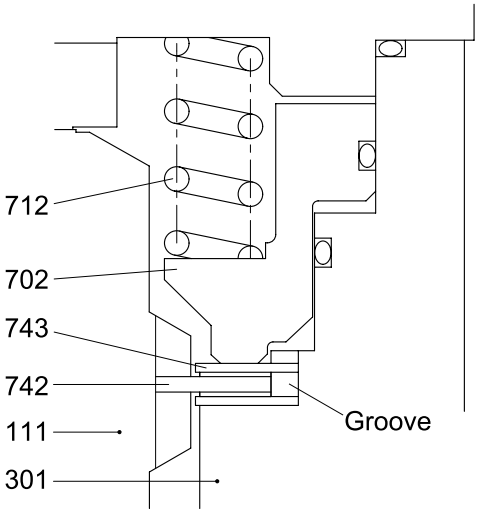
(2) Mechanical swing parking brake system

The mechanical swing parking brake system is installed to prevent the upper structure from swinging downhill because of its own weight when the excavator is parked on a slope since it completely eliminates the hydraulic drift of swing motion while the excavator is on a slope, work can be done more easily and safely.

① Brake assembly

Circumferential rotation of separate plate(743) is constrained by the groove located at casing(301). When housing is pressed down by brake spring(712) through friction plate(742), separate plate(743) and brake piston(702), friction force occurs there.

Cylinder(111) is constrained by this friction force and brake acts, while brake releases when hydraulic force exceeds spring force.

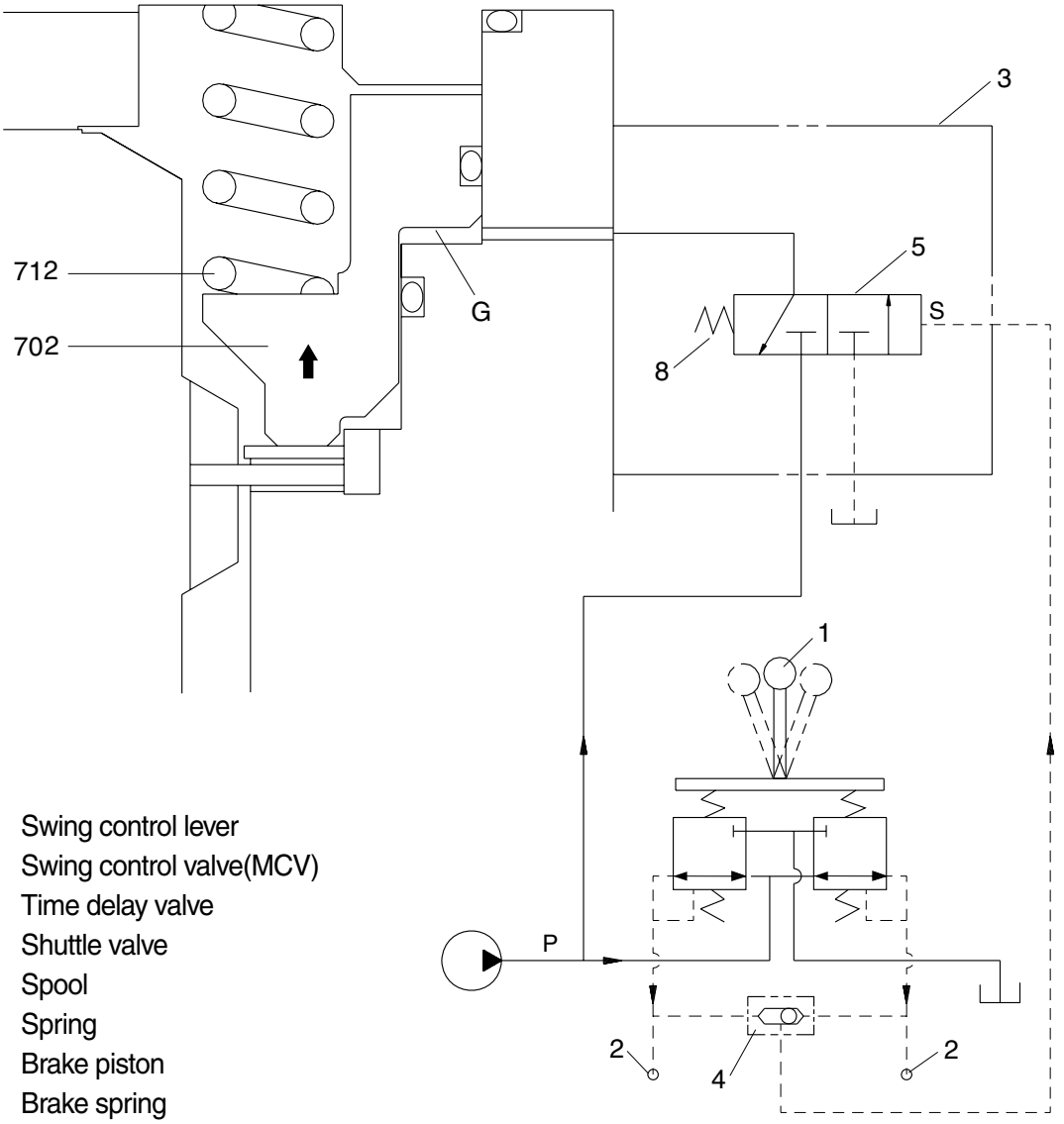


111	Cylinder	712	Brake spring
301	Casing	742	Friction plate
702	Brake piston	743	Separate plate

② **Operating principle**

a. When the swing control lever(1) is set to the swing position, the pilot oil go to the swing control valve(2) and to SH of the time delay valve(3) via the shuttle valve(4), this pressure move spool(5) to the leftward against the force of the spring(8), so pilot pump charged oil(P3) goes to the chamber G.

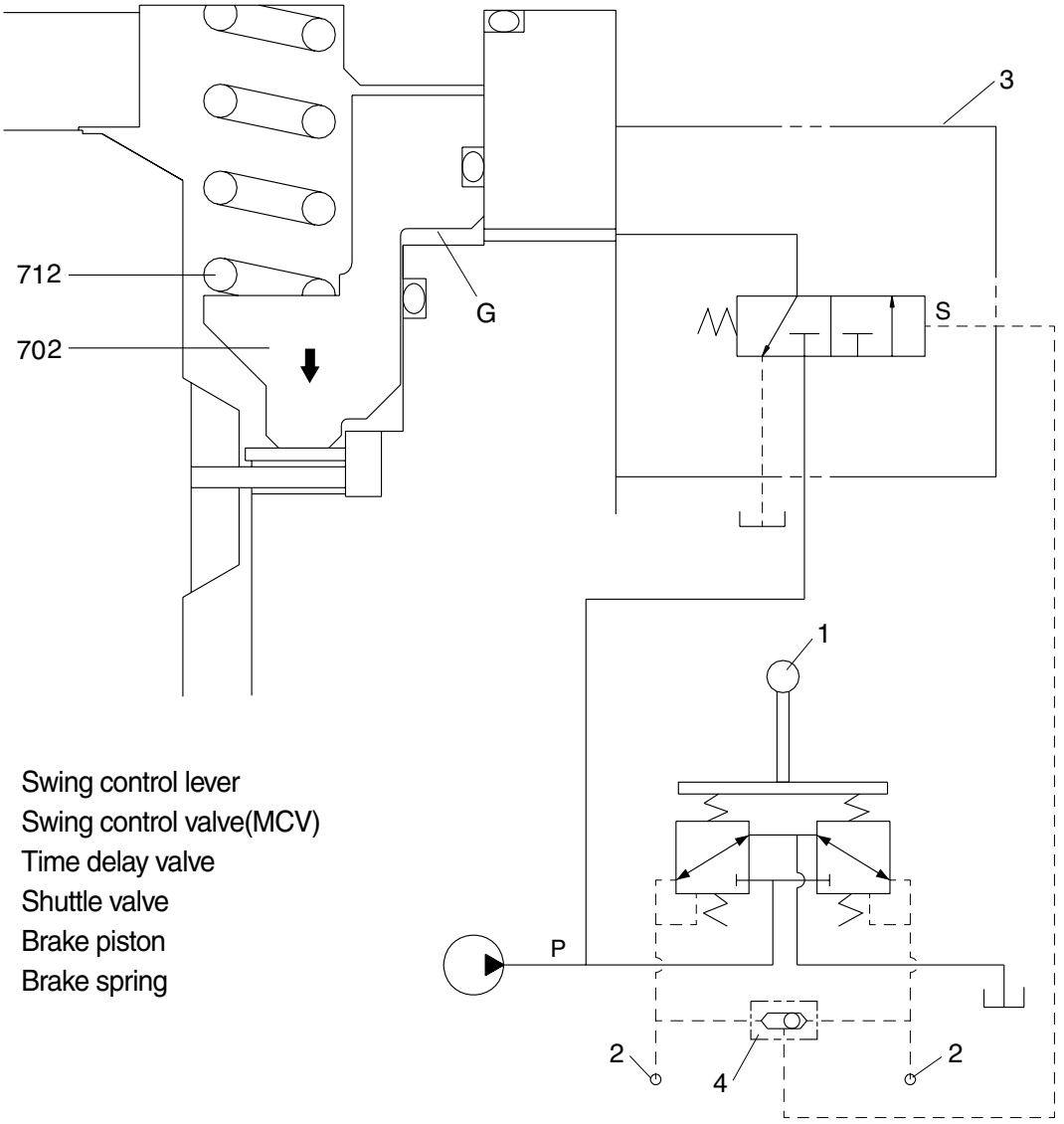
This pressure is applied to move the piston(702) to the upward against the force of the spring(712). Thus, it releases the brake force.



1 Swing control lever
2 Swing control valve(MCV)
3 Time delay valve
4 Shuttle valve
5 Spool
8 Spring
702 Brake piston
712 Brake spring

R290SM07(1)

b. When the swing control lever(1) is set the neutral position, the time delay valve(3) shifts the neutral position and the pilot oil blocked chamber G.
Then, the piston(702) is moved lower by spring(712) force and the return oil from the chamber G is drain.



1 Swing control lever
2 Swing control valve(MCV)
3 Time delay valve
4 Shuttle valve
702 Brake piston
712 Brake spring

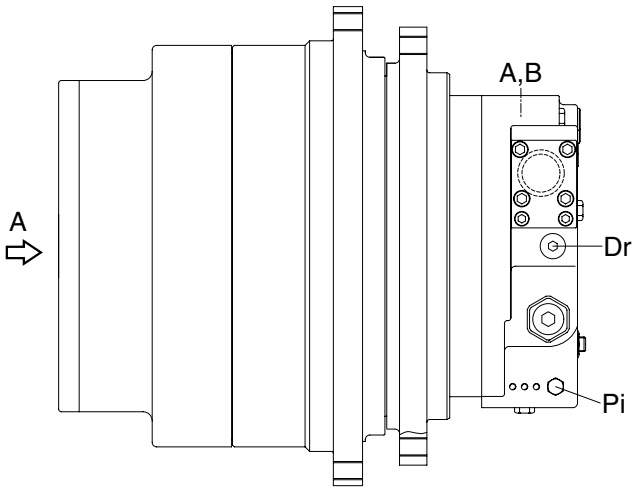
R290SM08(1)

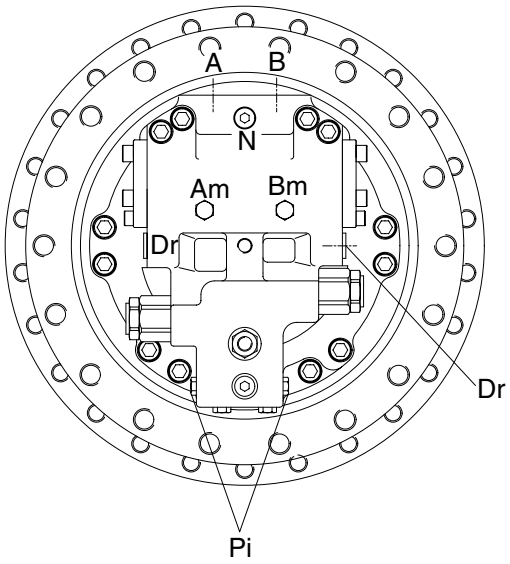
GROUP 4 TRAVEL DEVICE

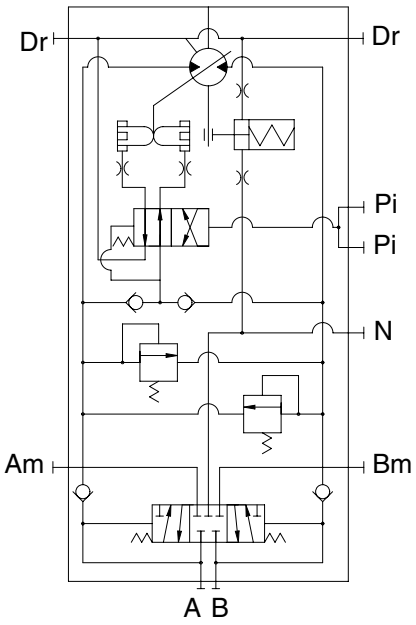
1. CONSTRUCTION

Travel device consists travel motor and gear box.

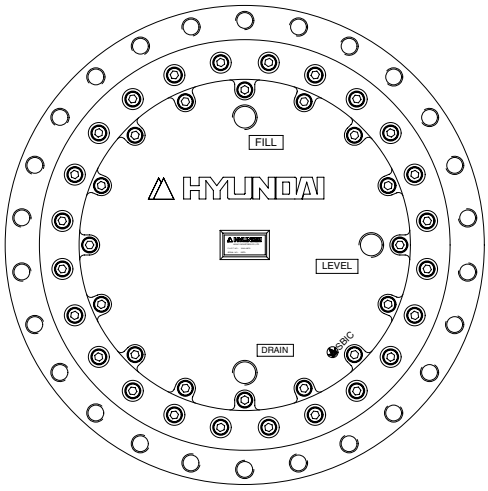
Travel motor includes brake valve, parking brake and high/low speed changeover mechanism.







CIRCUIT DIAGRAM



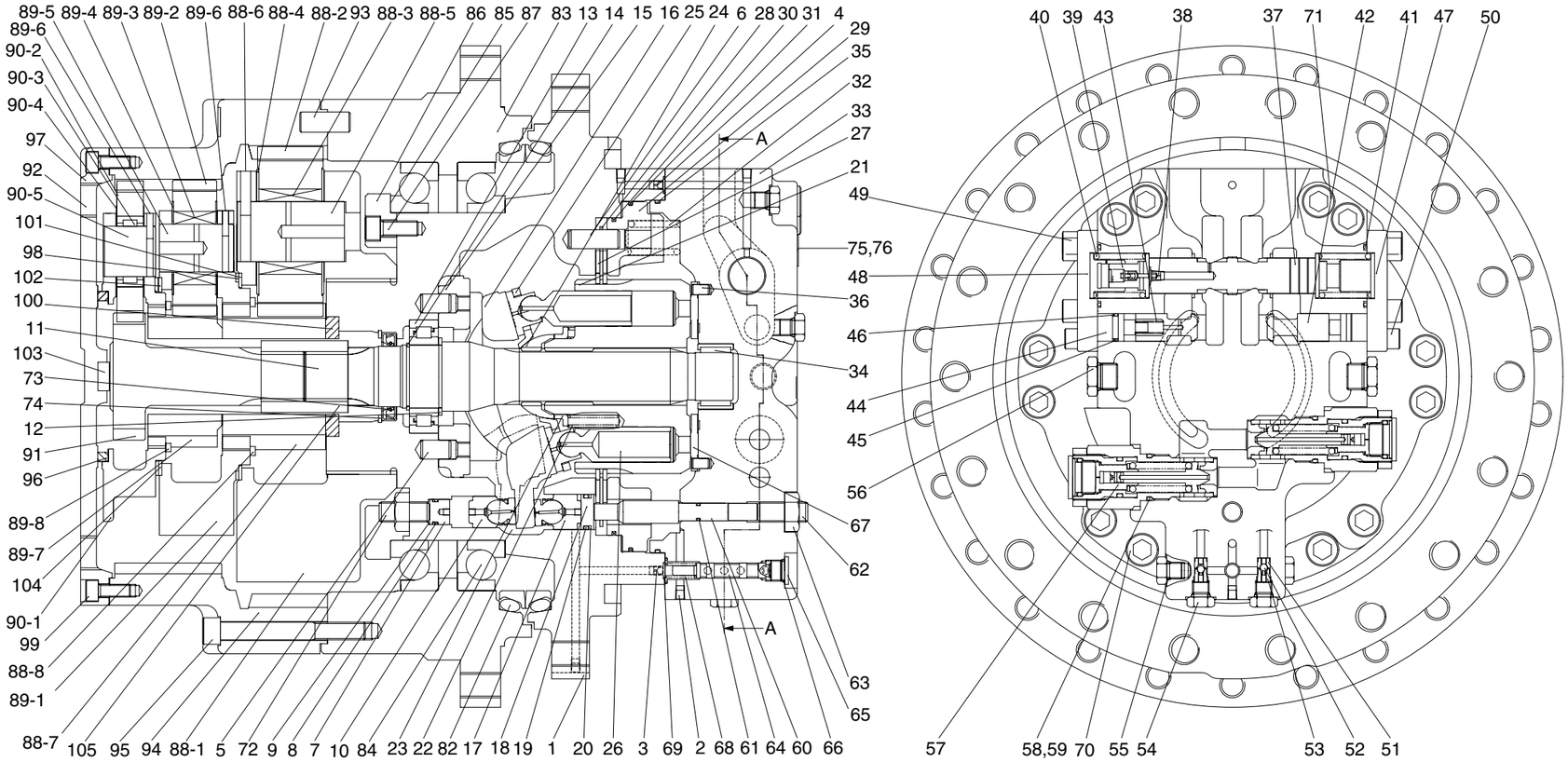
VIEW A

300072TM01A

Port	Port name	Port size
A, B	Valve port	SAE 6000psi 1"
Pi	Pilot port	PF 1/4
Dr	Drain port	PF 1/2
Am, Bm	Gage port	PF 1/4
N	Parking release port	PF 1/4

2. SPECIFICATION

1) TRAVEL MOTOR



SECTION A-A

2907A2TM01

1	Casing	16	Plate	31	Ring	46	Back up ring	61	O-ring	83	Housing	89-1	Carrier No.2	92	Plug
2	Plug	17	Piston	32	Spring	47	Cap	62	Lock screw	84	Bearing	89-2	Planetary gear No.2	93	Lock pin
3	Screw	18	Stopper	33	Valve casing	48	Cap	63	Nut	85	Shim	89-3	Needle No.2	94	Ring gear
4	Screw	19	O-ring	34	Needle bearing	49	Bolt	64	Spool	86	Retainer	89-4	Thrust washer No.2	95	Bolt
5	Pin	20	Back up ring	35	O-ring	50	Socket bolt	65	Plug	87	Bolt	89-5	Pin No.2	96	Thrust ring No.1
6	Pin	21	Cylinder block	36	Pin	51	Seat	66	O-ring	88	Carrier No.3	89-6	Spring pin No.2	97	Cover
7	Stopper	22	Cylinder spring	37	Spool	52	Steel ball	67	Valve plate	88-1	Carrier No.3	89-7	Sun gear No.2	98	Thrust ring No.2
8	O-ring	23	Spacer	38	Screw	53	Stopper	68	Spring	88-2	Planetary gear No.3	89-8	Snap ring No.2	99	Bolt
9	Back up ring	24	Guide	39	Damping check	54	Plug	69	O-ring	88-3	Needle No.3	90	Carrier No.1	100	Motor ring
10	Piston	25	Plate	40	Spring	55	O-ring	70	Socket bolt	88-4	Thrust washer No.3	90-1	Carrier No.1	101	Thrust ring No.3
11	Shaft	26	Piston & Shoe assy	41	O-ring	56	Plug	71	Socket bolt	88-5	Pin No.3	90-2	Planetary gear No.1	102	Thrust ring No.1
12	Spacer	27	Plate	42	Plunger	57	Relief valve	72	Lock screw	88-6	Spring pin No.3	90-3	Needle bearing No.1	103	Pad
13	Roller bearing	28	Plate	43	Spring	58	O-ring	73	Oil seal	88-7	Sun gear No.3	90-4	Thrust washer No.1	104	Thrust ring No.2
14	Stop ring	29	Brake	44	Stopper	59	Back up ring	74	Lock ring	88-8	Snap ring No.3	90-5	Pin No.1	105	Coupling
15	Support	30	Ring	45	O-ring	60	Rod	82	Floating Seal	89	Carrier No.2	91	Sun gear No.1		

3. PRINCIPLE OF DRIVING

1) WORKING OF ROTARY GROUP

The high pressurized hydraulic oil which is supplied from a hydraulic pump is flows into a cylinder(21) through the valve casing(33) of motor, and valve plate(67).

The rotary group has a construction that the above high pressurized hydraulic oil is flow only one side of the line Y-Y which connect the upper and lower dead point of the piston(26).

This high pressurized hydraulic oil works on the piston and generating the force F1, F1 = P * A(P : supplied pressure, A : pressure receving area), like following pictures.

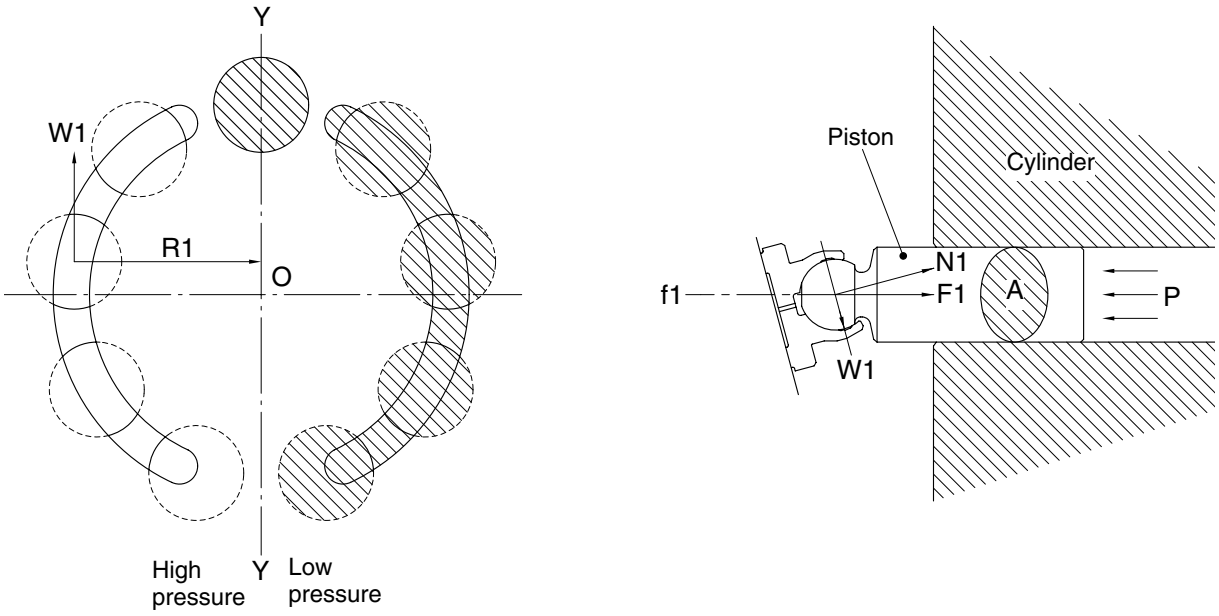
This force, F1, is devided by the swash plate(16) having a tilting angle α into the thrust component N1 and radial component W1.

The W1 generates torque, T = W1 * R1, in respect to the line Y-Y.

This torque generated by each piston on the high pressurized hydraulic oil side is summed up onto a resultant torque Σ(W1 * R1), which prodeces torque for rotation.

This torque transfers the rotation force to the cylinder(21) through the pistons.

Since the cylinder block is spline-coupled with the shaft, the rotation force is transmitted to the shaft accordingly.



2907A2TM07

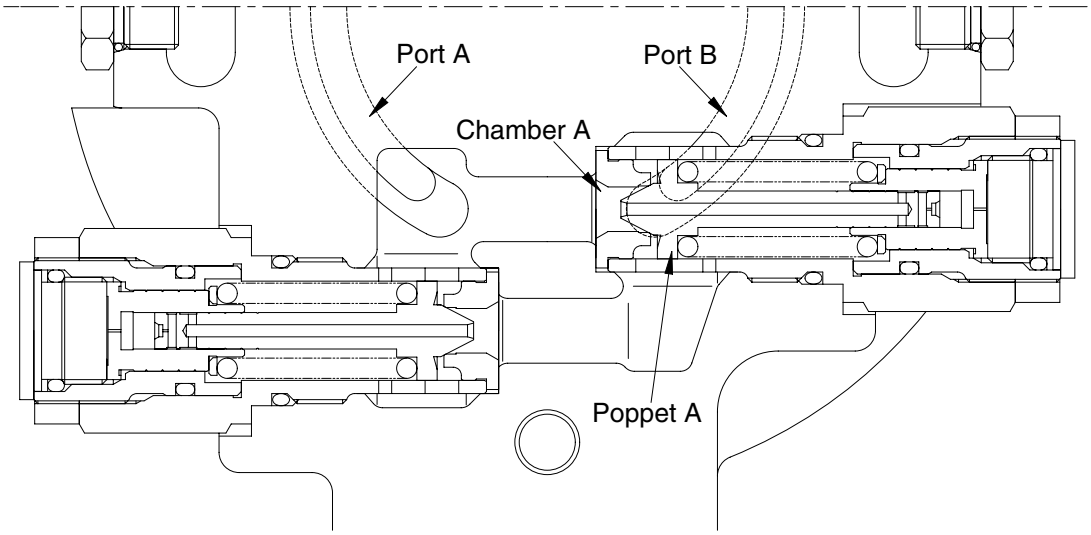
2) WORKING OF RELIEF VALVE

Relief valve carries on two function of following.

(1) Relief valve is to keep the starting pressure of the hydraulic motor at a constant value and bypass to the return line excessive oil generated at the motor inlet depending upon the acceleration speed of the inertia object.

(2) In case of an inertia object stopped, relief valve is generating a break pressure at the outlet and stop it forcedly.

The chamber A is always connect with port A of a motor. When the pressure at port A increase and the force pushing poppet A is higher than the pressure of the spring, then poppet A is pushed up from the contact surface of seat A, and oil flows from chamber A to port B.



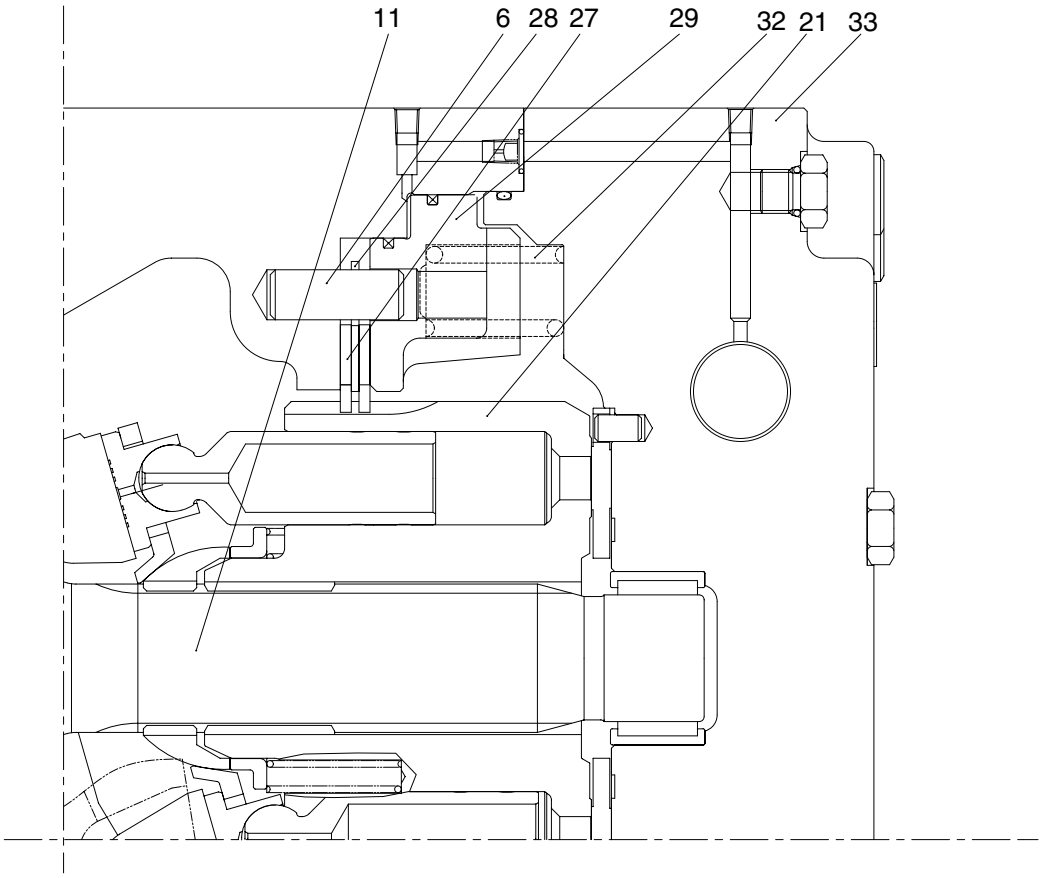
2907A2TM04

3) WORKING OF NEGATIVE BRAKE

The negative brake is released applying to the brake piston(29) the pressure led through built in the valve casing(33) spool. With no pressure working, the brake force is always ensured.

The brake force is generated by the frictional force among a plate(28) fixed by pin(6) and shaft casing, brake piston(29) and a frictional plate(27) connected through spline outside the cylinder block(21).

Without pressure being applied to the brake piston, the brake piston is pushed by ten brake springs(32) and the friction plate and separator plate are held between the brake piston and casing. This friction force restrains the shaft(11) spline-coupled with the cylinder block, and thus functions the brake.



300075TM05

4) COUNTERBALANCE VALVE

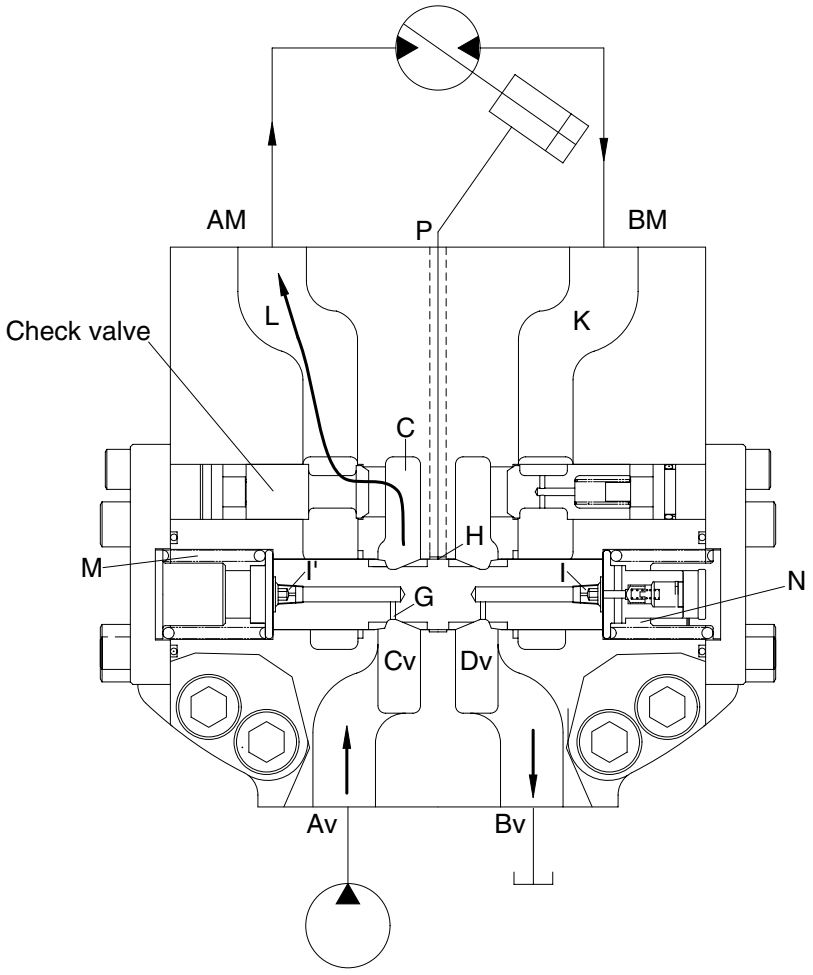
Av port is connected to a hydraulic pump : Bv port is connected to a tank.

The oil supplied from the hydraulic pump passed through Av → Cv → C sequence, pushed up the poppet of the check valve, passed through L to port AM, and is supplied to the hydraulic motor to turn it. But the brake is operated. Therefore, the pump discharge oil pressure is increases. And the pressure is led via passage G to spring room M. When the pressure in room M exceed the value equivalent to the force of the spring which holds the spool at its neutral position, the spool begins to move right.

The oil in room N is sent to room Dv by orifice I and discharged from Bv port to a tank. So spool moves to the right. The oil flows as the way of K → Dv → Bv sequence. Also according to the oil path as composed way Cv → H → P sequence, the pressure of Av pump is provided to the port P. An working oil in room N is discharged through orifice and a gap. Therefore the switching operation of spool is driving slowly.

When the pump discharge pressure fall, spool moves to the left side by a spring at the side of room N. Also spool moves to the left, the hydraulic oil in room M is sent to Cv room through orifice I' and discharged to the Av port.

When the pressure at port Av fall down to the tank pressure, the pressure of room M is as the same as that the tank pressure and becomes equal to that in room N, and so the spool returns to its neutral position.



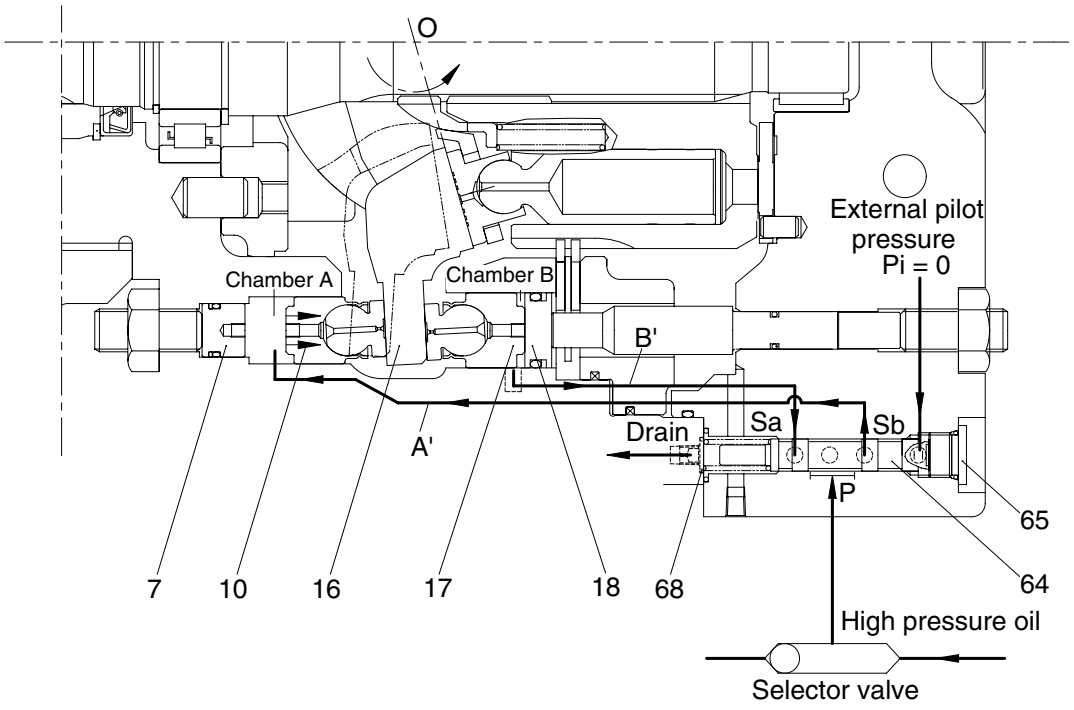
300072TM06

5) WORKING OF DISPLACEMENT CHANGEOVER

The capacity of the travel motor is changed by changing the tilting angle of this swash plate(16).
The tilting angle changes by displacement changeover valve.

(1) External pilot pressure : Pi = 0(Large displacement)

By means of the built-in high pressure selector mechanism in the valve casing(33), the high pressure oil working on the motor function to port P of the displacement-changeover valve.
A the spool(64) assembled in the displacement changeover valve is pressed to plug(65) by the spring(68), the high pressure oil at port P flows to port Sb.
This high pressure oil flows through oil passage(passage A') of valve casing(33) and shaft casing works to chamber A.
This oil in chamber B flows through passage B' and port Sa into the drain line. The displacement changeover piston(17) is pushed right and the swash plate(16) moves in the arrowed direction around rotation center 'O'. The swash plate moves until it touched stopper(18), and then is fixed there.



2907A2TM05

(2) External pilot pressure : Pi $\geq$ 20kgf/cm^2(small displacement)

If the force operating on spool(64) of the displacement changeover valve is stronger than the spring(68), and the spool moves to the left side.

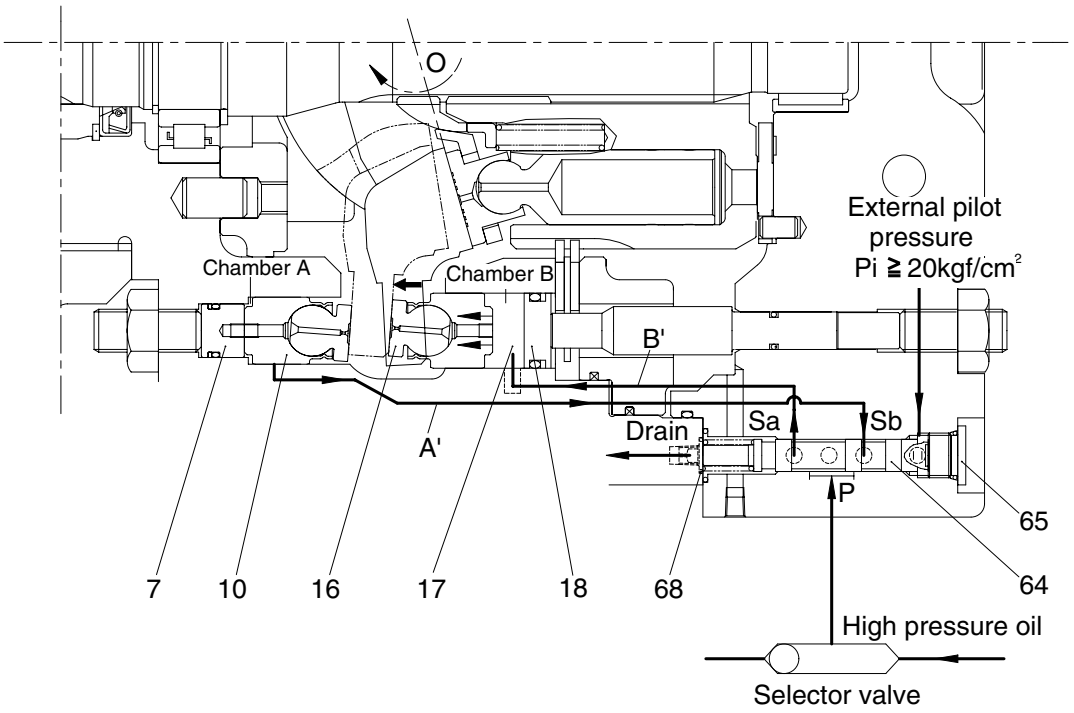
The high pressure oil is works on room B through passage Sa → B' from port P.

The oil in chamber A flows into the drain line through the passage A' → Sb.

The displacement changeover piston(17) is pushed left and the swash plate(16) moves in the arrowed direction around rotation center 'O'. The swash plate moves until it touches stopper(7), and then is fixed there.

If the load increase while the motor is working with its small displacement(Pi $\geq$ 20kgf/cm^2, 2nd speed) until the motor inlet port pressure reaches the preset value, the motor increase its displacement in response to the load, while maintaining the pressure at the preset value(automatic 2 -speed function). As motor inlet port pressure reaches the preset value and then spool(64) moves right side, inlet pressure oil flows into chamber A through port Sb and the swash plate moves until it touches stopper(17). If the load further increase until the displacement of the motor reaches the maximum value, the inlet port pressure increase further.

If the load decreases under this condition, the motor continues reducing its displacement in the reverse sequence. As the load and inlet port pressure decreases and reaches the preset value, spool(64) moves left side by the pilot pressure(Pi). Therefore inlet port pressure flow into chamber B through port Sa and the swash plate moves until it touches stopper(10).



2907A2TM06

6) REDUCTION GEAR

(1) Planetary gear mechanism

Reduction unit slows down the rotating speed of motor and converts motor torque to strong rotating force.

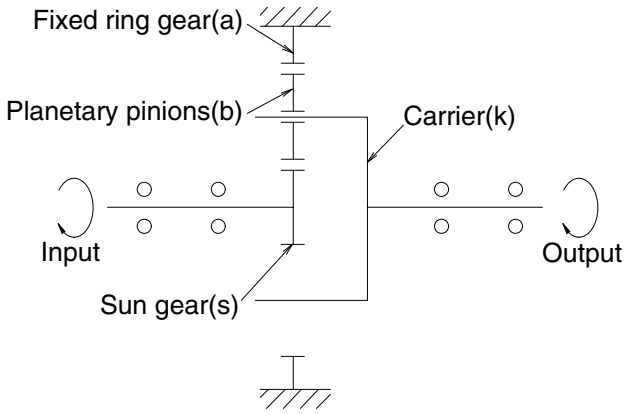
This reduction unit utilizes two stages, planetary reduction system.

Planetary reduction system consists of sun gear, planetary gears, carriers and ring gear.

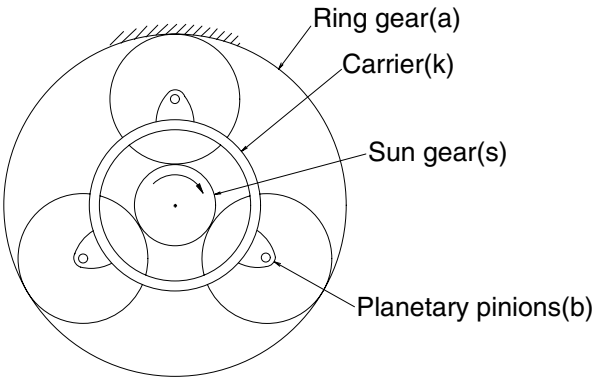
When the sun gear(s) is driven through input shaft, planetary pinions(b), rotating on their center, also move, meshing with fixed ring gear(a), around sun gears(s).

This movement is transferred to carrier(k) and deliver the torque.

This mechanism is called planetary gear mechanism.



29072TM10

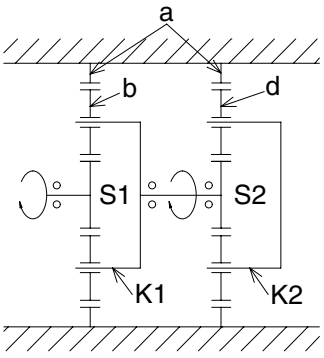


29072TM11

(2) Two stages reduction gear

When the sun gear **S1** is driven by input shaft, planetary action occurs among gears **S1**, **a** and **b** and revolution of gear **b** transfers the rotation of carrier **K1** to second sun gear **S2**, and also evokes planetary action between gear **S2**, **a** and **d**.

This time, because carrier **K2** is fixed to frame, gear **d** drives ring gear **a** and then ring gear a rotates to drive sprocket.

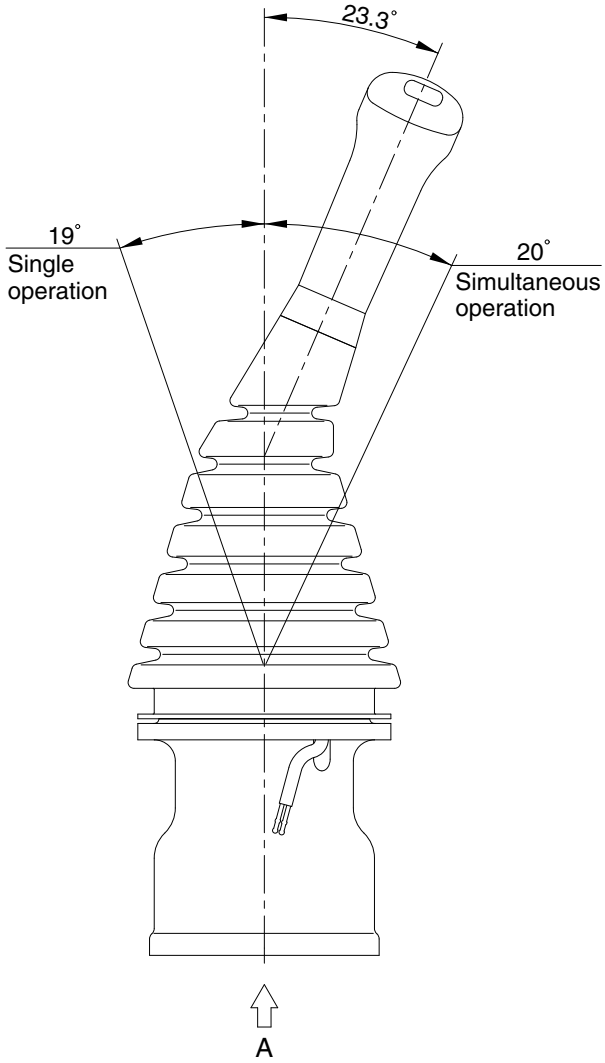


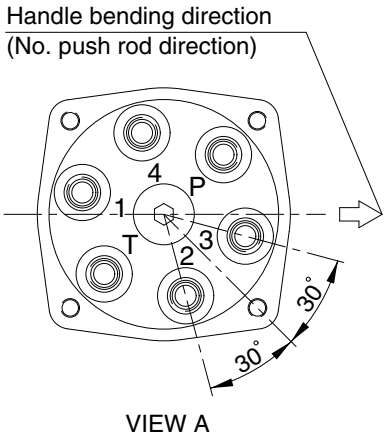
29072TM12

GROUP 5 RCV LEVER

1. STRUCTURE

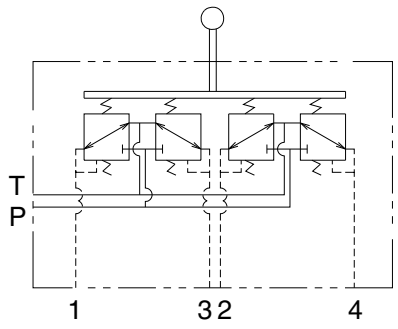
The casing has the oil inlet port P(Primary pressure) and the oil outlet port T(Tank). In addition the secondary pressure is taken out through ports 1, 2, 3 and 4 provided at the bottom face.





VIEW A

25032RL01



Hydraulic circuit

Port	LH	RH	Port size
P	Pilot oil inlet port	Pilot oil inlet port	PF 1/4
T	Pilot oil return port	Pilot oil return port	
1	Left swing port	Bucket out port	
2	Arm in port	Boom down port	
3	Right swing port	Bucket in port	
4	Arm out port	Boom up port	

CROSS SECTION

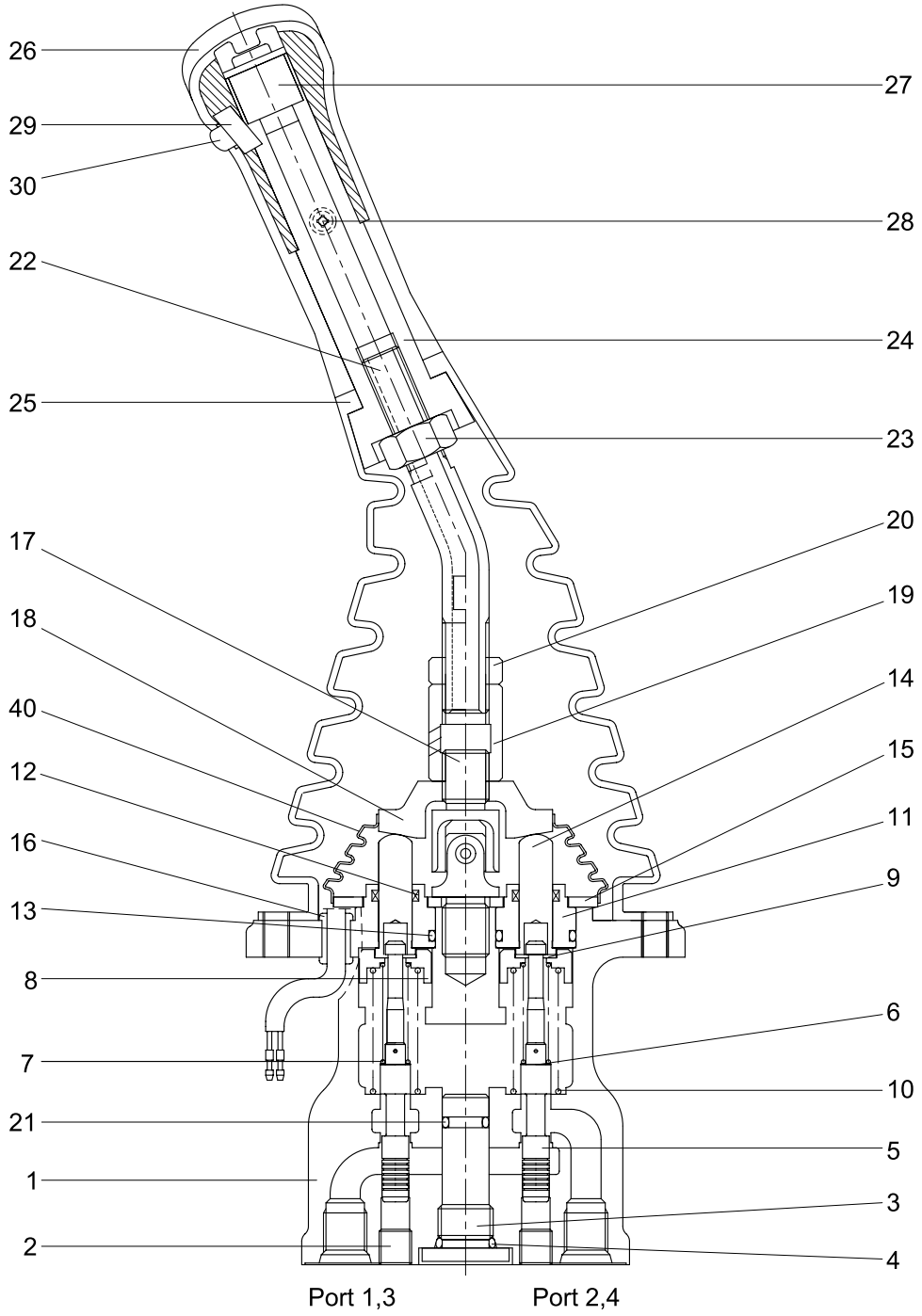
The construction of the pilot valve is shown in the attached cross section drawing. The casing has vertical holes in which reducing valves are assembled.

The pressure reducing section is composed of the spool(5), spring(7) for setting secondary pressure, return spring(10), stopper(9), spring seat(8) and shim(6). The spring for setting the secondary pressure has been generally so preset that the secondary pressure is 5 to 20.5kgf/cm^2(Depending on the type). The spool is pushed against the push rod(14) by the return spring.

When the push rod is pushed down by tilting the handle, the spring seat comes down simultaneously and changes setting of the secondary pressure spring.

1	Case	11	Plug	21	O-ring
2	Plug	12	Rod seal	22	Handle connector
3	Plug	13	O-ring	23	Nut
4	O-ring	14	Push rod	24	Insert
5	Spool	15	Plate	25	Boot
6	Shim	16	Bushing	26	Handle
7	Spring	17	Joint assembly	27	Switch assembly
8	Spring seat	18	Swash plate	28	Screw
9	Stopper	19	Adjusting nut	29	Switch assembly
10	Spring	20	Lock nut	30	Switch cover
				40	Boot

CROSS SECTION



14072SF80

2. FUNCTIONS

1) FUNDAMENTAL FUNCTIONS

The pilot valve is a valve that controls the spool stroke, direction, etc of a main control valve. This function is carried out by providing the spring at one end of the main control valve spool and applying the output pressure(Secondary pressure) of the pilot valve to the other end.

For this function to be carried out satisfactorily, the pilot valve is composed of the following elements.

(1) Inlet port(P) where oil is supplied from hydraulic pump.

(2) Output ports(1, 2, 3 & 4) to apply pressure supplied from inlet port to ends of control valve spools.

(3) Tank port(T) necessary to control the above output pressure.

(4) Spool to connect output port to inlet port or tank port.

(5) Mechanical means to control output pressure, including springs that work on the above spools.

2) FUNCTIONS OF MAJOR SECTIONS

The functions of the spool(5) are to receive the supply oil pressure from the hydraulic pump at its port P, and to change over oil paths to determine whether the pressure oil of port P is led to output ports 1, 2, 3 & 4 or the output port pressure oil to tank port T.

The spring(7) works on this spool to determine the output pressure.

The change the deflection of this spring, the push rod(14) is inserted and can slide in the plug(11).

For the purpose of changing the displacement of the push rod through the switch plate(19) and adjusting nut(20) are provided the handle(27) that can be tilted in any direction around the fulcrum of the universal joint(18) center.

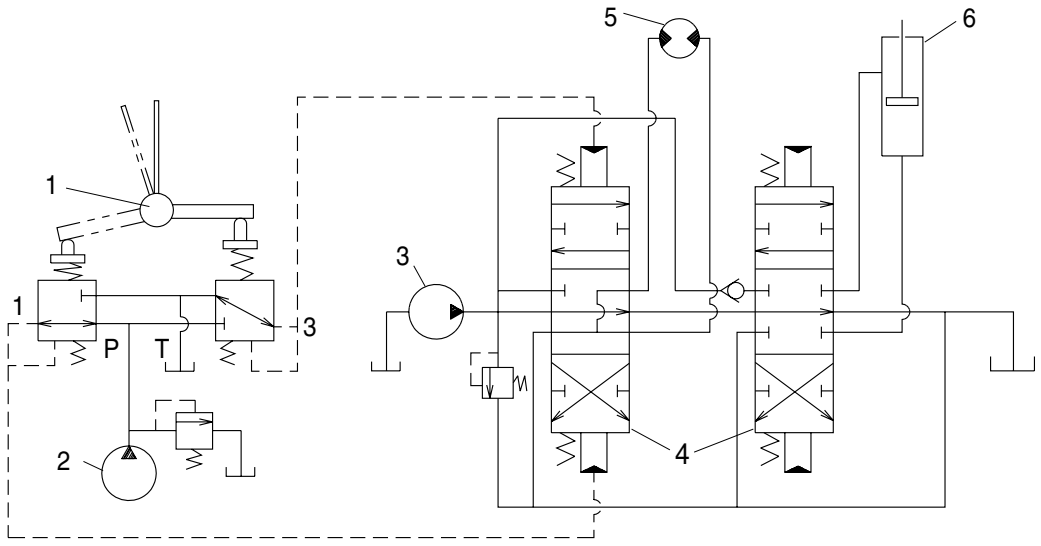
The spring(10) works on the case(1) and spring seat(8) and tries to return the push rod(14) to the zero-displacement position irrespective of the output pressure, securing its resetting to the center position.

This also has the effect of a reaction spring to give appropriate control feeling to the operator.

3) OPERATION

The operation of the pilot valve will be described on the basis of the hydraulic circuit diagram shown below and the attached operation explanation drawing.

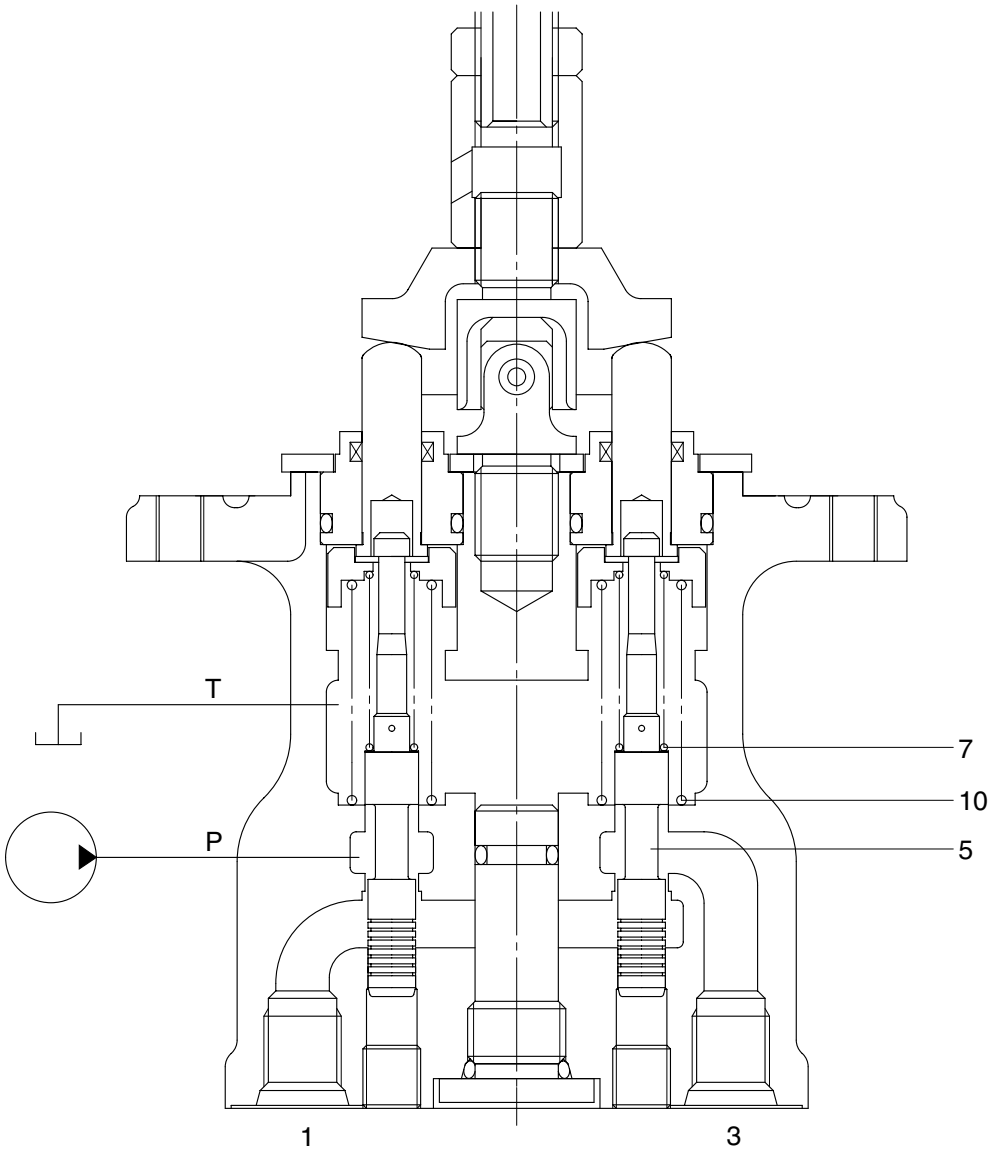
The diagram shown below is the typical application example of the pilot valve.



36072RL01

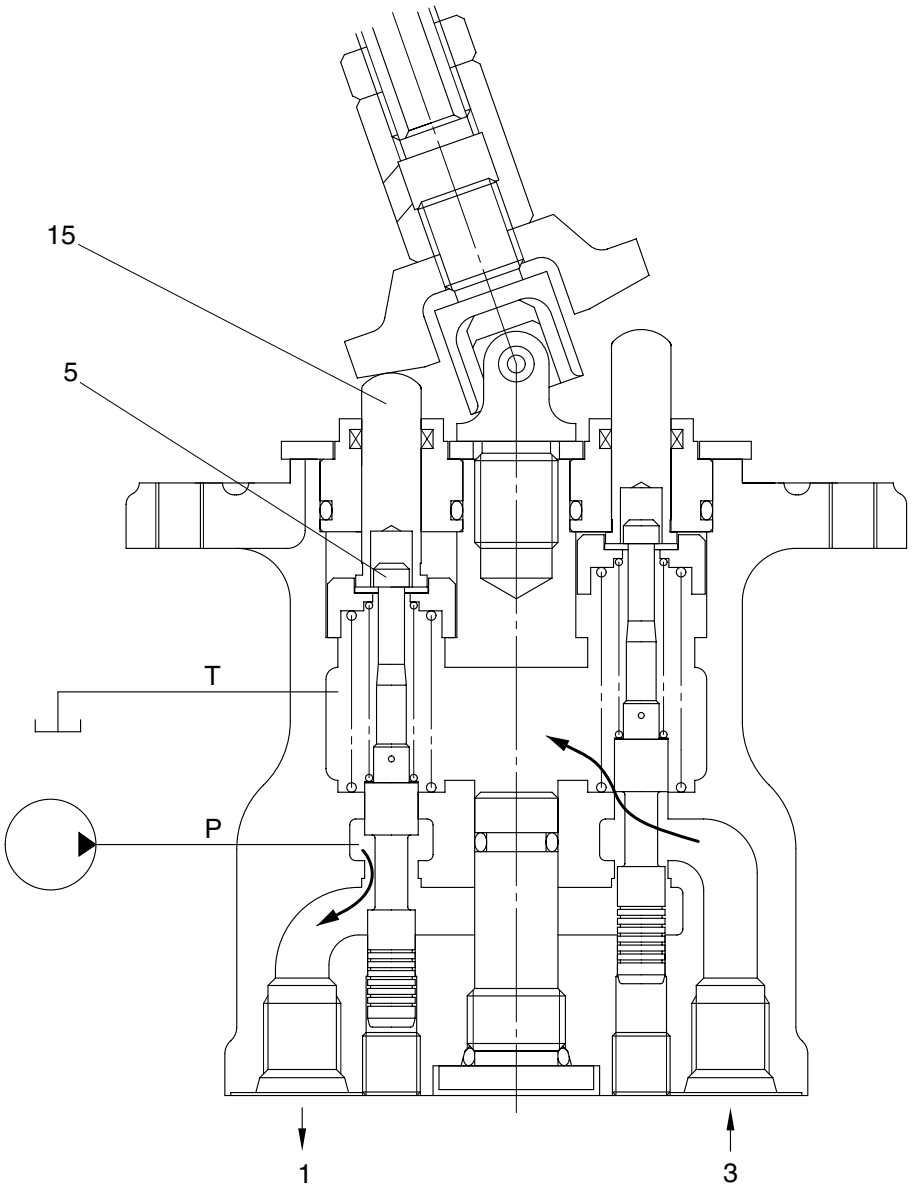
1	Pilot valve	3	Main pump	5	Hydraulic motor
2	Pilot pump	4	Main control valve	6	Hydraulic cylinder

(1) Case where handle is in neutral position



The force of the spring(7) that determines the output pressure of the pilot valve is not applied to the spool(5). Therefore, the spool is pushed up by the spring(10) to the position of port(1, 3) in the operation explanation drawing. Then, since the output port is connected to tank port T only, the output port pressure becomes equal to tank pressure.

(2) Case where handle is tilted

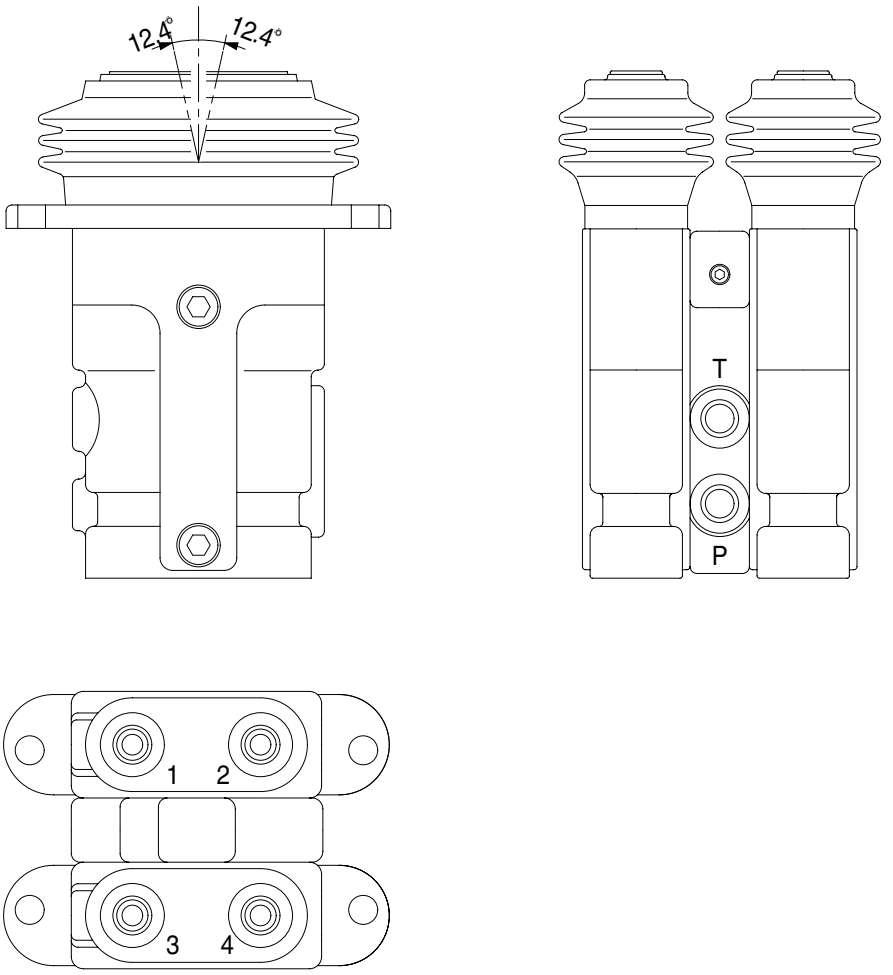


When the push rod(14) is stroked, the spool(5) moves downwards.
Then port P is connected with port(1) and the oil supplied from the pilot pump flows through port(1) to generate the pressure.
When the pressure at port(1) increases to the value corresponding to the spring force set by tilting the handle, the hydraulic pressure force balances with the spring force. If the pressure at port(1) increases higher than the set pressure, port P is disconnected from port(1) and port T is connected with port(1). If it decreases lower than the set pressure, port P is connected with port(1) and port T is disconnected from port 1.
In this manner the secondary pressure is kept at the constant value.
Besides, in some type, when the handle is tilted more than a certain angle, the upper end of the spool contacts with the inside bottom of the push rod and the output pressure is left to be connected with port P.

GROUP 6 RCV PEDAL

1. STRUCTURE

The casing(Spacer) has the oil inlet port P(Primary pressure), and the oil outlet port T(Tank). In addition the secondary pressure is taken out through ports 1, 2, 3 and 4 provided at the bottom face.



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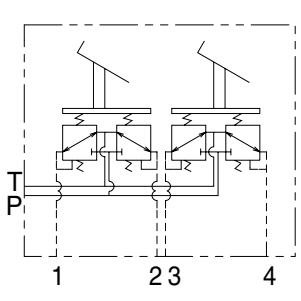


Hydraulic circuit

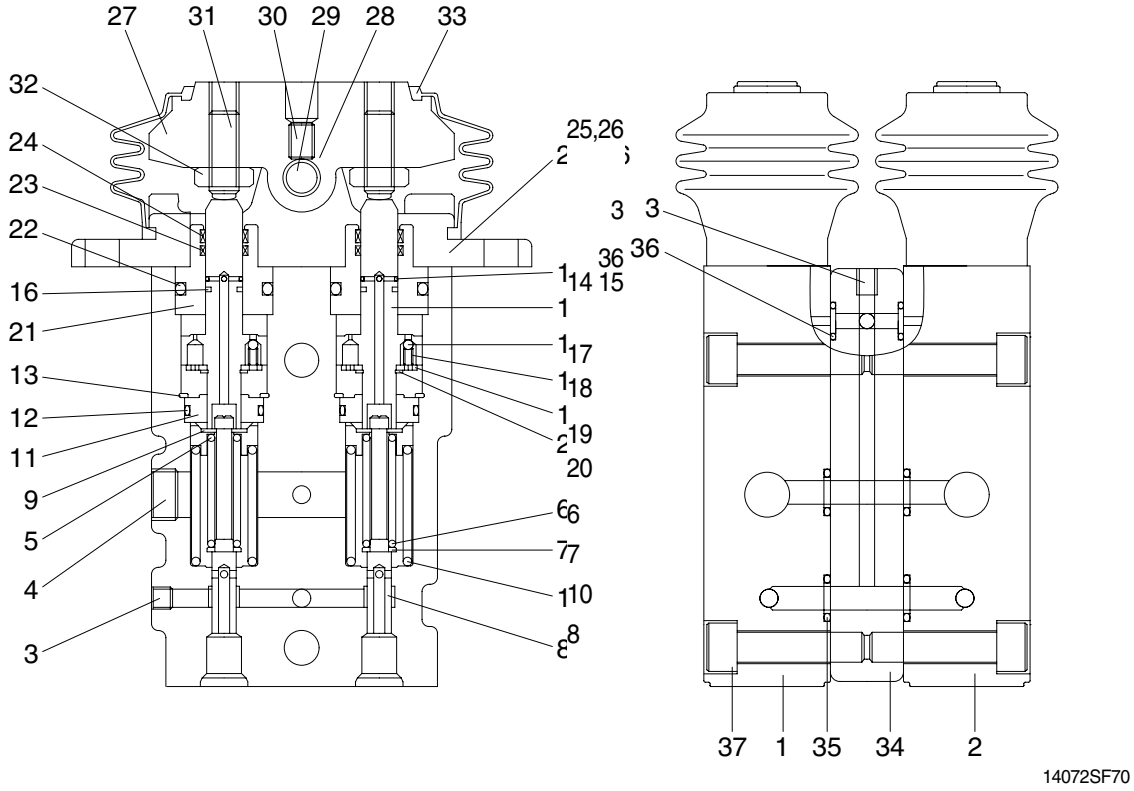
Port	Port	Port size
P	Pilot oil inlet port	PF 1/4
T	Pilot oil return port	
1	Travel(LH, Forward)	
2	Travel(LH, Backward)	
3	Travel(RH, Forward)	
4	Travel(RH, Backward)	

CROSS SECTION

The construction of the RCV pedal is shown in the below drawing. The casing has vertical holes in which reducing valves are assembled.

The pressure reducing section is composed of the spool(8), spring(6) for setting secondary pressure, return spring(10), stopper(9), and spring seat(7). The spring for setting the secondary pressure has been generally so preset that the secondary pressure is 5 to 19 kgf/cm^2 (depending on the type). The spool is pushed against the push rod(14) by the return spring.

When the push rod is pushed down by tilting pedal, the spring seat comes down simultaneously and changes setting of the secondary pressure spring.



1	Body(1)	13	Snap ring	25	Cover
2	Body(2)	14	Push rod	26	Bolt
3	Plug	15	Spring pin	27	Cam
4	Plug	16	Seal	28	Bushing
5	Spring seat	17	Steel ball	29	Cam shaft
6	Spring	18	Spring	30	Set screw
7	Spring seat	19	Plate	31	Set screw
8	Spool	20	Snap ring	32	Nut
9	Stopper	21	Plug	33	Bellows
10	Spring	22	O-ring	34	Space
11	Rod guide	23	Rod seal	35	O-ring
12	O-ring	24	Dust seal	36	O-ring
				37	Bolt

2. FUNCTION

1) FUNDAMENTAL FUNCTIONS

The pilot valve is a valve controls the spool stroke, direction, etc of a main control valve. This function is carried out by providing the spring at one end of the main control valve spool and applying the output pressure(Secondary pressure) of the pilot valve to the other end.

For this function to be carried out satisfactorily, the pilot valve is composed of the following elements.

Inlet port(P) where oil is supplied from hydraulic pump.

(1) Output port(1, 2, 3 & 4) to apply pressure supplied from inlet port to ends of control valve spools.

(2) Tank port(T) necessary to control the above output pressure.

(3) Spool to connect output port to inlet port tank port.

(4) Mechanical means to control output pressure, including springs that work on the above spools.

(5) FUNCTIONS OF MAJOR SECTIONS

2) The functions of the spool(8) are to receive the supply oil pressure from the hydraulic pump at its port P, and to change over oil paths to determine whether the pressure oil of port P is led to output ports 1, 2, 3 & 4 or the output spool to determine the output pressure.

The spring(6) works on this spool to determine the output pressure.

The change the deflection of this spring, the push rod(14) is inserted and can slide in the plug(21).

For the purpose of changing th displacement of the push rod through the cam(27) and adjusting nut(32) are provided the pedal that can be tilted in any direction around the fulcrum of the cam(27) center.

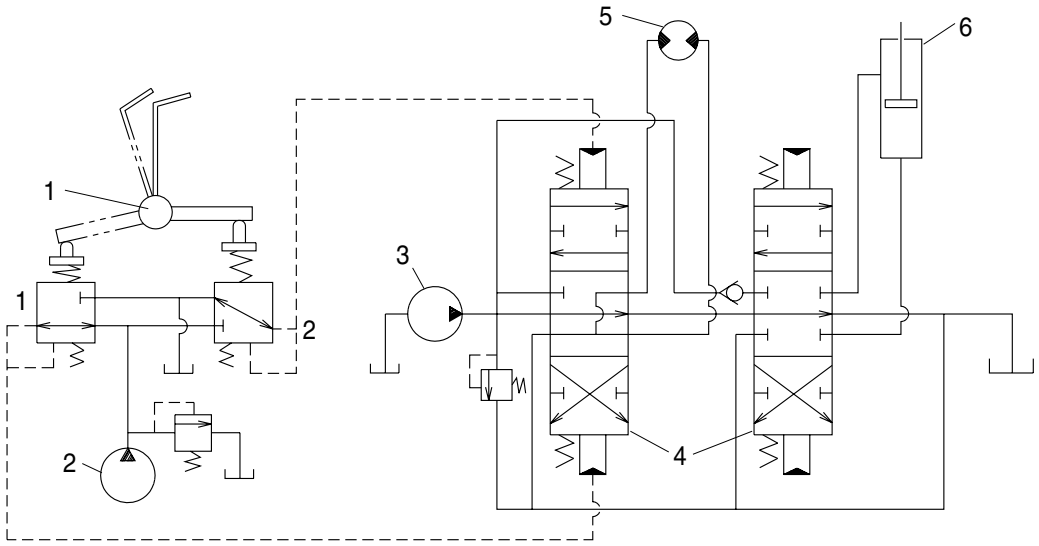
The spring(10) works on the casing(1) and spring seat(7) and tries to return the push rod(14) to the zero-displacement position irrespective of the output pressure, securing its resetting to the center position.

This also has the effect of a reaction spring to give appropriate control feeling to the operator.

3) OPERATION

The operation of the pilot valve will be described on the basis of the hydraulic circuit diagram shown below ant the attached operation explanation drawing.

The diagram shown below is the typical application example of the pilot valve.



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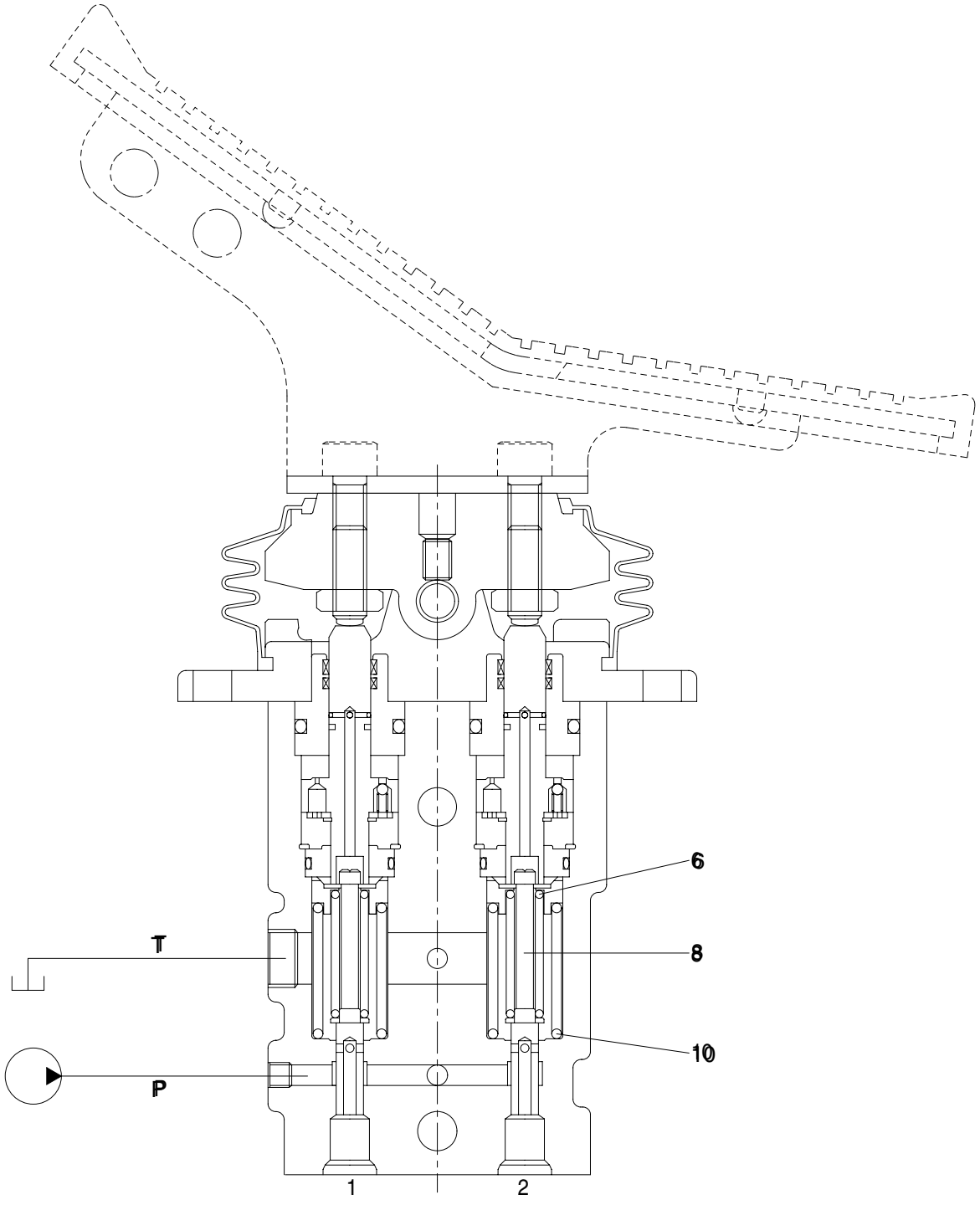
1	Pilot valve	3	Main pump	5	Hydraulic motor
2	Pilot pump	4	Main control valve	6	Hydraulic cylinder

(1) Case where pedal is in neutral position



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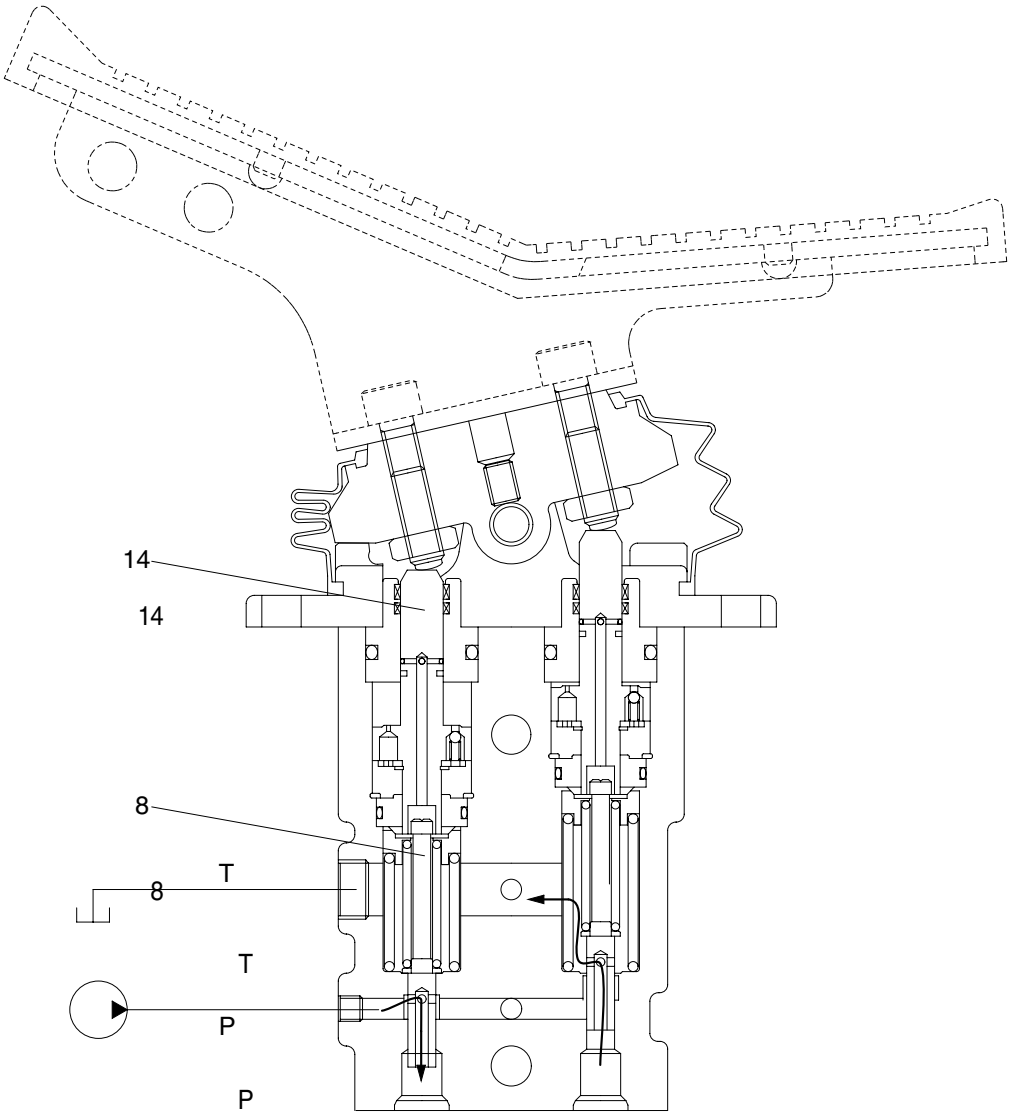
The force of the spring(6) that determines the output pressure of the pilot valve is not applied to the spool(8). Therefore, the spool is pushed up by the spring(10) to the position of port 2 in the operation explanation drawing. Then, since the output port is connected to tank port T only, the output port pressure becomes equal to tank pressure.

(2) Case where pedal is tilted



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When the push rod(14) is stroked, the spool(8) moves downwards.
Then port P is connected with port 1, and the oil supplied from the pilot pump flows through port 1 to generate the pressure.
When the pressure at port 1 increases to the value corresponding to the spring force set by tilting the handle, the hydraulic pressure force balances with the spring force. If the pressure at port 1 increases higher than the set pressure, port P is disconnected from port 1 and port T is connected with port 1. If it decreases lower than the set pressure, port P is connected with port 1 and port T is disconnected from port 1.
In this manner the secondary pressure is kept at the constant value.
Besides, in some type, when the handle is tilted more than a certain angle, the upper end of the spool contacts with inside bottom of the push rod and the output pressure is left to be connected with port P.